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- 4.1 a. narrow, needle-like leaves; evergreens; conifers
	- b. broadleafed; deciduous
	- c. softwoods

### 4.2 See Fig. 4.4.

a. Annual Ring – wood cells developed on the outside of the tree in one growing season

b. Latewood (summerwood) – smaller, darker, more dense, from late in growing season

Earlywood (springwood) – larger, light color, less dense, from early in growing season

c. Heartwood – at center of log; inactive cells; collects deposits as aged; darker Sapwood – outer, living, active cells; stores food and transports water

#### 4.3 a. Amount of water expressed as percent of dry weight of wood material: *x*100% *MC* =  $\frac{moistweight - overdryweight}{1}$

- *ovendryweight*
- b.  $FSP = MC$  at point of "no free water" and fully saturated (bound water) wood.
- c.  $EMC = MC$  wood assumes in service for a given set of atmospheric conditions

## 4.4 a.  $MC \le 19\%$ b.  $MC > 19\%$ Note: These values are for sawn lumber; for glulam see Chap. 5 (dry MC < 16%).

4.5 EMC for buildings in "dry southwest states" is approximately 9%, ranging from 7% to 12% (Ref. 4.7)

4.6 *National Design Specification for Wood Construction* (NDS) *Design Values for Wood Construction* (NDS Design Values Supplement) *Special Design Provisions for Wind & Seismic* (NDS Wind & Seismic Supplement) *Commentary on the National Design Specification for Wood Construction* (NDS Commentary) ASD/LRFD *Manual for Engineered Wood Construction*

- 4.7 a. NDS *Design Values Supplement* (provides design values for sawn lumber and glued laminated timber); *Wind & Seismic Supplement* (provides design guidelines and unit shear capacities for wood shearwalls and diaphragms resisting lateral wind and seismic loads).
	- b. The *Wind & Seismic Supplement* is separated from the NDS and the NDS *Design Values Supplement* due to the unique requirements for wind-resistant and seismic-resistant design.



4.9 a. Nominal 2 to 4 in. thick; any width; practically 2 x 2 to 4 x 16.

- b. Minimum nominal dimension of 5 in. or larger; includes beams  $\&$  stringers (width more than 2 in. > thickness) and posts  $&$  timbers (square; not more than 2 in. out of square).
- c. Design values for the main size categories are determined following different procedures.

Dimension Lumber- In-grade test program: ASTM D1990. Timbers – small clears: ASTM D2555 & D245 Dimension Lumber: NDS Supplement Tables 4A, 4B, 4C, 4F Timbers: NDS Supplement Table 4D

- 4.10 a. Timbers; 5 in. and thicker, width more than 2 in. > thickness
	- b. Dimension Lumber; 2 to 4 in. thick, 2 to 4 in. wide
	- c. Dimension Lumber; 2 to 4 in. thick, 4 in. and wider
	- d. Dimension Lumber, 2 to 4 in. thick, 5 in. and wider
	- e. Timbers; 5 in. and thicker, width not more than 2 in. > thickness
	- f. Dimension Lumber; 2 to 4 in. thick, 2 to 4 in. wide
	- g. Dimension Lumber; 2 to 4 in. thick, 2 in. and wider,  $\leq 10$  ft long
- 4.11 Visually graded is graded visually by inspecting and marking each piece. NDS Supplement Tables 4A, 4B, 4D, 4E, and 4F MSR refers to lumber graded mechanically by subjecting each piece to a nondestructive test that measure E about the weak axis. A visual check is also included. NDS Supplement Table 4C



4.12 Visually graded sawn lumber other than Southern Pine

4.13 Visually graded sawn lumber – Southern Pine



4.14 Stress grades of visually graded Hem-Fir

- a. Dimension Lumber (NDS Supplement Table 4A) Select Structural; No.1 & Btr; No.1; No.2; No.3; Stud; Construction; Standard; Utility
- b. Beams and Stringers (NDS Supplement Table 4D) Select Structural; No.1; No.2
- c. Posts and Timbers (NDS Supplement Table 4D) Select Structural; No.1; No.2

#### 4.15 Stress grades of visually graded Southern Pine

a. Dimension Lumber-stress grades vary with size (NDS Supplement Table 4B)





b. Beams & Stringers and c. Posts & Timbers (NDS Supplement Table 4D)



4.16 Reference design values for No. 1 DF-L

Dimension Lumber (NDS Supplement Table 4A)

- c. 4 x 16 Dimension Lumber
- d. 4 x 4 Dimension Lumber
- e. 2 x 10 Dimension Lumber

 $F_b$  $= 1000$  psi  $F_t$  $= 675 \text{ psi}$  $F_{v}$  $= 180 \text{ psi}$  $F_c \perp = 625 \text{ psi}$  $F_c$  $= 1500$  psi  $E = 1,700,000 \text{ psi}$  $E_{\rm min}$  $= 620,000 \text{ psi}$ 

