CHAPTER 2


## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:

$$
R=1391 \mathrm{kN}, \quad \alpha=47.8^{\circ}
$$

$$
\mathbf{R}=1391 \mathrm{~N}<^{\prime} 47.8^{\circ}
$$



## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:

$$
R=906 \mathrm{lb}, \quad \alpha=26.6^{\circ}
$$

$$
R=906 \mathrm{lb} \measuredangle 26.6^{\circ}
$$



## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:

$$
R=20.1 \mathrm{kN}, \quad \alpha=21.2^{\circ}
$$

$\mathbf{R}=20.1 \mathrm{kN}>21.2^{\circ}$


## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:
$R=8.03 \mathrm{kips}, \quad \alpha=3.8^{\circ}$
$\mathbf{R}=8.03 \mathrm{kips} \quad 7.8^{\circ}$


## SOLUTION



Using the triangle rule and the law of sines:
(a)

$$
\frac{120 \mathrm{~N}}{\sin 30^{\circ}}=\frac{P}{\sin 25^{\circ}}
$$

$$
P=101.4 \mathrm{~N}
$$

(b)

$$
\begin{array}{rlr}
30^{\circ}+\beta+25^{\circ} & =180^{\circ} \\
\beta & =180^{\circ}-25^{\circ}-30^{\circ} & \\
& =125^{\circ} & \\
\frac{120 \mathrm{~N}}{\sin 30^{\circ}} & =\frac{R}{\sin 125^{\circ}} & R=196.6 \mathrm{~N}
\end{array}
$$



## PROBLEM 2.6

A telephone cable is clamped at $A$ to the pole $A B$. Knowing that the tension in the left-hand portion of the cable is $T_{1}=800 \mathrm{lb}$, determine by trigonometry $(a)$ the required tension $T_{2}$ in the right-hand portion if the resultant $\mathbf{R}$ of the forces exerted by the cable at $A$ is to be vertical, (b) the corresponding magnitude of $\mathbf{R}$.

## SOLUTION



Using the triangle rule and the law of sines:
(a)

$$
\begin{aligned}
75^{\circ}+40^{\circ}+\alpha & =180^{\circ} \\
\alpha & =180^{\circ}-75^{\circ}-40^{\circ} \\
& =65^{\circ}
\end{aligned}
$$

$$
\frac{800 \mathrm{lb}}{\sin 65^{\circ}}=\frac{T_{2}}{\sin 75^{\circ}}
$$

$$
T_{2}=853 \mathrm{lb}
$$

(b)

$$
\frac{800 \mathrm{lb}}{\sin 65^{\circ}}=\frac{R}{\sin 40^{\circ}} \quad R=567 \mathrm{lb}
$$



## PROBLEM 2.7

A telephone cable is clamped at $A$ to the pole $A B$. Knowing that the tension in the right-hand portion of the cable is $T_{2}=1000 \mathrm{lb}$, determine by trigonometry $(a)$ the required tension $T_{1}$ in the left-hand portion if the resultant $\mathbf{R}$ of the forces exerted by the cable at $A$ is to be vertical, (b) the corresponding magnitude of $\mathbf{R}$.

## SOLUTION



Using the triangle rule and the law of sines:
(a)
(b)

$$
\begin{aligned}
75^{\circ}+40^{\circ}+\beta & =180^{\circ} \\
\beta=180^{\circ}-75^{\circ}-40^{\circ} & \\
& =65^{\circ} \\
\frac{1000 \mathrm{lb}}{\sin 75^{\circ}}=\frac{T_{1}}{\sin 65^{\circ}} & T_{1}=938 \mathrm{lb} \\
\frac{1000 \mathrm{lb}}{\sin 75^{\circ}}=\frac{R}{\sin 40^{\circ}} & R=665 \mathrm{lb}
\end{aligned}
$$

## PROBLEM 2.8

A disabled automobile is pulled by means of two ropes as shown. The tension in rope $A B$ is 2.2 kN , and the angle $\alpha$ is $25^{\circ}$. Knowing that the resultant of the two forces applied at $A$ is directed along the axis of the automobile, determine by trigonometry (a) the tension in rope $A C$, (b) the magnitude of the resultant of the two forces applied at $A$.

## SOLUTION



Using the law of sines:

$$
\begin{aligned}
\frac{T_{A C}}{\sin 30^{\circ}} & =\frac{R}{\sin 125^{\circ}}=\frac{2.2 \mathrm{kN}}{\sin 25^{\circ}} \\
T_{A C} & =2.603 \mathrm{kN} \\
R & =4.264 \mathrm{kN}
\end{aligned}
$$

(a)
$T_{A C}=2.60 \mathrm{kN}$
(b)

$$
R=4.26 \mathrm{kN}
$$



Using the law of cosines:

$$
\begin{aligned}
T_{A C}^{2} & =(3 \mathrm{kN})^{2}+(4.8 \mathrm{kN})^{2}-2(3 \mathrm{kN})(4.8 \mathrm{kN}) \cos 30^{\circ} \\
T_{A C} & =2.6643 \mathrm{kN}
\end{aligned}
$$

Using the law of sines: $\quad \frac{\sin \alpha}{3 \mathrm{kN}}=\frac{\sin 30^{\circ}}{2.6643 \mathrm{kN}}$
$\alpha=34.3^{\circ}$

$$
\mathbf{T}_{A C}=2.66 \mathrm{kN} \nabla 34.3^{\circ}
$$



## SOLUTION

Using the triangle rule and law of sines:
(a)

$$
\begin{aligned}
\frac{\sin \alpha}{50 \mathrm{~N}} & =\frac{\sin 25^{\circ}}{35 \mathrm{~N}} \\
\sin \alpha & =0.60374 \\
\alpha & =37.138^{\circ}
\end{aligned}
$$


(b)

$$
\begin{array}{rlr}
\alpha+\beta+25^{\circ} & =180^{\circ} \\
\beta & =180^{\circ}-25^{\circ}-37.138^{\circ} \\
& =117.862^{\circ} \\
\frac{R}{\sin 117.862^{\circ}} & =\frac{35 \mathrm{~N}}{\sin 25^{\circ}} \quad R=73.2 \mathrm{~N}
\end{array}
$$



## SOLUTION



Using the triangle rule and the law of sines:
(a)

$$
\begin{aligned}
\beta+50^{\circ}+60^{\circ} & =180^{\circ} \\
\beta & =180^{\circ}-50^{\circ}-60^{\circ} \\
& =70^{\circ}
\end{aligned}
$$

$$
\frac{425 \mathrm{lb}}{\sin 70^{\circ}}=\frac{P}{\sin 60^{\circ}}
$$

$$
P=392 \mathrm{lb}
$$

$$
\frac{425 \mathrm{lb}}{\sin 70^{\circ}}=\frac{R}{\sin 50^{\circ}}
$$

$$
R=346 \mathrm{lb}
$$



## PROBLEM 2.12

A steel tank is to be positioned in an excavation. Knowing that the magnitude of $\mathbf{P}$ is 500 lb , determine by trigonometry ( $a$ ) the required angle $\alpha$ if the resultant $\mathbf{R}$ of the two forces applied at $A$ is to be vertical, (b) the corresponding magnitude of $\mathbf{R}$.

