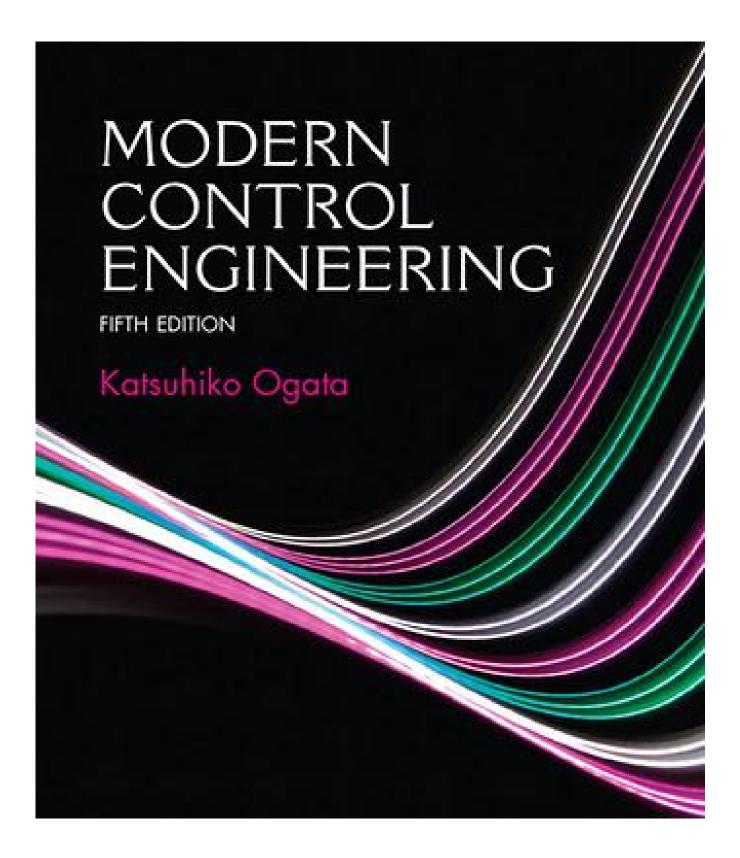
## SOLUTION MANUAL



### SOLUTIONS MANUAL FOR

# MODERN CONTROL ENGINEERING

5th Edition

(2010)

by

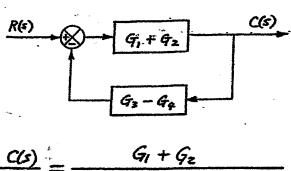
Katsuhiko Ogata

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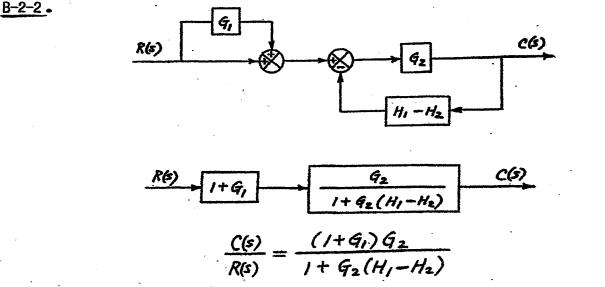
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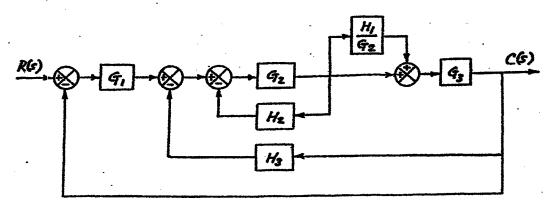
### **CHAPTER 2**