Beams and Stringers (NDS Supplement Table 4D)

f.  $6 \times 12$  – Beams & Stringers

h. 10 x 14 – Beams & Stringers  $F_b$  $= 1350$  psi

 $F_{t}$  $= 675 \text{ psi}$  $F<sub>v</sub>$  $= 170 \text{ psi}$  $F_{c}$  $\perp$  $= 625$  psi  $F_c$  $= 925$  psi  $E = 1,600,000 \text{ psi}$  $E_{\rm min}$  $= 580,000 \text{ psi}$ 

Posts and Timbers (NDS Supplement Table 4D)

a.  $10 \times 10$  – Posts & Timbers

b.  $12 \times 14$  – Posts & Timbers

g.  $6 \times 8$  – Posts & Timbers

 $F<sub>b</sub>$  $= 1200$  psi  $F_t$  $= 825$  psi  $F<sub>v</sub>$  $=$  170 psi  $F_c \perp$  $= 625 \text{ psi}$  $F_c$  $F_c$  = 1000 psi<br>*E* = 1.600.00  $= 1,600,000 \text{ psi}$  $E_{\rm min}$  $= 580,000 \text{ psi}$ 



# 4.18



- a. Load Duration Factor,  $C_D$ : The strength of a wood member is affected by the total accumulated length of time that a load is applied. The shorter the duration of load, the higher the strength of a wood member.  $C_D$  is the multiplier that adjusts the reference design value from normal duration (10 years) to other durations for ASD. See NDS 2.3.2 and Appendix B.
- b. Time Effect Factor,  $\lambda$ : The strength of a wood member is affected by the total accumulated length of time that a load is applied.  $\lambda$  is the multiplier that adjusts the LRFD resistance of wood members to ensure that consistent reliability is achieved for load duration effects in various LRFD load combinations. See NDS Appendix N.
- c. Wet Service Factor,  $C_M$ : The moisture content (MC) of a wood member affects its load capacity. Most reference design values for sawn lumber apply to  $MC \leq 19$  percent in service. Higher moisture contents require reduction of design values by  $C_M$ . The following reference design values are subject to adjustment for increased moisture content:  $F_b$ ,  $F_t$ ,  $F_v$ ,  $F_c \perp$ ,  $F_c$ ,  $E$  and  $E_{min}$ . See NDS 4.3.3 and the Adjustment Factors sections of NDS Supplement Tables 4A, 4B, 4C, 4D, 4E, and 4F.
- d. Size Factor,  $C_F$ : The size of a wood member affects its strength. For visually graded Dimension Lumber,  $C_F$  applies to  $F_b$ ,  $F_t$  and  $F_c$ . See NDS 4.3.6.1 and the Adjustment Factor section of NDS Supplement Tables 4A and 4F. For Timbers, the size factor applies to  $F_b$ ,  $E$  and  $E_{\text{min}}$ . See NDS 4.3.6.2 and the Adjustment Factors section of NDS Supplement Table 4D. For Decking,  $C_F$ applies only to  $F_b$ . See NDS 4.3.6.4 and the Adjustment Factors section of NDS Supplement Table 4E.
- e. Repetitive Member Factor,  $C_r$ : Applies to Dimension Lumber, but not to Timbers. When three or more wood members are spaced not more that 24 in. o/c and are connected together by a load distributing element (such as sheathing), the bending design value  $F_b$  may be increased by  $C_r = 1.15$ . This is a 15 percent increase over single member bending design values. The  $C_r$ adjustment recognizes that failure of a single member in a repetitive application will not mean failure of the overall system. The load will be distributed to other members.
- 4.20 Wood has the ability to support higher stresses for short periods of time. Both the load duration factor  $(C_D)$  and the time effect factor  $(\lambda)$  are employed to adjust wood strength properties based on the duration of applied design loads. The load duration factor  $(C_D)$  is used to adjust reference design values in allowable stress design (ASD), and is based on the shortest duration load in an ASD load combination. The time effect factor  $(\lambda)$  is used to adjust nominal design values in load and resistance factor design (LRFD), and is based on the dominant transient load in an LRFD load combination. Time effect factors  $(\lambda)$  are intended to ensure consistent reliability for load duration effects across various load combinations.

4.19





- 4.23 Compression perpendicular to grain  $(F<sub>c</sub>⊥)$ , average modulus of elasticity  $(E)$ , and reduced modulus of elasticity for stability calculations  $(E_{min})$  are not adjusted by  $C_D$  in ASD calculations. Average modulus of elasticity (*E*) and reduced modulus of elasticity for stability calculations ( $E_{\text{min}}$ ) are not adjusted by  $\lambda$  in LRFD calculations.
- 4.24 Most reference design values apply to dry service conditions<sup>\*</sup>. Dry service conditions are defined as:

a. Sawn Lumber  $MC \le 19$  percent b. Glulam  $MC < 16$  percent When the moisture content exceeds these limits, the reference design values are reduced by a wet service factor  $C_M$  that is less than 1.0 \* Southern Pine has  $C_M$  included in some of the reference design values (NDS Supplement Table 4B). *Many of the reference design values for Timbers* (NDS Supplement Table 4D) have already been adjusted for C<sub>M</sub>.