## SOLUTION



Using the triangle rule and the law of sines:
(a)

$$
\begin{aligned}
\left(\alpha+30^{\circ}\right)+60^{\circ}+\beta & =180^{\circ} \\
\beta & =180^{\circ}-\left(\alpha+30^{\circ}\right)-60^{\circ} \\
\beta & =90^{\circ}-\alpha \\
\frac{\sin \left(90^{\circ}-\alpha\right)}{425 \mathrm{lb}} & =\frac{\sin 60^{\circ}}{500 \mathrm{lb}}
\end{aligned}
$$

$$
90^{\circ}-\alpha=47.402^{\circ}
$$

$$
\alpha=42.6^{\circ}
$$

(b)

$$
\frac{R}{\sin \left(42.598^{\circ}+30^{\circ}\right)}=\frac{500 \mathrm{lb}}{\sin 60^{\circ}}
$$

$$
R=551 \mathrm{lb}
$$



## PROBLEM 2.13

A steel tank is to be positioned in an excavation. Determine by trigonometry $(a)$ the magnitude and direction of the smallest force $\mathbf{P}$ for which the resultant $\mathbf{R}$ of the two forces applied at $A$ is vertical, (b) the corresponding magnitude of $\mathbf{R}$.

## SOLUTION



The smallest force $P$ will be perpendicular to $R$.
(a) $P=(425 \mathrm{lb}) \cos 30^{\circ}$
(b) $R=(425 \mathrm{lb}) \sin 30^{\circ}$

$$
\begin{array}{r}
\mathbf{P}=368 \mathrm{lb} \longrightarrow \\
R=213 \mathrm{lb}
\end{array}
$$



## SOLUTION



The smallest force $P$ will be perpendicular to $R$.
(a)
$P=(50 \mathrm{~N}) \sin 25^{\circ}$
(b)
$R=(50 \mathrm{~N}) \cos 25^{\circ}$

$$
\begin{gathered}
\mathbf{P}=21.1 \mathrm{~N} \downarrow \\
R=45.3 \mathrm{~N}
\end{gathered}
$$



## SOLUTION



Using the law of cosines:

$$
\begin{aligned}
R^{2}= & (200 \mathrm{lb})^{2}+(300 \mathrm{lb})^{2} \\
& -2(200 \mathrm{lb})(300 \mathrm{lb}) \cos \left(45^{\circ}+65^{\circ}\right) \\
R= & 413.57 \mathrm{lb}
\end{aligned}
$$

Using the law of sines:

$$
\begin{gathered}
\frac{\sin \alpha}{300 \mathrm{lb}}=\frac{\sin \left(45^{\circ}+65^{\circ}\right)}{413.57 \mathrm{lb}} \\
\alpha=42.972^{\circ} \\
\beta=90^{\circ}+25^{\circ}-42.972^{\circ} \quad \mathbf{R}=414 \mathrm{lb} \text { ■ } 72.0^{\circ}
\end{gathered}
$$

## PROBLEM 2.16



Solve Prob. 2.1 by trigonometry.

## PROBLEM 2.1

Two forces are applied as shown to a hook. Determine graphically the magnitude and direction of their resultant using (a) the parallelogram law, (b) the triangle rule.

## SOLUTION



Using the law of cosines:

$$
\begin{aligned}
R^{2}= & (900 \mathrm{~N})^{2}+(600 \mathrm{~N})^{2} \\
& -2(900 \mathrm{~N})(600 \mathrm{~N}) \cos \left(135^{\circ}\right) \\
R= & 1390.57 \mathrm{~N}
\end{aligned}
$$

Using the law of sines:

$$
\begin{aligned}
\frac{\sin \left(\alpha-30^{\circ}\right)}{600 \mathrm{~N}} & =\frac{\sin \left(135^{\circ}\right)}{1390.57 \mathrm{~N}} \\
\alpha-30^{\circ} & =17.7642^{\circ} \\
\alpha & =47.764^{\circ}
\end{aligned}
$$

$$
\mathbf{R}=1391 \mathrm{~N}, ~ 47.8^{\circ}
$$



## SOLUTION

Using the force triangle and the laws of cosines and sines:
We have:

$$
\begin{aligned}
\gamma & =180^{\circ}-\left(50^{\circ}+25^{\circ}\right) \\
& =105^{\circ}
\end{aligned}
$$



Then

$$
\begin{aligned}
R^{2} & =(4 \mathrm{kips})^{2}+(6 \mathrm{kips})^{2}-2(4 \mathrm{kips})(6 \mathrm{kips}) \cos 105^{\circ} \\
& =64.423 \mathrm{kips}^{2} \\
R & =8.0264 \mathrm{kips}
\end{aligned}
$$

And

$$
\begin{aligned}
\frac{4 \mathrm{kips}}{\sin \left(25^{\circ}+\alpha\right)} & =\frac{8.0264 \mathrm{kips}}{\sin 105^{\circ}} \\
\sin \left(25^{\circ}+\alpha\right) & =0.48137 \\
25^{\circ}+\alpha & =28.775^{\circ} \\
\alpha & =3.775^{\circ}
\end{aligned}
$$

$$
\mathbf{R}=8.03 \mathrm{kips} \bar{y} 3.8^{\circ}
$$



## PROBLEM 2.18

For the stake of Prob. 2.5, knowing that the tension in one rope is 120 N , determine by trigonometry the magnitude and direction of the force $\mathbf{P}$ so that the resultant is a vertical force of 160 N .

PROBLEM 2.5 A stake is being pulled out of the ground by means of two ropes as shown. Knowing that $\alpha=30^{\circ}$, determine by trigonometry (a) the magnitude of the force $\mathbf{P}$ so that the resultant force exerted on the stake is vertical, (b) the corresponding magnitude of the resultant.

## SOLUTION



Using the laws of cosines and sines:

$$
P^{2}=(120 \mathrm{~N})^{2}+(160 \mathrm{~N})^{2}-2(120 \mathrm{~N})(160 \mathrm{~N}) \cos 25^{\circ}
$$

$$
\begin{aligned}
& \qquad P=72.096 \mathrm{~N} \\
& \qquad \begin{aligned}
& \sin \alpha \\
& 120 \mathrm{~N}=\frac{\sin 25^{\circ}}{72.096 \mathrm{~N}} \\
& \sin \alpha=0.70343 \\
& \alpha=44.703^{\circ}
\end{aligned}
\end{aligned}
$$

$$
\mathbf{P}=72.1 \mathrm{~N} b 44.7^{\circ}
$$



## SOLUTION

Using the force triangle and the laws of cosines and sines:
We have

$$
\begin{aligned}
\gamma & =180^{\circ}-\left(20^{\circ}+10^{\circ}\right) \\
& =150^{\circ}
\end{aligned}
$$

Then

$$
R^{2}=(48 \mathrm{~N})^{2}+(60 \mathrm{~N})^{2}
$$

$$
-2(48 \mathrm{~N})(60 \mathrm{~N}) \cos 150^{\circ}
$$

$$
R=104.366 \mathrm{~N}
$$

and

$$
\frac{48 \mathrm{~N}}{\sin \alpha}=\frac{104.366 \mathrm{~N}}{\sin 150^{\circ}}
$$

$$
\sin \alpha=0.22996
$$

$$
\alpha=13.2947^{\circ}
$$

Hence:

$$
\begin{aligned}
\phi & =180^{\circ}-\alpha-80^{\circ} \\
& =180^{\circ}-13.2947^{\circ}-80^{\circ} \\
& =86.705^{\circ}
\end{aligned}
$$

$$
\mathbf{R}=104.4 \mathrm{~N} \searrow 86.7^{\circ}
$$



## SOLUTION

Using the force triangle and the laws of cosines and sines:
We have

$$
\begin{aligned}
\gamma & =180^{\circ}-\left(20^{\circ}+10^{\circ}\right) \\
& =150^{\circ}
\end{aligned}
$$