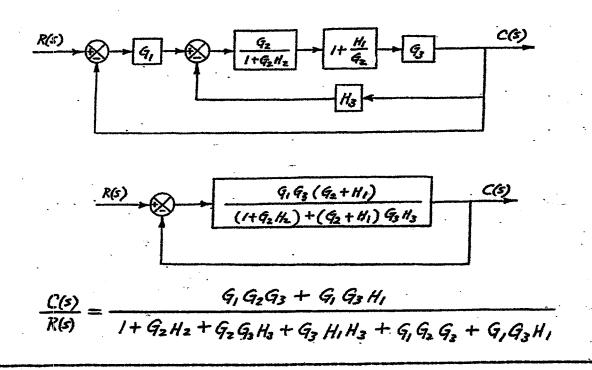


$$\frac{C(s)}{R(s)} = \frac{G_1 + G_2}{1 + (G_1 + G_2)(G_3 - G_4)}$$

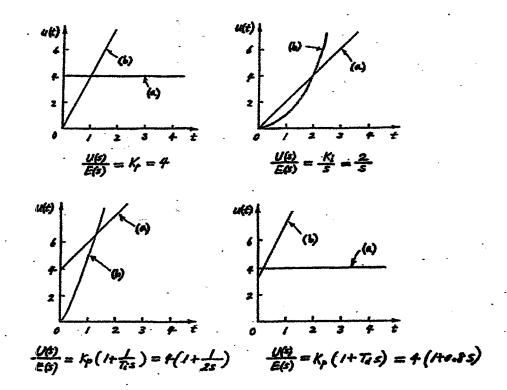


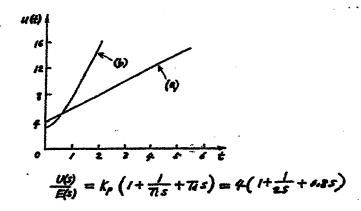
B-2-3





B-2-4. In the following diagrams, (a) denotes the unit-step response and (b) corresponds to the unit-ramp response.





B-2-5. When D(s) is zero, the closed-loop transfer function  $C_R(s)/R(s)$  is

$$\frac{C_R(s)}{R(s)} = \frac{G_C(s) G_P(s)}{1 + G_C(s) G_P(s)}$$

When R(s) = 0, the closed-loop transfer function  $C_D(s)/D(s)$  is

$$\frac{C_{D}(5)}{D(5)} = \frac{1}{1 + G_{C}(5) G_{D}(5)}$$

When both the reference input and disturbance input are present, the output C(s) is the sum of  $C_R(s)$  and  $C_D(s)$ . Hence

$$C(s) = C_{R}(s) + C_{D}(s) = \frac{1}{1 + G_{C}(s)G_{P}(s)} \left[ G_{C}(s)G_{P}(s)R(s) + D(s) \right]$$

B-2-6. When only the reference input R(s) is present, the output  $C_R(s)$  is given

$$\frac{C_{R}(s)}{R(s)} = \frac{G_{1}(s) G_{2}(s)}{(1 + G_{1}(s) G_{2}(s))}$$

For the reference input R(s), the desired output is R(s) for the unity-feedback system such as the present system. Thus, the error  $E_R(s)$  is the difference between R(s) and the actual output  $C_R(s)$ . The error  $E_R(s)$  is given by

$$E_{R}(s) = R(s) - C_{R}(s) = R(s) \left[ 1 - \frac{C_{R}(s)}{R(s)} \right]$$

$$= R(s) \left[ 1 - \frac{G_{1}(s) G_{2}(s)}{1 + G_{1}(s) G_{2}(s)} \right] = \frac{1}{1 + G_{1}(s) G_{2}(s)} R(s)$$

Assuming the system to be stable, the steady-state error  $e_{SSR}(t)$  can be given

by  $C_{SSR}(t) = \lim_{R \to \infty} C_{R}(t) = \lim_{R \to \infty} SE_{R}(t) = \lim_{R \to \infty} \frac{SR(t)}{1 + G_{R}(t)G_{R}(t)}$  © 2010 Pearson Education, Inc., Upper Saddle River, NJ. All rights reserved. This publication is protected by Copyright and Written permission obtained from the publisher prior to any prohibited reproduction, storage in a rotrious system.