4.25 The load capacity of wood decreases as the temperature increases. Reductions in strength caused by heating up to 150 degrees F are generally reversible when the temperature returns to normal. Reductions in strength may not be reversible when heating exceeds 150 degrees F. Reduction in strength occurs when the member is subjected to the full design capacity. When a wood member is consistently heated above 100 degrees F and is subjected to the full design load, an adjustment for temperature effects will be required. This may occur in an industrial plant, but reductions are not normally required in ordinary roof structures. See NDS 2.3.3 and Appendix C.

4.26 Pressure preservative treated wood has chemicals impregnated in the treated zone which resist attack by decay, termites and other insects, and marine borers. Reference design values apply directly to preservative treated wood, and an adjustment factor is not required unless the member has been incised to increase the penetration of preservatives.

Fire-retardant-treated wood has much higher concentrations of chemicals than preservative treated wood. At one time a 10 percent reduction in reference design values was specified. However, the reduction in strength varies with the treating process, and the NDS refers the designer to the company providing the fire retardant treatment for appropriate reduction factors.

NOTE: The performance of fire-retardant-treated wood products has been the subject of considerable concern and litigation.

4.27 Wood that is *continuously submerged* in fresh water will not decay and does not need to be preservative treated. However, wood that is partially submerged, or wood that undergoes cycles of submersion followed by exposure to the atmosphere, should be preservative treated. Wood that is submerged in salt water should be preservative treated to protect from marine borers. Preservative treated wood has an extensive record of resisting attack in both fresh water and salt water environments.



4.28 Critical ASD load combination for a fully braced member

\* The largest load in this column defines the critical ASD load combination for a fully braced member:  $D + L$ 

Note: Actual loads (i.e., loads that have not been divided by  $C_D$ ) should be used in design calculations. The load duration factor should then be used to adjust the appropriate reference design values. In this problem, the actual critical load is 19 k and  $C_D = 1.0$ . The above analysis is used to define the critical combination only for fully braced members, tension members, or connections.





\* The largest load in this column defines the critical ASD load combinationfor a fully braced member:  $\mathbf{D} + 0.75(\mathbf{L} + \mathbf{S})$ 

Note: Actual loads (i.e., loads that have not been divided by  $C_D$ ) should be used in design calculations. The load duration factor should then be used to adjust the appropriate reference design values. In this problem, the actual critical load is  $35.75$  k and  $C_D = 1.15$ . The above analysis is used to define the critical combination only for fully braced members, tension members, or connections.

4.30 ASD – Hem-Fir No.2, fully braced; bending about strong axis a. **2 x 10 joists** at 16 in. o.c.  $(C_r = 1.15)$ ; D + S load combination  $(C_D = 1.15)$  $C_F$  applies for bending, tension and compression; other factors equal to unity  $F_b' = F_b (C_D) (C_F) (C_r) = 850 \text{ psi} (1.15) (1.1) (1.15) = 1237 \text{ psi}$  $F_t' = F_t(C_D)(C_F) = 525$  psi  $(1.15)(1.1) = 664$  psi  $F_v' = F_v(C_D) = 150 \text{ psi} (1.15) = 173 \text{ psi}$  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 405 psi  $F_c' = F_c (C_D)(C_F) = 1300 \text{ psi} (1.15)(1.0) = 1495 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,300,000 \text{ psi}$  $E_{\text{min}}$ <sup>'</sup> =  $E_{\text{min}}$  = 470,000 psi **6 x 14** (Beams & Stringers); supports permanent load  $(C_D = 0.9)$  $C_F = (12/13.5)^{0.111} = 0.987$  for bending; other factors equal to unity  $F_b' = F_b (C_D) (C_F) = 675$  psi (0.9)(0.987) = 600 psi  $F_t' = F_t(C_D) = 350 \text{ psi } (0.9) = 315 \text{ psi}$  $F_v' = F_v(C_D) = 140 \text{ psi } (0.9) = 126 \text{ psi }$  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 405 psi  $F_c' = F_c (C_D) = 500 \text{ psi} (0.9) = 450 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,100,000 \text{ psi}$  $E_{\text{min}}$ <sup>'</sup> =  $E_{\text{min}}$  = 400,000 psi c. **4 x 14 purlins** at 8 ft o.c.; D + L<sub>r</sub> load combination ( $C_D = 1.25$ )  $C_F$  applies for bending, tension and compression; other factors equal to unity  $F_b' = F_b (C_D) (C_F) = 850$  psi (1.25)(1.0) = 1063 psi  $F_t' = F_t(C_D)(C_F) = 525$  psi (1.25)(0.9) = 591 psi  $F_v' = F_v(C_D) = 150 \text{ psi} (1.25) = 188 \text{ psi}$  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 405 psi  $F_c' = F_c(C_D)(C_F) = 1300 \text{ psi } (1.25)(0.9) = 1463 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,300,000 \text{ psi}$  $E_{\text{min}}$ <sup>'</sup> =  $E_{\text{min}}$  = 470,000 psi d. **4 x 6 beams** at 4 ft o.c.; D + L load combination  $(C_D = 1.0)$ ;  $C_M$  applies  $C_F$  applies for bending, tension and compression; other factors equal to unity  $F_b' = F_b (C_D) (C_M) (C_F) = 850$  psi (1.0)(1.0)(1.3) = 1105 psi  $C_M = 1.0$  for bending since  $F_b$  ( $C_F$ ) = 1105 psi < 1150 psi  $F_t' = F_t(C_D)(C_M)(C_F) = 525$  psi (1.0)(1.0)(1.3) = 683 psi  $F_v' = F_v(C_D)(C_M) = 150 \text{ psi } (1.0)(0.97) = 146 \text{ psi }$  $F_c\perp$ <sup>2</sup> =  $F_c\perp$  (*C<sub>M</sub>*) = 405 psi (0.67) = 271 psi  $F_c' = F_c$  (*C<sub>D</sub>*)(*C<sub>M</sub>*)(*C<sub>F</sub>*) = 1300 psi (1.0)(0.8)(1.1) = 1144 psi Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E(C_M) = 1,300,000 \text{ psi } (0.9) = 1,170,000 \text{ psi }$  $E_{\text{min}}$ <sup>\*</sup> =  $E_{\text{min}}$  (*C<sub>M</sub>*) = 470,000 psi (0.9) = 423,000 psi