Then

$$
R^{2}=(60 \mathrm{~N})^{2}+(48 \mathrm{~N})^{2}
$$

$$
-2(60 \mathrm{~N})(48 \mathrm{~N}) \cos 150^{\circ}
$$

$$
R=104.366 \mathrm{~N}
$$

and

$$
\frac{60 \mathrm{~N}}{\sin \alpha}=\frac{104.366 \mathrm{~N}}{\sin 150^{\circ}}
$$

$$
\sin \alpha=0.28745
$$

$$
\alpha=16.7054^{\circ}
$$

Hence:

$$
\begin{aligned}
\phi & =180^{\circ}-\alpha-180^{\circ} \\
& =180^{\circ}-16.7054^{\circ}-80^{\circ} \\
& =83.295^{\circ}
\end{aligned}
$$



$$
\mathbf{R}=104.4 \mathrm{~N} \triangle 83.3^{\circ}
$$



## PROBLEM 2.21

Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

Compute the following distances:

29-lb Force:

$$
\begin{aligned}
O A & =\sqrt{(84)^{2}+(80)^{2}} \\
& =116 \mathrm{in} . \\
O B & =\sqrt{(28)^{2}+(96)^{2}} \\
& =100 \mathrm{in} . \\
O C & =\sqrt{(48)^{2}+(90)^{2}} \\
& =102 \mathrm{in} .
\end{aligned}
$$

$$
\begin{aligned}
& F_{x}=+(29 \mathrm{lb}) \frac{84}{116} \\
& F_{y}=+(29 \mathrm{lb}) \frac{80}{116}
\end{aligned}
$$


$F_{x}=+21.0 \mathrm{lb}$
$F_{y}=+20.0 \mathrm{lb}$
$F_{x}=-14.00 \mathrm{lb}$
$F_{y}=+48.0 \mathrm{lb}$
$F_{y}=+(50 \mathrm{lb}) \frac{96}{100}$
$F_{x}=+(51 \mathrm{lb}) \frac{48}{102}$
$F_{y}=-(51 \mathrm{lb}) \frac{90}{102}$
$F_{x}=+24.0 \mathrm{lb}$
$F_{y}=-45.0 \mathrm{lb}$


## SOLUTION

Compute the following distances:

$$
\begin{aligned}
O A & =\sqrt{(600)^{2}+(800)^{2}} \\
& =1000 \mathrm{~mm} \\
O B & =\sqrt{(560)^{2}+(900)^{2}} \\
& =1060 \mathrm{~mm} \\
O C & =\sqrt{(480)^{2}+(900)^{2}} \\
& =1020 \mathrm{~mm}
\end{aligned}
$$

800-N Force:

$$
\begin{aligned}
& F_{x}=+(800 \mathrm{~N}) \frac{800}{1000} \\
& F_{y}=+(800 \mathrm{~N}) \frac{600}{1000}
\end{aligned}
$$



$$
F_{x}=+640 \mathrm{~N}
$$

$$
F_{y}=+480 \mathrm{~N}
$$

424-N Force:

$$
F_{x}=-(424 \mathrm{~N}) \frac{560}{1060}
$$

$$
F_{x}=-224 \mathrm{~N}
$$

$$
F_{y}=-(424 \mathrm{~N}) \frac{900}{1060}
$$

$$
F_{y}=-360 \mathrm{~N}
$$

408-N Force:

$$
\begin{aligned}
& F_{x}=+(408 \mathrm{~N}) \frac{480}{1020} \\
& F_{y}=-(408 \mathrm{~N}) \frac{900}{1020}
\end{aligned}
$$

$$
F_{x}=+192.0 \mathrm{~N}
$$

$$
F_{y}=-360 \mathrm{~N}
$$



## SOLUTION

80-N Force
$F_{x}=+(80 \mathrm{~N}) \cos 40^{\circ}$
$F_{x}=61.3 \mathrm{~N}$
$F_{y}=+(80 \mathrm{~N}) \sin 40^{\circ}$
$F_{y}=51.4 \mathrm{~N}$
120-N Force:
$F_{x}=+(120 \mathrm{~N}) \cos 70^{\circ}$
$F_{x}=41.0 \mathrm{~N}$
$F_{y}=+(120 \mathrm{~N}) \sin 70^{\circ}$
$F_{y}=112.8 \mathrm{~N}$
150-N Force:

$$
\begin{aligned}
& F_{x}=-(150 \mathrm{~N}) \cos 35^{\circ} \\
& F_{y}=+(150 \mathrm{~N}) \sin 35^{\circ}
\end{aligned}
$$

$$
\begin{array}{r}
F_{x}=-122.9 \mathrm{~N} \\
F_{y}=86.0 \mathrm{~N}
\end{array}
$$



## SOLUTION

| 40-lb Force: | $F_{x}=+(40 \mathrm{lb}) \cos 60^{\circ}$ | $F_{x}=20.0 \mathrm{lb}$ |
| :--- | :--- | :---: |
|  | $F_{y}=-(40 \mathrm{lb}) \sin 60^{\circ}$ | $F_{y}=-34.6 \mathrm{lb}$ |
| $50-\mathrm{lb}$ Force: | $F_{x}=-(50 \mathrm{lb}) \sin 50^{\circ}$ | $F_{x}=-38.3 \mathrm{lb}$ |
| 60-lb Force: | $F_{y}=-(50 \mathrm{lb}) \cos 50^{\circ}$ | $F_{y}=-32.1 \mathrm{lb}$ |
|  | $F_{x}=+(60 \mathrm{lb}) \cos 25^{\circ}$ | $F_{x}=54.4 \mathrm{lb}$ |
|  | $F_{y}=+(60 \mathrm{lb}) \sin 25^{\circ}$ | $F_{y}=25.4 \mathrm{lb}$ |



## PROBLEM 2.25

Member $B C$ exerts on member $A C$ a force $\mathbf{P}$ directed along line $B C$. Knowing that $\mathbf{P}$ must have a $325-\mathrm{N}$ horizontal component, determine ( $a$ ) the magnitude of the force $\mathbf{P},(b)$ its vertical component.

## SOLUTION

$$
\begin{aligned}
B C & =\sqrt{(650 \mathrm{~mm})^{2}+(720 \mathrm{~mm})^{2}} \\
& =970 \mathrm{~mm}
\end{aligned}
$$

(a)

$$
P_{x}=P\left(\frac{650}{970}\right)
$$

or

$$
\begin{aligned}
P & =P_{x}\left(\frac{970}{650}\right) \\
& =325 \mathrm{~N}\left(\frac{970}{650}\right) \\
& =485 \mathrm{~N}
\end{aligned}
$$



$$
P=485 \mathrm{~N}
$$

(b)

$$
\begin{aligned}
P_{y} & =P\left(\frac{720}{970}\right) \\
& =485 \mathrm{~N}\left(\frac{720}{970}\right) \\
& =360 \mathrm{~N}
\end{aligned}
$$



## SOLUTION

(a)
$P \sin 35^{\circ}=300 \mathrm{lb}$

$$
P=\frac{300 \mathrm{lb}}{\sin 35^{\circ}}
$$

$$
P=523 \mathrm{lb}
$$

(b) Vertical component

$$
\begin{array}{rlr}
P_{v} & =P \cos 35^{\circ} \\
& =(523 \mathrm{lb}) \cos 35^{\circ} & P_{v}=428 \mathrm{lb}
\end{array}
$$

## PROBLEM 2.27

The hydraulic cylinder $B C$ exerts on member $A B$ a force $\mathbf{P}$ directed along line $B C$. Knowing that $\mathbf{P}$ must have a $600-\mathrm{N}$ component perpendicular to member $A B$, determine $(a)$ the magnitude of the force $\mathbf{P},(b)$ its component along line $A B$.