When only the disturbance input D(s) is present, the output  $C_D(s)$  is given by

$$\frac{C_0(5)}{D(5)} = \frac{G_2(5)}{1 + G_1(5)G_2(5)}$$

Since the desired output to the disturbance input D(s) is zero, the error  $E_{\overline{D}}(s)$  can be given by

$$E_0(s) = 0 - C_0(s) = -C_0(s)$$

Hence

$$E_D(s) = -C_D(s) = -\frac{G_2(s)}{1 + G_1(s) G_2(s)}D(s)$$

For the stable system, the steady-state error essD(t) is given by

$$C_{SSD}(t) = \lim_{t \to \infty} C_D(t) = \lim_{s \to 0} S_D(s) = \lim_{s \to 0} \frac{-s G_2(s) D(s)}{1 + G_1(s)G_2(s)}$$

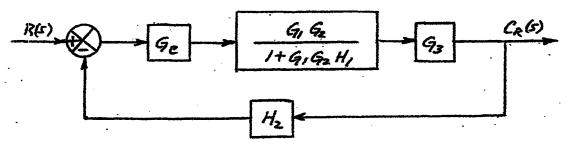
The steady-state error when both the reference input R(s) and disturbance input D(s) are present is the sum of  $e_{SSR}(t)$  and  $e_{SSD}(t)$  and is given by

$$e_{ss(t)} = e_{ssR}(t) + e_{ssp}(t)$$

$$= \lim_{s \to 0} \left[ \frac{sR(s)}{1 + G_1(s) G_2(s)} - \frac{sG_2(s)D(s)}{1 + G_1(s)G_2(s)} \right]$$

$$= \lim_{s \to 0} \left\{ \frac{s}{1 + G_1(s)G_2(s)} \left[ R(s) - G_2(s)D(s) \right] \right\}$$

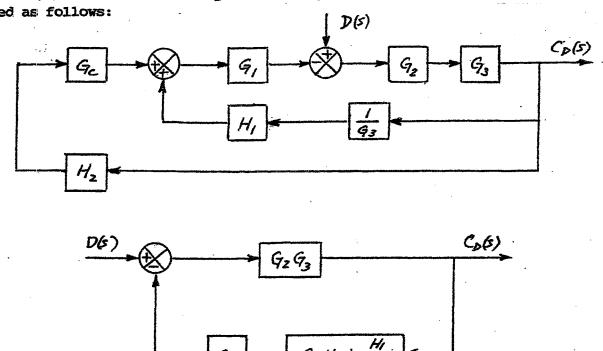
B-2-7. When D(s) = 0, the block diagram of the system can be simplified as follows:



The closed-loop transfer function  $C_R(s)/R(s)$  can be given by

$$\frac{C_{R}(5)}{R(5)} = \frac{\frac{G_{c}G_{1}G_{2}G_{3}}{1+G_{1}G_{2}H_{1}}}{\frac{G_{c}G_{1}G_{2}G_{3}H_{2}}{1+G_{1}G_{2}G_{3}H_{2}}} = \frac{G_{c}G_{1}G_{2}G_{3}}{1+G_{1}G_{2}G_{3}H_{2}}$$

When R(s) = 0, the block diagram of the system shown in Figure 2-34 can be modified as follows:



Hence

$$\frac{C_D(5)}{D(5)} = \frac{G_2 G_3}{1 + G_2 G_3 G_1 \left(G_C H_2 + \frac{H_1}{G_3}\right)} = \frac{G_2 G_3}{1 + G_1 G_2 G_3 G_C H_2 + G_1 G_2 H_1}$$

B-2-8. There are infinitely many state-space representations for this system. We shall give two of the possible state-space representations.