4.31 LRFD  $(K_F$  and  $\Phi$  apply) – Hem-Fir No.2, fully braced; bending about strong axis

a. **2 x 10 joists** at 16 in. o.c.  $(C_r = 1.15)$ ; 1.2D + 1.6S load combination ( $\lambda = 0.8$ )  $C_F$  applies for bending, tension and compression; other factors equal to unity  $A = 13.88 \text{ in}^2$ ;  $S_x = 21.39 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 850 psi (2.54) = 2159 psi = 2.16 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_F$ )( $C_r$ ) = 2.16 ksi (0.85)(0.8)(1.1)(1.15) = 1.86 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 1.86$  ksi  $(21.39 \text{ in}^3) = 39.7$  k-in. Tension  $F_{tn} = F_t (K_F) = 525$  psi (2.7) = 1418 psi = 1.42 ksi  $F_m' = F_m (\Phi_t)(\lambda)(C_F) = 1.42$  ksi  $(0.8)(0.8)(1.1) = 0.998$  ksi  $T_n' = F_{tn}$ <sup>'</sup> (*A*) = 0.998 ksi (13.88 in<sup>2</sup>) = 13.9 k Flexural Shear  $F_{vn} = F_v (K_F) = 150 \text{ psi} (2.88) = 432 \text{ psi} = 0.432 \text{ ksi}$  $F_{vn}$ <sup>\*</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.432 ksi (0.75)(0.8) = 0.259 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.259 ksi (2/3)(13.88 in<sup>2</sup>) = 2.4 k Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 405$  psi (2.083) = 844 psi = 0.844 ksi  $F_c \perp n' = F_c \perp n (\Phi_c)(\lambda) = 0.844$  ksi (0.9)(0.8) = 0.607 ksi Compression parallel to grain  $F_{cn} = F_c$  (*K<sub>F</sub>*) = 1300 psi (2.4) = 3120 psi = 3.12 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ )( $C_F$ ) = 3.12 ksi (0.9)(0.8)(1.0) = 2.25 ksi  $P_n$ <sup>2</sup> =  $F_{cn}$ <sup>2</sup> (*A*) = 2.25 ksi (13.88 in<sup>2</sup>) = 31.2 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,300,000 \text{ psi} = 1300 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 470,000 psi (1.765) = 829,550 psi = 830 ksi  $E_{\min-n}$ <sup>'</sup> =  $E_{\min-n}$  ( $\Phi_s$ ) = 830 ksi (0.85) = 705 ksi b. **6 x 16** (Beams & Stringers);  $1.2D + 1.6L$  (storage) load ( $\lambda = 0.7$ )  $C_F = (12/15.5)^{0.111} = 0.972$  for bending; other factors equal to unity  $A = 85.25 \text{ in}^2$ ;  $S_x = 220.2 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  (*K<sub>F</sub>*) = 675 psi (2.54) = 1714.5 psi = 1.71 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_F$ ) = 1.71 ksi (0.85)(0.7)(0.972) = 0.992 ksi  $M_n' = F_{bn}$ <sup>'</sup> (*S<sub>x</sub>*) = 0.992 ksi (220.2 in<sup>3</sup>) = 218 k-in. Tension  $F_{tn} = F_t (K_F) = 350 \text{ psi} (2.7) = 945 \text{ psi} = 0.945 \text{ ksi}$  $F_{tn}$ <sup>\*</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ ) = 0.945 ksi (0.8)(0.7) = 0.529 ksi  $T_n' = F_{tn}$ <sup>'</sup> (*A*) = 0.529 ksi (85.25 in<sup>2</sup>) = 45.1 k **Flexural Shear**  $F_{vn} = F_v (K_F) = 140 \text{ psi} (2.88) = 403 \text{ psi} = 0.403 \text{ ksi}$  $F_{vn}$ <sup>2</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.403 ksi (0.75)(0.7) = 0.212 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.212 ksi (2/3)(85.25 in<sup>2</sup>) = 12.0 k

Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 405$  psi (2.083) = 844 psi = 0.844 ksi  $F_c \perp_n^{\bullet} = F_c \perp_n (\Phi_c)(\lambda) = 0.844$  ksi (0.9)(0.7) = 0.531 ksi Compression parallel to grain  $F_{cn} = F_c$  ( $K_F$ ) = 500 psi (2.4) = 1200 psi = 1.20 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ ) = 1.20 ksi (0.9)(0.7) = 0.756 ksi  $P_n$ <sup>2</sup> =  $F_{cn}$ <sup>2</sup> (*A*) = 0.756 ksi (85.25 in<sup>2</sup>) = 64.4 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,100,000 \text{ psi} = 1100 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 400,000 psi (1.765) = 706,000 psi = 706 ksi  $E_{\min-n}$ <sup>'</sup> =  $E_{\min-n}$  ( $\Phi_s$ ) = 706 ksi (0.85) = 600 ksi c. **4 x 14 purlins** at 8 ft o.c.;  $1.2D + 1.6L_r$  load combination ( $\lambda = 0.8$ )  $C_F$  applies for bending, tension and compression; other factors equal to unity  $A = 46.38 \text{ in}^2$ ;  $S_x = 102.4 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 850 psi (2.54) = 2159 psi = 2.16 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_F$ ) = 2.16 ksi (0.85)(0.8)(1.0) = 1.47 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 1.47$  ksi  $(102.4 \text{ in}^3) = 150$  k-in. Tension  $F_{tn} = F_t (K_F) = 525$  psi (2.7) = 1418 psi = 1.42 ksi  $F_{tn}$ <sup>t</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ )( $C_F$ ) = 1.42 ksi (0.8)(0.8)(0.9) = 0.816 ksi  $T_n' = F_{tn}$ <sup>'</sup> (*A*) = 0.816 ksi (46.38 in<sup>2</sup>) = 37.9 k Flexural Shear  $F_{vn} = F_v (K_F) = 150 \text{ psi} (2.88) = 432 \text{ psi} = 0.432 \text{ ksi}$  $F_{vn}$ <sup>\*</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.432 ksi (0.75)(0.8) = 0.259 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.259 ksi (2/3)(46.38 in<sup>2</sup>) = 8.0 k Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 405$  psi (2.083) = 844 psi = 0.844 ksi  $F_c \perp n' = F_c \perp n (\Phi_c)(\lambda) = 0.844$  ksi (0.9)(0.8) = 0.607 ksi Compression parallel to grain  $F_{cn} = F_c$  (*K<sub>F</sub>*) = 1300 psi (2.4) = 3120 psi = 3.12 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ )( $C_F$ ) = 3.12 ksi (0.9)(0.8)(0.9) = 2.02 ksi  $P_n' = F_{cn}$ <sup>2</sup> (*A*) = 2.02 ksi (46.38 in<sup>2</sup>) = 93.8 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,300,000 \text{ psi} = 1300 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 470,000 psi (1.765) = 829,550 psi = 830 ksi  $E_{\min-n}$ <sup>'</sup> =  $E_{\min-n}$  ( $\Phi_s$ ) = 830 ksi (0.85) = 705 ksi

d. **4 x 6 beams** at 4 ft o.c.;  $1.2D + 1.6L$  (occupancy) load ( $\lambda = 0.8$ );  $C_M$  applies  $C_F$  applies for bending, tension and compression; other factors equal to unity  $A = 19.25 \text{ in}^2$ ;  $S_x = 17.65 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 850 psi (2.54) = 2159 psi = 2.16 ksi  $C_M = 1.0$  for bending since  $F_b$  ( $C_F$ ) = 1105 psi < 1150 psi  $F_{bn}$ <sup>2</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_M$ )( $C_F$ ) = 2.16 ksi (0.85)(0.8)(1.0)(1.3) = 1.91 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 1.91$  ksi  $(17.65 \text{ in}^3) = 33.7 \text{ k-in.}$ Tension  $F_{tn} = F_t (K_F) = 525$  psi (2.7) = 1418 psi = 1.42 ksi  $F_{tn}$ <sup>\*</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ )( $C_M$ )( $C_F$ ) = 1.42 ksi (0.8)(0.8)(1.0)(1.3) = 1.18 ksi  $T_n' = F_{tn}$ <sup>2</sup> (*A*) = 1.18 ksi (19.25 in<sup>2</sup>) = 22.7 k Flexural Shear  $F_{vn} = F_v (K_F) = 150 \text{ psi} (2.88) = 432 \text{ psi} = 0.432 \text{ ksi}$  $F_{vn}$ <sup>\*</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ )( $C_M$ ) = 0.432 ksi (0.75)(0.8)(0.97) = 0.251 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.251 ksi (2/3)(19.25 in<sup>2</sup>) = 3.2 k Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 405$  psi (2.083) = 844 psi = 0.844 ksi  $F_c \perp_n$ <sup>*r*</sup> =  $F_c \perp_n (\Phi_c)(\lambda)(C_M) = 0.844$  ksi (0.9)(0.8)(0.67) = 0.407 ksi Compression parallel to grain  $F_{cn} = F_c$  (*K<sub>F</sub>*) = 1300 psi (2.4) = 3120 psi = 3.12 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ )( $C_M$ )( $C_F$ ) = 3.12 ksi (0.9)(0.8)(0.8)(1.1) = 1.98 ksi  $P_n$ <sup>2</sup> =  $F_{cn}$ <sup>2</sup> (*A*) = 1.98 ksi (19.25 in<sup>2</sup>) = 38.1 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E(C_M) = 1,300,000 \text{ psi } (0.9) = 1,170,000 = 1170 \text{ ksi }$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 470,000 psi (1.765) = 829,550 psi = 830 ksi  $E_{\min-n}$ <sup>\*</sup> =  $E_{\min-n}$  ( $\Phi_s$ )( $C_M$ ) = 830 ksi (0.85)(0.9) = 635 ksi