## SOLUTION

(a)

$$
\begin{aligned}
180^{\circ} & =45^{\circ}+\alpha+90^{\circ}+30^{\circ} \\
\alpha & =180^{\circ}-45^{\circ}-90^{\circ}-30^{\circ} \\
& =15^{\circ} \\
\cos \alpha & =\frac{P_{x}}{P} \\
P & =\frac{P_{x}}{\cos \alpha} \\
& =\frac{600 \mathrm{~N}}{\cos 15^{\circ}} \\
& =621.17 \mathrm{~N} \quad A \quad P=621 \mathrm{~N}
\end{aligned}
$$

(b)

$$
\begin{aligned}
\tan \alpha & =\frac{P_{y}}{P_{x}} \\
P_{y} & =P_{x} \tan \alpha \\
& =(600 \mathrm{~N}) \tan 15^{\circ} \\
& =160.770 \mathrm{~N}
\end{aligned}
$$

$$
P_{y}=160.8 \mathrm{~N}
$$



## SOLUTION

(a)
(b)

(a)

$$
\begin{array}{rlr}
P & =\frac{P_{y}}{\cos 55^{\circ}} & \\
& =\frac{350 \mathrm{lb}}{\cos 55^{\circ}} & \\
& =610.21 \mathrm{lb} & P=610 \mathrm{lb} \\
P_{x} & =P \sin 55^{\circ} & \\
& =(610.21 \mathrm{lb}) \sin 55^{\circ} & \\
& =499.85 \mathrm{lb} & P_{x}=500 \mathrm{lb}
\end{array}
$$



## PROBLEM 2.29

The hydraulic cylinder $B D$ exerts on member $A B C$ a force $\mathbf{P}$ directed along line $B D$. Knowing that $\mathbf{P}$ must have a $750-\mathrm{N}$ component perpendicular to member $A B C$, determine ( $a$ ) the magnitude of the force $\mathbf{P},(b)$ its component parallel to $A B C$.

## SOLUTION


(a)
(b)

$$
\begin{array}{rlr}
750 \mathrm{~N} & =P \sin 20^{\circ} & \\
P & =2192.9 \mathrm{~N} & P=2190 \mathrm{~N} \\
P_{A B C} & =P \cos 20^{\circ} & \\
& =(2192.9 \mathrm{~N}) \cos 20^{\circ} & P_{A B C}=2060 \mathrm{~N}
\end{array}
$$



## PROBLEM 2.30

The guy wire $B D$ exerts on the telephone pole $A C$ a force $\mathbf{P}$ directed along $B D$. Knowing that $\mathbf{P}$ must have a $720-\mathrm{N}$ component perpendicular to the pole $A C$, determine $(a)$ the magnitude of the force $\mathbf{P},(b)$ its component along line $A C$.

## SOLUTION

(a)

$$
\begin{aligned}
P & =\frac{37}{12} P_{x} \\
& =\frac{37}{12}(720 \mathrm{~N}) \\
& =2220 \mathrm{~N}
\end{aligned}
$$



$$
P=2.22 \mathrm{kN}
$$

(b)

$$
\begin{aligned}
P_{y} & =\frac{35}{12} P_{x} \\
& =\frac{35}{12}(720 \mathrm{~N}) \\
& =2100 \mathrm{~N}
\end{aligned}
$$

$$
P_{y}=2.10 \mathrm{kN}
$$



## SOLUTION

Components of the forces were determined in Problem 2.21:

| Force | $x$ Comp. (lb) | $y$ Comp. (lb) |
| :---: | :---: | :---: |
| 29 lb | +21.0 | +20.0 |
| 50 lb | -14.00 | +48.0 |
| 51 lb | +24.0 | -45.0 |
|  | $R_{x}=+31.0$ | $R_{y}=+23.0$ |

$$
\begin{array}{rlrl}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(31.0 \mathrm{lb}) \mathbf{i}+(23.0 \mathrm{lb}) \mathbf{j} & R_{y}= & 23.0 \overrightarrow{\mathbf{j}} \\
\tan \alpha & =\frac{R_{y}}{R_{x}} & \\
& =\frac{23.0}{31.0} \\
\alpha & =36.573^{\circ} \\
R & =\frac{23.0 \mathrm{lb}}{\sin \left(36.573^{\circ}\right)} \\
& =38.601 \mathrm{lb} & & \\
R_{x}=31.0 \vec{i} \\
\hline
\end{array}
$$



## SOLUTION

Components of the forces were determined in Problem 2.23:

| Force | $x$ Comp. (N) | $y$ Comp. (N) |
| :---: | :---: | :---: |
| 80 N | +61.3 | +51.4 |
| 120 N | +41.0 | +112.8 |
| 150 N | -122.9 | +86.0 |

$$
\begin{array}{rlrl}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} & \\
& =(-20.6 \mathrm{~N}) \mathbf{i}+(250.2 \mathrm{~N}) \mathbf{j} & R \\
\tan \alpha & =\frac{R_{y}}{R_{x}} & & \underline{R}_{y}=250.2 \underline{j} \\
\tan \alpha & =\frac{250.2 \mathrm{~N}}{20.6 \mathrm{~N}} & & R_{x}=-20.6 \underline{c}^{\prime} \\
\tan \alpha & =12.1456 \\
\alpha & =85.293^{\circ} & & \mathbf{R}=251 \mathrm{~N} \quad 85.3^{\circ} .
\end{array}
$$



## PROBLEM 2.33

Determine the resultant of the three forces of Problem 2.24.
PROBLEM 2.24 Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

| Force | $x$ Comp. (lb) | $y$ Comp. (b) |
| :---: | :---: | :---: |
| 40 lb | +20.00 | -34.64 |
| 50 lb | -38.30 | -32.14 |
| 60 lb | +54.38 | +25.36 |
|  | $R_{x}=+36.08$ | $R_{y}=-41.42$ |

$$
\begin{aligned}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(+36.08 \mathrm{lb}) \mathbf{i}+(-41.42 \mathrm{lb}) \mathbf{j} \\
\tan \alpha & =\frac{R_{y}}{R_{x}} \\
\tan \alpha & =\frac{41.42 \mathrm{lb}}{36.08 \mathrm{lb}} \\
\tan \alpha & =1.14800 \\
\alpha & =48.942^{\circ} \\
R & =\frac{41.42 \mathrm{lb}}{\sin 48.942^{\circ}}
\end{aligned}
$$

$$
\mathbf{R}=54.9 \mathrm{lb}\left\ulcorner 48.9^{\circ}\right.
$$



## SOLUTION

Components of the forces were determined in Problem 2.22:

| Force | $x$ Comp. (N) | $y$ Comp. (N) |
| :---: | :---: | :---: |
| 800 lb | +640 | +480 |
| 424 lb | -224 | -360 |
| 408 lb | +192 | -360 |

$$
\begin{array}{rlrl}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(608 \mathrm{lb}) \mathbf{i}+(-240 \mathrm{lb}) \mathbf{j} & \\
\tan \alpha & =\frac{R_{y}}{R_{x}} & & \\
& =\frac{240}{608} & R_{x}=608 \underline{i} \\
\alpha & =21.541^{\circ} \\
R & =\frac{240 \mathrm{~N}}{\sin \left(21.541^{\circ}\right)} & \underline{R_{y}}=-240 j \\
& =653.65 \mathrm{~N} & & \\
\mathbf{R}=654 \mathrm{~N} \div 21.5^{\circ}
\end{array}
$$



## SOLUTION

100-N Force:
$F_{x}=+(100 \mathrm{~N}) \cos 35^{\circ}=+81.915 \mathrm{~N}$
$F_{y}=-(100 \mathrm{~N}) \sin 35^{\circ}=-57.358 \mathrm{~N}$
150-N Force:
$F_{x}=+(150 \mathrm{~N}) \cos 65^{\circ}=+63.393 \mathrm{~N}$
$F_{y}=-(150 \mathrm{~N}) \sin 65^{\circ}=-135.946 \mathrm{~N}$

200-N Force:
$F_{x}=-(200 \mathrm{~N}) \cos 35^{\circ}=-163.830 \mathrm{~N}$
$F_{y}=-(200 \mathrm{~N}) \sin 35^{\circ}=-114.715 \mathrm{~N}$

| Force | $x$ Comp. (N) | $y$ Comp. (N) |
| :---: | ---: | ---: |
| 100 N | +81.915 | -57.358 |
| 150 N | +63.393 | -135.946 |
| 200 N | -163.830 | -114.715 |
|  | $R_{x}=-18.522$ | $R_{y}=-308.02$ |



$$
\begin{array}{rlr}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(-18.522 \mathrm{~N}) \mathbf{i}+(-308.02 \mathrm{~N}) \mathbf{j} \\
\tan \alpha & =\frac{R_{y}}{R_{x}} \\
& =\frac{308.02}{18.522} \\
\alpha & =86.559^{\circ} \\
R & =\frac{308.02 \mathrm{~N}}{\sin 86.559} \quad \mathbf{R}=309 \mathrm{~N} \square 86.6^{\circ}
\end{array}
$$



## PROBLEM 2.36

Knowing that the tension in rope $A C$ is 365 N , determine the resultant of the three forces exerted at point $C$ of post $B C$.