State-space representation 1: From Figure 2-35, we obtain

$$\frac{\gamma(s)}{T(s)} = \frac{\frac{s+1}{s+p} \frac{1}{s^2}}{1 + \frac{s+p}{s+p} \frac{1}{s^2}} = \frac{s+p}{s^2 + s + 2}$$

which is equivalent to

Comparing this equation with the standard equation

$$\ddot{y} + q_1 \ddot{y} + a_2 \dot{y} + a_3 y = b_1 \ddot{u} + b_1 \ddot{u} + b_2 \ddot{u} + b_3 u$$
we obtain

$$a_1=p$$
,  $a_2=l$ ,  $a_3=2$ ,  $b_0=0$ ,  $b_1=0$ ,  $b_2=l$ ,  $b_3=2$ 

Define

$$x_1 = y - \beta_0 u$$
  
 $x_2 = \dot{y} - \beta_0 \dot{u} - \beta_1 u = \dot{x}_1 - \beta_1 u$   
 $x_3 = \dot{x}_2 - \beta_2 u$ 

where

$$\beta_0 = b_0 = 0$$
 $\beta_1 = b_1 - a_1 \beta_0 = 0$ 
 $\beta_2 = b_2 - a_1 \beta_1 - a_2 \beta_0 = 1$ 

Also, define

Then, state-space equations can be given by

$$\begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -z & -1 & -p \end{bmatrix} \begin{bmatrix} z_1 \\ x_2 \\ \beta_3 \end{bmatrix} + \begin{bmatrix} \beta_1 \\ \beta_2 \\ \beta_3 \end{bmatrix} \mathcal{U}$$

$$\mathbf{y} = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \\ \beta_3 \end{bmatrix} + \beta_0 \mathcal{U}$$

$$\mathbf{y} = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \\ z_3 \end{bmatrix} + \beta_0 \mathcal{U}$$

or

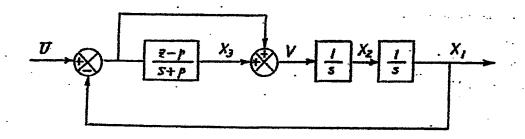
$$\begin{bmatrix} \dot{x}_{1} \\ \dot{x}_{2} \end{bmatrix} = \begin{bmatrix} 0 & / & 0 \\ 0 & 0 & / \\ -\bar{z} & -/ & -P \end{bmatrix} \begin{bmatrix} x_{1} \\ x_{2} \\ x_{3} \end{bmatrix} + \begin{bmatrix} 0 \\ / \\ \bar{z} - P \end{bmatrix} \mathcal{X}$$

$$y = \begin{bmatrix} / & 0 & 0 \end{bmatrix} \begin{bmatrix} x_{1} \\ x_{2} \\ x_{3} \end{bmatrix}$$

State-space representation 2: Since

$$\frac{5+2}{5+p} = \frac{5+p+2-p}{5+p} = 1 + \frac{2-p}{5+p}$$

we can redraw the block diagram as shown below.



From this block disgram we get the following equations:

$$V = \overline{U} - X_1 + X_3$$

$$\frac{X_3}{\overline{U} - X_1} = \frac{\overline{z} - p}{\overline{s} + p}$$

$$\frac{X_2}{\overline{U} - X_1 + X_3} = \frac{1}{s}$$

$$\frac{X_1}{X_2} = \frac{1}{s}$$

from which we obtain

$$\dot{x}_s + px_s = (z-p)u - (z-p)x_j$$

$$\dot{x}_s = -x_1 + x_3 + u$$

$$\dot{x}_i = x_s$$

Rewriting, we have

$$\dot{x}_1 = x_2$$
 $\dot{x}_2 = -x_1 + x_3 + u$ 

$$\dot{x}_3 = -(z-p)x_1 - px_3 + (z-p)u$$
 $\dot{y} = x_1$ 

or

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & / & 0 \\ -/ & 0 & / \\ p-z & 0 & -p \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ / \\ z-p \end{bmatrix} u$$

$$\mathcal{J} = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

B-2-9.