4.32 ASD – Select Structural Southern Pine, fully braced; bending about strong axis a. **2 x 6 joists** at 24 in. o.c.  $(C_r = 1.15)$ ; D + S load combination  $(C_D = 1.15)$  $C_F$  is already included in NDS Table 4B values; other factors equal to unity  $F_b' = F_b (C_D) (C_r) = 2550 \text{ psi} (1.15) (1.15) = 3372 \text{ psi}$  $F_t' = F_t(C_D) = 1400 \text{ psi} (1.15) = 1610 \text{ psi}$  $F_v' = F_v(C_D) = 175$  psi (1.15) = 201 psi  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 565 psi  $F_c' = F_c (C_D) = 2000 \text{ psi} (1.15) = 2300 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,800,000 \text{ psi}$  $E_{\min}$ <sup>'</sup> =  $E_{\min}$  = 660,000 psi b. **4 x 12 beam**;  $D + L + L_r$  load combination ( $C_D = 1.25$ )  $C_F = 1.1$  for bending; other factors equal to unity  $F_b' = F_b (C_D) (C_F) = 1900 \text{ psi} (1.25) (1.1) = 2613 \text{ psi}$  $F_t$ <sup> $\cdot$ </sup> =  $F_t$  (*C<sub>D</sub>*) = 1050 psi (1.25) = 1313 psi  $F_v' = F_v(C_D) = 175$  psi (1.25) = 219 psi  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 565 psi  $F_c' = F_c (C_D) = 1800 \text{ psi} (1.25) = 2250 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,800,000 \text{ psi}$  $E_{\min}$ <sup>'</sup> =  $E_{\min}$  = 660,000 psi c. **2 x 10 purlins** at 4 ft o.c.; D + L<sub>r</sub> load combination ( $C_D = 1.25$ )  $C_F$  is already incorporated in NDS Table 4B values; other factors equal to unity  $F_b' = F_b (C_D) = 2050 \text{ psi} (1.25) = 2563 \text{ psi}$  $F_t' = F_t(C_D) = 1100 \text{ psi} (1.25) = 1375 \text{ psi}$  $F_v' = F_v(C_D) = 175$  psi (1.25) = 219 psi  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 565 psi  $F_c' = F_c (C_D) = 1850 \text{ psi} (1.25) = 2313 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,800,000 \text{ psi}$  $E_{\min}$ <sup>'</sup> =  $E_{\min}$  = 660,000 psi d. **4 x 10 beams** at 4 ft o.c.;  $D + L + W$  load combination ( $C_D = 1.6$ )  $C_F = 1.1$  for bending; other factors equal to unity  $F_b' = F_b (C_D) (C_F) = 2050 \text{ psi} (1.6)(1.1) = 3608 \text{ psi}$  $F_t' = F_t(C_D) = 1100 \text{ psi} (1.6) = 1760 \text{ psi}$  $F_v' = F_v(C_D) = 175$  psi (1.6) = 280 psi  $F_c \perp$ <sup>2</sup> =  $F_c \perp$  = 565 psi  $F_c' = F_c (C_D) = 1850 \text{ psi} (1.6) = 2960 \text{ psi}$ Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member.  $E' = E = 1,800,000 \text{ psi}$  $E_{\min}$ <sup>'</sup> =  $E_{\min}$  = 660,000 psi