## SOLUTION

Determine force components:
Cable force $A C: \quad F_{x}=-(365 \mathrm{~N}) \frac{960}{1460}=-240 \mathrm{~N}$

$$
F_{y}=-(365 \mathrm{~N}) \frac{1100}{1460}=-275 \mathrm{~N}
$$

500-N Force: $\quad F_{x}=(500 \mathrm{~N}) \frac{24}{25}=480 \mathrm{~N}$

$$
F_{y}=(500 \mathrm{~N}) \frac{7}{25}=140 \mathrm{~N}
$$



200-N Force: $\quad F_{x}=(200 \mathrm{~N}) \frac{4}{5}=160 \mathrm{~N}$

$$
F_{y}=-(200 \mathrm{~N}) \frac{3}{5}=-120 \mathrm{~N}
$$

and

$$
\begin{aligned}
R_{x} & =\Sigma F_{x}=-240 \mathrm{~N}+480 \mathrm{~N}+160 \mathrm{~N}=400 \mathrm{~N} \\
R_{y} & =\Sigma F_{y}=-275 \mathrm{~N}+140 \mathrm{~N}-120 \mathrm{~N}=-255 \mathrm{~N} \\
R & =\sqrt{R_{x}^{2}+R_{y}^{2}} \\
& =\sqrt{(400 \mathrm{~N})^{2}+(-255 \mathrm{~N})^{2}} \\
& =474.37 \mathrm{~N}
\end{aligned}
$$



Further: $\quad \tan \alpha=\frac{255}{400}$

$$
\alpha=32.5^{\circ}
$$

$$
\mathbf{R}=474 \mathrm{~N} \subset 32.5^{\circ}
$$



## SOLUTION

60-lb Force:
$F_{x}=(60 \mathrm{lb}) \cos 20^{\circ}=56.382 \mathrm{lb}$
$F_{y}=(60 \mathrm{lb}) \sin 20^{\circ}=20.521 \mathrm{lb}$

80-lb Force: $\quad F_{x}=(80 \mathrm{lb}) \cos 60^{\circ}=40.000 \mathrm{lb}$ $F_{y}=(80 \mathrm{lb}) \sin 60^{\circ}=69.282 \mathrm{lb}$

120-lb Force:

$$
\begin{aligned}
& F_{x}=(120 \mathrm{lb}) \cos 30^{\circ}=103.923 \mathrm{lb} \\
& F_{y}=-(120 \mathrm{lb}) \sin 30^{\circ}=-60.000 \mathrm{lb}
\end{aligned}
$$


and

$$
\begin{aligned}
R_{x} & =\Sigma F_{x}=200.305 \mathrm{lb} \\
R_{y} & =\Sigma F_{y}=29.803 \mathrm{lb} \\
R & =\sqrt{(200.305 \mathrm{lb})^{2}+(29.803 \mathrm{lb})^{2}} \\
& =202.510 \mathrm{lb}
\end{aligned}
$$

Further: $\quad \tan \alpha=\frac{29.803}{200.305}$

$$
\begin{array}{rlr}
\alpha & =\tan ^{-1} \frac{29.803}{200.305} \\
& =8.46^{\circ} \quad \mathbf{R}=203 \mathrm{lb}<8.46^{\circ}
\end{array}
$$



## SOLUTION




## PROBLEM 2.39

For the collar of Problem 2.35, determine $(a)$ the required value of $\alpha$ if the resultant of the three forces shown is to be vertical, (b) the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
R_{x} & =\Sigma F_{x} \\
& =(100 \mathrm{~N}) \cos \alpha+(150 \mathrm{~N}) \cos \left(\alpha+30^{\circ}\right)-(200 \mathrm{~N}) \cos \alpha \\
R_{x} & =-(100 \mathrm{~N}) \cos \alpha+(150 \mathrm{~N}) \cos \left(\alpha+30^{\circ}\right)  \tag{1}\\
R_{y} & =\Sigma F_{y} \\
& =-(100 \mathrm{~N}) \sin \alpha-(150 \mathrm{~N}) \sin \left(\alpha+30^{\circ}\right)-(200 \mathrm{~N}) \sin \alpha \\
R_{y} & =-(300 \mathrm{~N}) \sin \alpha-(150 \mathrm{~N}) \sin \left(\alpha+30^{\circ}\right) \tag{2}
\end{align*}
$$

(a) For $\mathbf{R}$ to be vertical, we must have $R_{x}=0$. We make $R_{x}=0$ in Eq. (1):

$$
\begin{aligned}
-100 \cos \alpha+150 \cos \left(\alpha+30^{\circ}\right) & =0 \\
-100 \cos \alpha+150\left(\cos \alpha \cos 30^{\circ}-\sin \alpha \sin 30^{\circ}\right) & =0 \\
29.904 \cos \alpha & =75 \sin \alpha \\
\tan \alpha & =\frac{29.904}{75} \\
& =0.39872 \\
\alpha & =21.738^{\circ}
\end{aligned}
$$

(b) Substituting for $\alpha$ in Eq. (2):

$$
\begin{aligned}
R_{y} & =-300 \sin 21.738^{\circ}-150 \sin 51.738^{\circ} \\
& =-228.89 \mathrm{~N}
\end{aligned}
$$

$$
R=\left|R_{y}\right|=228.89 \mathrm{~N} \quad R=229 \mathrm{~N}
$$



## PROBLEM 2.40

For the post of Prob. 2.36, determine $(a)$ the required tension in rope $A C$ if the resultant of the three forces exerted at point $C$ is to be horizontal, $(b)$ the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
& R_{x}=\Sigma F_{x}=-\frac{960}{1460} T_{A C}+\frac{24}{25}(500 \mathrm{~N})+\frac{4}{5}(200 \mathrm{~N}) \\
& R_{x}=-\frac{48}{73} T_{A C}+640 \mathrm{~N}  \tag{1}\\
& R_{y}=\Sigma F_{y}=-\frac{1100}{1460} T_{A C}+\frac{7}{25}(500 \mathrm{~N})-\frac{3}{5}(200 \mathrm{~N}) \\
& R_{y}=-\frac{55}{73} T_{A C}+20 \mathrm{~N} \tag{2}
\end{align*}
$$

(a) For $\mathbf{R}$ to be horizontal, we must have $R_{y}=0$.