$$\ddot{y} + 3\ddot{y} + 2\dot{y} = \mu$$

Define

$$x_1 = y$$

$$x_2 = y$$

$$x_3 = y$$

Then

$$\dot{x}_3 + 3x_3 + 2x_2 = u$$

$$\dot{x}_2 = x_3$$

$$\dot{x}_1 = x_2$$

or

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{z}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

B-2-10

$$A = \begin{bmatrix} -4 & -1 \\ 3 & -1 \end{bmatrix}, \quad B = \begin{bmatrix} 1 \\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 0 \end{bmatrix}$$

The transfer function G(s) of the system is given by

$$q(s) = C(sI - A)^{-1}B = [1 \ o]\begin{bmatrix} s+4 \ -3 \ s+1 \end{bmatrix}^{-1}\begin{bmatrix} 1 \ 1 \end{bmatrix}$$

$$= [/ o] \frac{/}{(s+4)(s+1)+3} \begin{bmatrix} s+/ & -/ \\ 3 & s+4 \end{bmatrix} [/]$$

$$= \frac{/}{s^2 + ss + 7} [/ o] \begin{bmatrix} s \\ s+7 \end{bmatrix}$$

$$= \frac{s}{s^2 + ss + 7}$$

$$A = \begin{bmatrix} -s & -1 \\ 3 & -1 \end{bmatrix}, B = \begin{bmatrix} 2 \\ s \end{bmatrix}, C = \begin{bmatrix} 1 & 2 \end{bmatrix}$$

The transfer function G(s) of the system is given by

$$G(s) = C(sI-A)^{-1}B = \begin{bmatrix} 1 & 2 \end{bmatrix} \begin{bmatrix} s+s & 1 \\ -3 & s+1 \end{bmatrix}^{-1} \begin{bmatrix} 2 \\ s \end{bmatrix}$$

$$= \begin{bmatrix} 1 & 2 \end{bmatrix} \frac{1}{(s+s)(s+1)+3} \begin{bmatrix} s+1 & -1 \\ 3 & s+s \end{bmatrix} \begin{bmatrix} 2 \\ s \end{bmatrix}$$

$$= \frac{1}{s^2 + 6s + 8} \begin{bmatrix} 1 & 2 \end{bmatrix} \begin{bmatrix} 2s-3 \\ ss+31 \end{bmatrix} = \frac{12s + 59}{s^2 + 6s + 8}$$

A MATLAB solution to this problem is given below.

$$A = \begin{bmatrix} 0 & i & 0 \\ 0 & 0 & i \\ -2 & -4 & -6 \end{bmatrix}, B = \begin{bmatrix} 0 & 0 \\ 0 & i \\ 0 & i \end{bmatrix}, C = \begin{bmatrix} i & 0 & 0 \\ 0 & i & 0 \end{bmatrix}$$

The transfer matrix of the system can be given by

$$G(s) = C(sI - A)^{-1}B = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} s & -1 & 0 \\ 0 & s & -1 \\ 2 & 4 & s+6 \end{bmatrix} \begin{bmatrix} 0 & 0 \\ 0 & 1 \\ 1 & 0 \end{bmatrix}$$

$$= \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \end{bmatrix} \frac{1}{s^3 + 6s^2 + 4s + 2} \begin{bmatrix} s^2 + s + 4 & s + 6 & 1 \\ -2 & s^2 + 6s & s \\ -2s & -4s - 2 & s^2 \end{bmatrix} \begin{bmatrix} 0 & 0 \\ 0 & 1 \\ 1 & 0 \end{bmatrix}$$

$$= \frac{1}{s^3 + 6s^2 + 4s + 2} \begin{bmatrix} 1 & s + 6 \\ s & s^2 + 6s \end{bmatrix}$$

$$= \begin{bmatrix} \frac{1}{s^3 + 6s^2 + 4s + 2} & \frac{s^2 + 6s}{s^3 + 6s^2 + 4s + 2} \\ \frac{s}{s^3 + 6s^2 + 4s + 2} & \frac{s^2 + 6s}{s^3 + 6s^2 + 4s + 2} \end{bmatrix}$$

A MATTAB solution to this problem is given below.