4.33 LRFD  $(K_F$  and  $\Phi$  apply) – Select Structural Southern Pine, fully braced; bending about strong axis

a. **2 x 6 joists** at 24 in. o.c.  $(C_r = 1.15)$ ; 1.2D + 1.6S load combination ( $\lambda = 0.8$ )  $C_F$  is already included in NDS Table 4B values; other factors equal to unity  $A = 8.25 \text{ in}^2$ ;  $S_x = 7.563 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 2550 psi (2.54) = 6477 psi = 6.48 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_r$ ) = 6.48 ksi (0.85)(0.8)(1.15) = 5.07 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 5.07$  ksi  $(7.563 \text{ in}^3) = 38.3 \text{ k-in.}$ Tension  $F_{tn} = F_t (K_F) = 1400 \text{ psi} (2.7) = 3780 \text{ psi} = 3.78 \text{ ksi}$  $F_{tn}$ <sup>\*</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ ) = 3.78 ksi (0.8)(0.8) = 2.42 ksi  $T_n' = F_{tn}$ <sup>2</sup> (*A*) = 2.42 ksi (8.25 in<sup>2</sup>) = 20.0 k Flexural Shear  $F_{vn} = F_v (K_F) = 175$  psi (2.88) = 504 psi = 0.504 ksi  $F_{vn}$ <sup>\*</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.504 ksi (0.75)(0.8) = 0.302 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.302 ksi (2/3)(8.25 in<sup>2</sup>) = 1.7 k Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 565$  psi (2.083) = 1177 psi = 1.177 ksi  $F_c \perp n' = F_c \perp n (\Phi_c)(\lambda) = 1.177$  ksi (0.9)(0.8) = 0.847 ksi Compression parallel to grain  $F_{cn} = F_c$  (*K<sub>F</sub>*) = 2000 psi (2.4) = 4800 psi = 4.80 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ ) = 4.80 ksi (0.9)(0.8) = 3.46 ksi  $P_n$ <sup>2</sup> =  $F_{cn}$ <sup>2</sup> (*A*) = 3.46 ksi (8.25 in<sup>2</sup>) = 28.5 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,800,000 \text{ psi} = 1800 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 660,000 psi (1.765) = 1,165,000 psi = 1165 ksi  $E_{\min n}$ <sup>'</sup> =  $E_{\min n} (\Phi_s) = 1165$  ksi (0.85) = 990 ksi b. **4 x 12 beam**;  $1.2D + 1.6L + 0.5L_f$  (occupancy) load combination ( $\lambda = 0.8$ )  $C_F = 1.1$  for bending; other factors equal to unity  $A = 39.38 \text{ in}^2$ ;  $S_x = 73.83 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 1900 psi (2.54) = 4826 psi = 4.83 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_F$ ) = 4.83 ksi (0.85)(0.8)(1.1) = 3.61 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 3.61$  ksi (73.83 in<sup>3</sup>) = 267 k-in. Tension  $F_{tn} = F_t (K_F) = 1050 \text{ psi} (2.7) = 2835 \text{ psi} = 2.84 \text{ ksi}$  $F_{tn}$ <sup>\*</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ ) = 2.84 ksi (0.8)(0.8) = 1.81 ksi  $T_n' = F_{tn}$ <sup>2</sup> (*A*) = 1.81 ksi (39.38 in<sup>2</sup>) = 71.5 k Flexural Shear  $F_{vn} = F_v (K_F) = 175$  psi (2.88) = 504 psi = 0.504 ksi  $F_{vn}$ <sup>\*</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.504 ksi (0.75)(0.8) = 0.302 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.302 ksi (2/3)(39.38 in<sup>2</sup>) = 7.9 k

Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 565$  psi (2.083) = 1177 psi = 1.177 ksi  $F_c \perp_n^{\bullet} = F_c \perp_n (\Phi_c)(\lambda) = 1.177$  ksi (0.9)(0.8) = 0.847 ksi Compression parallel to grain  $F_{cn} = F_c$  (*K<sub>F</sub>*) = 1800 psi (2.4) = 4320 psi = 4.32 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ ) = 4.32 ksi (0.9)(0.8) = 3.11 ksi  $P_n$ <sup>2</sup> =  $F_{cn}$ <sup>2</sup> (*A*) = 3.11 ksi (39.38 in<sup>2</sup>) = 122 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,800,000 \text{ psi} = 1800 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 660,000 psi (1.765) = 1,165,000 psi = 1165 ksi  $E_{\min-n}$ <sup>'</sup> =  $E_{\min-n}$  ( $\Phi_s$ ) = 1165 ksi (0.85) = 990 ksi c. **2 x 10 purlins** at 4 ft o.c.;  $1.2D + 1.6L_r$  load combination ( $\lambda = 0.8$ )  $C_F$  is already incorporated in NDS Table 4B values; other factors equal to unity  $A = 13.88 \text{ in}^2$ ;  $S_x = 21.39 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 2050 psi (2.54) = 5207 psi = 5.21 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ ) = 5.21 ksi (0.85)(0.8) = 3.54 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 3.54$  ksi  $(21.39 \text{ in}^3) = 75.8$  k-in. Tension  $F_m = F_t (K_F) = 1100 \text{ psi} (2.7) = 2970 \text{ psi} = 2.97 \text{ ksi}$  $F_{tn}$ <sup>\*</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ ) = 2.97 ksi (0.8)(0.8) = 1.90 ksi  $T_n' = F_{tn}$ <sup>2</sup> (*A*) = 1.90 ksi (13.88 in<sup>2</sup>) = 26.4 k Flexural Shear  $F_{vn} = F_v (K_F) = 175$  psi (2.88) = 504 psi = 0.504 ksi  $F_{vn}$ <sup>\*</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.504 ksi (0.75)(0.8) = 0.302 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.302 ksi (2/3)(13.88 in<sup>2</sup>) = 2.8 k Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 565$  psi (2.083) = 1177 psi = 1.177 ksi  $F_c \perp n' = F_c \perp n (\Phi_c)(\lambda) = 1.177$  ksi (0.9)(0.8) = 0.847 ksi Compression parallel to grain  $F_{cn} = F_c$  (*K<sub>F</sub>*) = 1850 psi (2.4) = 4440 psi = 4.44 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ ) = 4.44 ksi (0.9)(0.8) = 3.20 ksi  $P_n' = F_{cn}$ <sup>2</sup> (*A*) = 3.20 ksi (13.88 in<sup>2</sup>) = 44.4 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,800,000 \text{ psi} = 1800 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 660,000 psi (1.765) = 1,165,000 psi = 1165 ksi  $E_{\min n}$ <sup>'</sup> =  $E_{\min n} (\Phi_s) = 1165$  ksi (0.85) = 990 ksi