Set $R_{y}=0$ in Eq. (2):

$$
\begin{array}{rlr}
-\frac{55}{73} T_{A C}+20 \mathrm{~N} & =0 \\
T_{A C} & =26.545 \mathrm{~N} & T_{A C}=26.5 \mathrm{~N}
\end{array}
$$

(b) Substituting for $T_{A C}$ into Eq. (1) gives

$$
\begin{aligned}
& R_{x}=-\frac{48}{73}(26.545 \mathrm{~N})+640 \mathrm{~N} \\
& R_{x}=622.55 \mathrm{~N} \\
& R=R_{x}=623 \mathrm{~N} \\
& R=623 \mathrm{~N}
\end{aligned}
$$



## SOLUTION



Using the $x$ and $y$ axes shown:

$$
\begin{align*}
R_{x}=\Sigma F_{x} & =T_{A C} \sin 10^{\circ}+(50 \mathrm{lb}) \cos 35^{\circ}+(75 \mathrm{lb}) \cos 60^{\circ} \\
& =T_{A C} \sin 10^{\circ}+78.458 \mathrm{lb}  \tag{1}\\
R_{y}=\Sigma F_{y} & =(50 \mathrm{lb}) \sin 35^{\circ}+(75 \mathrm{lb}) \sin 60^{\circ}-T_{A C} \cos 10^{\circ} \\
R_{y}= & 93.631 \mathrm{lb}-T_{A C} \cos 10^{\circ} \tag{2}
\end{align*}
$$

(a) $\operatorname{Set} R_{y}=0$ in Eq. (2):

$$
\begin{aligned}
93.631 \mathrm{lb}-T_{A C} \cos 10^{\circ} & =0 \\
T_{A C} & =95.075 \mathrm{lb} \quad T_{A C}=95.1 \mathrm{lb}
\end{aligned}
$$

(b) Substituting for $T_{A C}$ in Eq. (1):

$$
\begin{array}{rlr}
R_{x} & =(95.075 \mathrm{lb}) \sin 10^{\circ}+78.458 \mathrm{lb} & \\
& =94.968 \mathrm{lb} \\
R & =R_{x} & R=95.0 \mathrm{lb}
\end{array}
$$



## PROBLEM 2.42

For the block of Problems 2.37 and 2.38, determine $(a)$ the required value of $\alpha$ if the resultant of the three forces shown is to be parallel to the incline, $(b)$ the corresponding magnitude of the resultant.

## SOLUTION



Select the $x$ axis to be along $a a^{\prime}$.
Then

$$
\begin{equation*}
R_{x}=\Sigma F_{x}=(60 \mathrm{lb})+(80 \mathrm{lb}) \cos \alpha+(120 \mathrm{lb}) \sin \alpha \tag{1}
\end{equation*}
$$

and

$$
\begin{equation*}
R_{y}=\Sigma F_{y}=(80 \mathrm{lb}) \sin \alpha-(120 \mathrm{lb}) \cos \alpha \tag{2}
\end{equation*}
$$

(a) Set $R_{y}=0$ in Eq. (2).

$$
(80 \mathrm{lb}) \sin \alpha-(120 \mathrm{lb}) \cos \alpha=0
$$

Dividing each term by $\cos \alpha$ gives:

$$
\begin{aligned}
(80 \mathrm{lb}) \tan \alpha & =120 \mathrm{lb} \\
\tan \alpha & =\frac{120 \mathrm{lb}}{80 \mathrm{lb}} \\
\alpha & =56.310^{\circ} \quad \alpha=56.3^{\circ}
\end{aligned}
$$

(b) Substituting for $\alpha$ in Eq. (1) gives:

$$
R_{x}=60 \mathrm{lb}+(80 \mathrm{lb}) \cos 56.31^{\circ}+(120 \mathrm{lb}) \sin 56.31^{\circ}=204.22 \mathrm{lb} \quad R_{x}=204 \mathrm{lb}
$$



## PROBLEM 2.43

Two cables are tied together at $C$ and are loaded as shown. Determine the tension $(a)$ in cable $A C,(b)$ in cable $B C$.

## SOLUTION

## Free-Body Diagram



## Force Triangle



Law of sines:
$\frac{T_{A C}}{\sin 60^{\circ}}=\frac{T_{B C}}{\sin 40^{\circ}}=\frac{400 \mathrm{lb}}{\sin 80^{\circ}}$
(a)
(b)
$T_{A C}=\frac{400 \mathrm{lb}}{\sin 80^{\circ}}\left(\sin 60^{\circ}\right)$
$T_{A C}=352 \mathrm{lb}$
$T_{B C}=\frac{400 \mathrm{lb}}{\sin 80^{\circ}}\left(\sin 40^{\circ}\right)$
$T_{B C}=261 \mathrm{lb}$


## SOLUTION

## Free-Body Diagram

Force Triangle


Law of sines:

$$
\frac{T_{A C}}{\sin 60^{\circ}}=\frac{T_{B C}}{\sin 35^{\circ}}=\frac{6 \mathrm{kN}}{\sin 85^{\circ}}
$$

(a)

$$
T_{A C}=\frac{6 \mathrm{kN}}{\sin 85^{\circ}}\left(\sin 60^{\circ}\right)
$$

$$
T_{A C}=5.22 \mathrm{kN}
$$

(b)

$$
T_{B C}=\frac{6 \mathrm{kN}}{\sin 85^{\circ}}\left(\sin 35^{\circ}\right)
$$

$$
T_{B C}=3.45 \mathrm{kN}
$$



## PROBLEM 2.45

Two cables are tied together at $C$ and loaded as shown. Determine the tension $(a)$ in cable $A C,(b)$ in cable $B C$.

## SOLUTION

$$
\begin{aligned}
\tan \alpha & =\frac{1.4}{4.8} \\
\alpha & =16.2602^{\circ} \\
\tan \beta & =\frac{1.6}{3} \\
\beta & =28.073^{\circ}
\end{aligned}
$$



## Force Triangle


(a)

$$
T_{A C}=\frac{1.98 \mathrm{kN}}{\sin 44.333^{\circ}} \sin 61.927^{\circ} \quad T_{A C}=2.50 \mathrm{kN}
$$

(b)

$$
T_{B C}=\frac{1.98 \mathrm{kN}}{\sin 44.333^{\circ}} \sin 73.740^{\circ}
$$

$$
T_{B C}=2.72 \mathrm{kN}
$$



## PROBLEM 2.46

Two cables are tied together at $C$ and are loaded as shown. Knowing that $\mathbf{P}=500 \mathrm{~N}$ and $\alpha=60^{\circ}$, determine the tension in $(a)$ in cable $A C,(b)$ in cable $B C$.

## SOLUTION

## Free-Body Diagram



Law of sines:

$$
\frac{T_{A C}}{\sin 35^{\circ}}=\frac{T_{B C}}{\sin 75^{\circ}}=\frac{500 \mathrm{~N}}{\sin 70^{\circ}}
$$

(a)
(b)

$$
T_{A C}=\frac{500 \mathrm{~N}}{\sin 70^{\circ}} \sin 35^{\circ}
$$

$$
T_{A C}=305 \mathrm{~N}
$$

$$
T_{B C}=\frac{500 \mathrm{~N}}{\sin 70^{\circ}} \sin 75^{\circ}
$$

$$
T_{B C}=514 \mathrm{~N}
$$



## SOLUTION

Free-Body Diagram


$$
\begin{aligned}
W & =\mathrm{mg} \\
& =(200 \mathrm{~kg})\left(9.81 \mathrm{~m} / \mathrm{s}^{2}\right) \\
& =1962 \mathrm{~N}
\end{aligned}
$$

Law of sines:

$$
\frac{T_{A C}}{\sin 15^{\circ}}=\frac{T_{B C}}{\sin 105^{\circ}}=\frac{1962 \mathrm{~N}}{\sin 60^{\circ}}
$$

(a)
$T_{A C}=\frac{(1962 \mathrm{~N}) \sin 15^{\circ}}{\sin 60^{\circ}}$
$T_{B C}=\frac{(1962 \mathrm{~N}) \sin 105^{\circ}}{\sin 60^{\circ}}$