B-2-13. Define

$$Z = \chi^2 + 8\chi y + 3y^2 = f(\chi, y)$$
  
 $2 \le \chi \le 4$ ,  $10 \le y \le 12$ 

Let us choose  $\bar{x} = 3$  and  $\bar{y} = 11$ . Then

$$\overline{z} = \overline{x}^2 + 8\overline{x}\overline{y} + 3\overline{y}^2 = 9 + 264 + 363 = 636$$

We shall obtain a linearized equation for the nonlinear equation near the point  $\bar{x}=3$ ,  $\bar{y}=11$ . Expanding the nonlinear equation into a Taylor series about point  $x=\bar{x}$ ,  $y=\bar{y}$  and neglecting the higher-order terms, we obtain

$$Z - \overline{Z} = K_1(x - \overline{x}) + K_2(y - \overline{y})$$

where

$$K_{1} = \frac{\partial f}{\partial x}\Big|_{x=\overline{x}, \ y=\overline{y}} = 2\overline{x} + 8\overline{y}\Big|_{\overline{x}=3, \ \overline{y}=11} = 6 + 88 = 94$$

$$K_{2} = \frac{\partial f}{\partial y}\Big|_{x=\overline{x}, \ y=\overline{y}} = 8\overline{x} + 6\overline{y}\Big|_{\overline{x}=3, \ \overline{y}=11} = 24 + 66 = 90$$

Hence the linearized equation is

$$2-636 = 94(x-3) + 90(y-11)$$

or

$$2 = 94x + 90y - 636$$

B-2-14. Define

$$y = 0.2 x^3 = f(x)$$
,  $\bar{x} = 2$ 

Then

$$\mathcal{Y} = f(x) = f(x) + \frac{\partial f}{\partial x} (x - \overline{x}) + \cdots$$

Since the higher-order terms in this equation are small, neglecting those terms, we obtain

$$4-f(\overline{x})=0.6\,\overline{x}^2(x-\overline{x})$$

or

$$y - 0.2 \times 2^2 = 0.6 \times 2^2 (x-2)$$

Thus, linear approximation of the given nonlinear equation near the operating point is

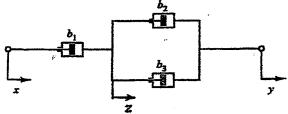
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#### CHAPTER 3

B-3-1. Since the same force transmits the shaft, we have

$$f = b_1(\dot{z} - \dot{z}) = b_2(\dot{y} - \dot{z}) + b_3(\dot{y} - \dot{z})$$
 (1)

where displacement z is defined in the figure below.



In terms of the equivalent viscous friction coefficient, the force f is given by

$$f = b_{xy} (\dot{y} - \dot{x}) \tag{2}$$

From Equation (1) we have

or

$$\dot{z} = \frac{1}{b_1 + b_2 + b_3} \left[ b_1 \dot{z} + (b_2 + b_3) \dot{y} \right]$$
 (3)

By substituting Equation (3) into Equation (1), we have

$$f = b_1(\dot{z} - \dot{x}) = b_1 \left\{ \frac{1}{b_1 + b_2 + b_3} \left[ b_1 \dot{x} + (b_2 + b_3) \dot{y} \right] - \dot{x} \right\}$$

$$= b_1 \frac{b_2 + b_3}{b_1 + b_2 + b_3} (\dot{y} - \dot{x})$$
(4)

Hence, by comparing Equations (2) and (4), we obtain

$$beg = b_1 \frac{b_2 + b_3}{b_1 + b_2 + b_3} = \frac{1}{\frac{1}{b_2 + b_3} + \frac{1}{b_1}}$$

B-3-2

$$m\ddot{x} + kx = u$$

$$\frac{\chi(s)}{U(s)} = \frac{1}{ms^2 + k}$$