d. **4 x 10 beams** at 4 ft o.c.;  $1.2D + 1.6W + L$  load combination ( $\lambda = 1.0$ )  $C_F = 1.1$  for bending; other factors equal to unity  $A = 32.38 \text{ in}^2$ ;  $S_x = 49.91 \text{ in}^3$ Bending moment  $F_{bn} = F_b$  ( $K_F$ ) = 2050 psi (2.54) = 5207 psi = 5.21 ksi  $F_{bn}$ <sup>\*</sup> =  $F_{bn}$  ( $\Phi_b$ )( $\lambda$ )( $C_F$ ) = 5.21 ksi (0.85)(1.0)(1.1) = 4.87 ksi  $M_n' = F_{bn}$ ,  $(S_x) = 4.87$  ksi (49.91 in<sup>3</sup>) = 243 k-in. Tension  $F_{tn} = F_t (K_F) = 1100 \text{ psi} (2.7) = 2970 \text{ psi} = 2.97 \text{ ksi}$  $F_{tn}$ <sup>'</sup> =  $F_{tn}$  ( $\Phi_t$ )( $\lambda$ ) = 2.97 ksi (0.8)(1.0) = 2.38 ksi  $T_n' = F_{tn}$ <sup>2</sup> (*A*) = 2.38 ksi (32.38 in<sup>2</sup>) = 76.9 k Flexural Shear  $F_{vn} = F_v (K_F) = 175$  psi (2.88) = 504 psi = 0.504 ksi  $F_{vn}$ <sup>2</sup> =  $F_{vn}$  ( $\Phi_v$ )( $\lambda$ ) = 0.504 ksi (0.75)(1.0) = 0.378 ksi  $V_n' = F_{vn}$ <sup>2</sup> (2/3)(*A*) = 0.378 ksi (2/3)(32.38 in<sup>2</sup>) = 8.2 k Compression perpendicular to grain (bearing)  $F_c \perp_n = F_c \perp (K_F) = 565$  psi (2.083) = 1177 psi = 1.177 ksi  $F_c \perp_n^{\bullet} = F_c \perp_n (\Phi_c)(\lambda) = 1.177$  ksi  $(0.9)(1.0) = 1.06$  ksi Compression parallel to grain  $F_{cn} = F_c$  ( $K_F$ ) = 1850 psi (2.4) = 4440 psi = 4.44 ksi  $F_{cn}$ <sup>\*</sup> =  $F_{cn}$  ( $\Phi_c$ )( $\lambda$ ) = 4.44 ksi (0.9)(1.0) = 4.00 ksi  $P_n$ <sup>2</sup> =  $F_{cn}$ <sup>2</sup> (*A*) = 4.00 ksi (32.38 in<sup>2</sup>) = 129 k Note that C<sub>P</sub> would also apply for column buckling of an axially loaded member. Modulus of Elasticity  $E' = E = 1,800,000 \text{ psi} = 1800 \text{ ksi}$  $E_{\min-n} = E_{\min}$  ( $K_F$ ) = 660,000 psi (1.765) = 1,165,000 psi = 1165 ksi  $E_{\min-n}$ <sup>'</sup> =  $E_{\min-n}$  ( $\Phi_s$ ) = 1165 ksi (0.85) = 990 ksi

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4.34 
$$
SV = \frac{6}{30} = 0.2\%
$$
 per 1% change in MC = 0.002 in./in./percent change in MC  
\n $\Delta MC = 10 - 19 = -9$  (drying)  
\nShrinkage = (SV)( $\Delta MC$ )  $d = (0.002)(-9)(13.25 \text{ in.}) = 0.24 \text{ in.}$   
\nDepth after shrinkage = 13.25 - 0.24 in. = 13 in.

4.35  $SV = \frac{6}{10} = 0.2\%$ 30  $SV = \frac{6}{100} = 0.2\%$  per 1% change in MC = 0.002 in./in./percent change in MC  $\Delta MC = 9 - 19 = -10$  (drying) Shrinkage in 2 x 10 = (SV)( $\Delta$ MC)  $d = (0.002)(-10)(9.25 \text{ in.}) = 0.185 \text{ in.}$ Shrinkage in 2 x plate = (SV)( $\Delta$ MC)  $d = (0.002)(-10)(1.5 \text{ in.}) = 0.03 \text{ in.}$  three 2 x 12's; twelve 2 x plates Total shrinkage =  $3(0.185 \text{ in.}) + 12(0.03 \text{ in.}) = 0.9 \text{ in.}$