## Force Triangle


$T_{A C}=586 \mathrm{~N}$
$T_{B C}=2190 \mathrm{~N}$


## SOLUTION

Free-Body Diagram


## Force Triangle



Law of sines:

$$
\frac{T_{A C}}{\sin 110^{\circ}}=\frac{T_{B C}}{\sin 5^{\circ}}=\frac{1200 \mathrm{lb}}{\sin 65^{\circ}}
$$

(a)
(b)

$$
\begin{array}{ll}
T_{A C}=\frac{1200 \mathrm{lb}}{\sin 65^{\circ}} \sin 110^{\circ} & T_{A C}=1244 \mathrm{lb} \\
T_{B C}=\frac{1200 \mathrm{lb}}{\sin 65^{\circ}} \sin 5^{\circ} & T_{B C}=115.4 \mathrm{lb}
\end{array}
$$



## SOLUTION

## Free-Body Diagram



$$
\xrightarrow{+} \Sigma F_{x}=0 \quad-T_{C A} \sin 30^{\circ}+T_{C B} \sin 30^{\circ}-P \cos 45^{\circ}-200 \mathrm{~N}=0
$$

For $P=200 \mathrm{~N}$ we have,

$$
\begin{array}{r}
-0.5 T_{C A}+0.5 T_{C B}+212.13-200=0 \\
+\uparrow \Sigma F_{y}=0 \quad T_{C A} \cos 30^{\circ}-T_{C B} \cos 30^{\circ}-P \sin 45^{\circ}=0 \\
0.86603 T_{C A}+0.86603 T_{C B}-212.13=0
\end{array}
$$

Solving equations (1) and (2) simultaneously gives,

$$
T_{C A}=134.6 \mathrm{~N}
$$

$$
T_{C B}=110.4 \mathrm{~N}
$$



## PROBLEM 2.50

Two cables are tied together at $C$ and are loaded as shown. Determine the range of values of $\mathbf{P}$ for which both cables remain taut.

## SOLUTION

## Free-Body Diagram


$\xrightarrow{+} \Sigma F_{x}=0 \quad-T_{C A} \sin 30^{\circ}+T_{C B} \sin 30^{\circ}-P \cos 45^{\circ}-200 \mathrm{~N}=0$
For $T_{C A}=0$ we have,

$$
\begin{equation*}
0.5 T_{C B}+0.70711 P-200=0 \tag{1}
\end{equation*}
$$

$+\dagger \Sigma F_{y}=0 \quad T_{C A} \cos 30^{\circ}-T_{C B} \cos 30^{\circ}-P \sin 45^{\circ}=0$; again setting $T_{C A}=0$ yields,
$0.86603 T_{C B}-0.70711 P=0$
Adding equations (1) and (2) gives, $1.36603 T_{C B}=200$ hence $T_{C B}=146.410 \mathrm{~N}$ and $P=179.315 \mathrm{~N}$ Substituting for $T_{C B}=0$ into the equilibrium equations and solving simultaneously gives,

$$
\begin{aligned}
& -0.5 T_{C A}+0.70711 P-200=0 \\
& 0.86603 T_{C A}-0.70711 P=0
\end{aligned}
$$

And $T_{C A}=546.40 \mathrm{~N}, P=669.20 \mathrm{~N}$ Thus for both cables to remain taut, load $P$ must be within the range of 179.315 N and 669.20 N .

$$
179.3 \mathrm{~N}<P<669 \mathrm{~N}
$$



## SOLUTION

## Free-Body Diagram

Resolving the forces into $x$ - and $y$-directions:

$$
\mathbf{R}=\mathbf{P}+\mathbf{Q}+\mathbf{F}_{A}+\mathbf{F}_{B}=0
$$

Substituting components:

$$
\begin{aligned}
\mathbf{R}= & -(500 \mathrm{lb}) \mathbf{j}+\left[(650 \mathrm{lb}) \cos 50^{\circ}\right] \mathbf{i} \\
& -\left[(650 \mathrm{lb}) \sin 50^{\circ}\right] \mathbf{j} \\
& +F_{B} \mathbf{i}-\left(F_{A} \cos 50^{\circ}\right) \mathbf{i}+\left(F_{A} \sin 50^{\circ}\right) \mathbf{j}=0 \quad \underline{F}_{B}
\end{aligned}
$$

In the $y$-direction (one unknown force):

$$
-500 \mathrm{lb}-(650 \mathrm{lb}) \sin 50^{\circ}+F_{A} \sin 50^{\circ}=0
$$



Thus,

$$
\begin{aligned}
F_{A} & =\frac{500 \mathrm{lb}+(650 \mathrm{lb}) \sin 50^{\circ}}{\sin 50^{\circ}} \\
& =1302.70 \mathrm{lb}
\end{aligned}
$$

$$
F_{A}=1303 \mathrm{lb}
$$

In the $x$-direction:
$(650 \mathrm{lb}) \cos 50^{\circ}+F_{B}-F_{A} \cos 50^{\circ}=0$
Thus,

$$
\begin{aligned}
F_{B} & =F_{A} \cos 50^{\circ}-(650 \mathrm{lb}) \cos 50^{\circ} \\
& =(1302.70 \mathrm{lb}) \cos 50^{\circ}-(650 \mathrm{lb}) \cos 50^{\circ} \\
& =419.55 \mathrm{lb}
\end{aligned} F_{B}=420 \mathrm{lb}
$$



## SOLUTION

## Free-Body Diagram

Resolving the forces into $x$ - and $y$-directions:

$$
\text { Substituting components: } \quad \begin{aligned}
\mathbf{R}= & \mathbf{P}+\mathbf{Q}+\mathbf{F}_{A}+\mathbf{F}_{B}=0 \\
\mathbf{R}= & -P \mathbf{j}+Q \cos 50^{\circ} \mathbf{i}-Q \sin 50^{\circ} \mathbf{j} \\
& -\left[(750 \mathrm{lb}) \cos 50^{\circ}\right] \mathbf{i} \\
& +\left[(750 \mathrm{lb}) \sin 50^{\circ}\right] \mathbf{j}+(400 \mathrm{lb}) \mathbf{i}
\end{aligned}
$$

In the $x$-direction (one unknown force):

$$
\begin{aligned}
& Q \cos 50^{\circ}-\left[(750 \mathrm{lb}) \cos 50^{\circ}\right]+400 \mathrm{lb}=0 \\
& Q
\end{aligned} \begin{aligned}
Q & =\frac{(750 \mathrm{lb}) \cos 50^{\circ}-400 \mathrm{lb}}{\cos 50^{\circ}} \\
& =127.710 \mathrm{lb}
\end{aligned}
$$

In the $y$-direction:

$$
-P-Q \sin 50^{\circ}+(750 \mathrm{lb}) \sin 50^{\circ}=0
$$

$$
\begin{aligned}
P & =-Q \sin 50^{\circ}+(750 \mathrm{lb}) \sin 50^{\circ} \\
& =-(127.710 \mathrm{lb}) \sin 50^{\circ}+(750 \mathrm{lb}) \sin 50^{\circ} \\
& =476.70 \mathrm{lb}
\end{aligned}
$$

$$
P=477 \mathrm{lb} ; \quad Q=127.7 \mathrm{lb}
$$



## PROBLEM 2.53

A welded connection is in equilibrium under the action of the four forces shown. Knowing that $F_{A}=8 \mathrm{kN}$ and $F_{B}=16 \mathrm{kN}$, determine the magnitudes of the other two forces.

## SOLUTION

## Free-Body Diagram of Connection



With

$$
\begin{array}{rlr}
F_{A} & =8 \mathrm{kN} \\
F_{B} & =16 \mathrm{kN} & \\
F_{C} & =\frac{4}{5}(16 \mathrm{kN})-\frac{4}{5}(8 \mathrm{kN}) & F_{C}=6.40 \mathrm{kN} \\
\Sigma F_{y} & =0:-F_{D}+\frac{3}{5} F_{B}-\frac{3}{5} F_{A}=0 &
\end{array}
$$

With $F_{A}$ and $F_{B}$ as above: $\quad F_{D}=\frac{3}{5}(16 \mathrm{kN})-\frac{3}{5}(8 \mathrm{kN}) \quad F_{D}=4.80 \mathrm{kN}$


## SOLUTION

or
With

$$
\begin{array}{rlrl}
F_{A} & =5 \mathrm{kN}, \quad F_{D}=8 \mathrm{kN} & \\
F_{B} & =\frac{5}{3}\left[6 \mathrm{kN}+\frac{3}{5}(5 \mathrm{kN})\right] & F_{B}=15.00 \mathrm{kN} \\
\Sigma F_{x} & =0:-F_{C}+\frac{4}{5} F_{B}-\frac{4}{5} F_{A}=0 & \\
F_{C} & =\frac{4}{5}\left(F_{B}-F_{A}\right) & & \\
& =\frac{4}{5}(15 \mathrm{kN}-5 \mathrm{kN}) & F_{C}=8.00 \mathrm{kN}
\end{array}
$$



## SOLUTION

## Free-Body Diagram



$$
\begin{align*}
&+ \\
& \xrightarrow{+} F_{x}=0: T_{A C B} \cos 10^{\circ}-T_{A C B} \cos 30^{\circ}-T_{C D} \cos 30^{\circ}=0  \tag{1}\\
& T_{C D}=0.137158 T_{A C B} \\
&+\uparrow \Sigma F_{y}=0: T_{A C B} \sin 10^{\circ}+T_{A C B} \sin 30^{\circ}+T_{C D} \sin 30^{\circ}-200=0  \tag{2}\\
& 0.67365 T_{A C B}+0.5 T_{C D}=200
\end{align*}
$$

(a) Substitute (1) into (2): $0.67365 T_{A C B}+0.5\left(0.137158 T_{A C B}\right)=200$

$$
\begin{array}{rlr}
T_{A C B} & =269.46 \mathrm{lb} & T_{A C B}=269 \mathrm{lb} \\
T_{C D} & =0.137158(269.46 \mathrm{lb}) & T_{C D}=37.0 \mathrm{lb}
\end{array}
$$

(b) From (1):


## SOLUTION

Free-Body Diagram

$\xrightarrow{+} \Sigma F_{x}=0: \quad T_{A C B} \cos 15^{\circ}-T_{A C B} \cos 25^{\circ}-(20 \mathrm{lb}) \cos 25^{\circ}=0$
$T_{A C B}=304.04 \mathrm{lb}$
$+\uparrow \Sigma F_{y}=0: \quad(304.04 \mathrm{lb}) \sin 15^{\circ}+(304.04 \mathrm{lb}) \sin 25^{\circ}$
$+(20 \mathrm{lb}) \sin 25^{\circ}-W=0$

$$
W=215.64 \mathrm{lb}
$$

(a) $\quad W=216 \mathrm{lb}$
(b) $T_{A C B}=304 \mathrm{lb}$


## SOLUTION

## Free-Body Diagram

## Force Triangle


(a) For a minimum tension in cable $B C$, set angle between cables to 90 degrees.

By inspection,

$$
\begin{aligned}
& T_{A C}=(6 \mathrm{kN}) \cos 35^{\circ} \\
& T_{B C}=(6 \mathrm{kN}) \sin 35^{\circ}
\end{aligned}
$$

$$
T_{A C}=4.91 \mathrm{kN}
$$

$$
T_{B C}=3.44 \mathrm{kN}
$$

(b) For equal tension in both cables, the force triangle will be an isosceles.

Therefore, by inspection,


$$
T_{A C}=T_{B C}=(1 / 2) \frac{6 \mathrm{kN}}{\cos 35^{\circ}}
$$

$$
T_{A C}=T_{B C}=3.66 \mathrm{kN}
$$



## PROBLEM 2.58

For the cables of Problem 2.46, it is known that the maximum allowable tension is 600 N in cable $A C$ and 750 N in cable $B C$. Determine (a) the maximum force $\mathbf{P}$ that can be applied at $C$, (b) the corresponding value of $\alpha$.

## SOLUTION

## Free-Body Diagram



(a) Law of cosines

$$
P^{2}=(600)^{2}+(750)^{2}-2(600)(750) \cos \left(25^{\circ}+45^{\circ}\right)
$$

$$
P=784.02 \mathrm{~N}
$$

$$
P=784 \mathrm{~N}
$$

(b) Law of sines

$$
\begin{aligned}
\frac{\sin \beta}{600 \mathrm{~N}} & =\frac{\sin \left(25^{\circ}+45^{\circ}\right)}{784.02 \mathrm{~N}} \\
\beta & =46.0^{\circ} \quad \therefore \quad \alpha=46.0^{\circ}+25^{\circ} \quad \alpha=71.0^{\circ}
\end{aligned}
$$



## SOLUTION



## Force Triangle



To be smallest, $T_{B C}$ must be perpendicular to the direction of $T_{A C}$.
(a) Thus,
$\alpha=5.00^{\circ}$
$\alpha=5.00^{\circ}$
$T_{B C}=104.6 \mathrm{lb}$


## SOLUTION

## Free-Body Diagram


$\Sigma F_{x}=0: \quad-T_{B C}-Q \cos 60^{\circ}+75 \mathrm{lb}=0$

$$
\begin{equation*}
T_{B C}=75 \mathrm{lb}-Q \cos 60^{\circ} \tag{1}
\end{equation*}
$$

$$
\Sigma F_{y}=0: \quad T_{A C}-Q \sin 60^{\circ}=0
$$

$$
\begin{equation*}
T_{A C}=Q \sin 60^{\circ} \tag{2}
\end{equation*}
$$

Requirement: $\quad T_{A C}=60 \mathrm{lb}:$
From Eq. (2): $\quad Q \sin 60^{\circ}=60 \mathrm{lb}$

$$
Q=69.3 \mathrm{lb}
$$

Requirement:

$$
T_{B C}=60 \mathrm{lb}:
$$

From Eq. (1): $\quad 75 \mathrm{lb}-Q \cos 60^{\circ}=60 \mathrm{lb}$

$$
Q=30.0 \mathrm{lb} 30.0 \mathrm{lb} \leq Q \leq 69.3 \mathrm{lb}
$$



## PROBLEM 2.61

A movable bin and its contents have a combined weight of 2.8 kN . Determine the shortest chain sling $A C B$ that can be used to lift the loaded bin if the tension in the chain is not to exceed 5 kN .

## SOLUTION

## Free-Body Diagram



Isosceles Force Triangle


Law of sines: $\quad \sin \alpha=\frac{\frac{1}{2}(2.8 \mathrm{kN})}{T_{A C}}$

$$
\begin{aligned}
T_{A C} & =5 \mathrm{kN} \\
\sin \alpha & =\frac{\frac{1}{2}(2.8 \mathrm{kN})}{5 \mathrm{kN}}
\end{aligned}
$$

$$
\alpha=16.2602^{\circ}
$$

From Eq. (1): $\tan 16.2602^{\circ}=\frac{h}{0.6 \mathrm{~m}} \quad \therefore \quad h=0.175000 \mathrm{~m}$
Half-length of chain $=A C=\sqrt{(0.6 \mathrm{~m})^{2}+(0.175 \mathrm{~m})^{2}}$

$$
=0.625 \mathrm{~m}
$$

Total length:

$$
=2 \times 0.625 \mathrm{~m}
$$

$$
1.250 \mathrm{~m}
$$



## PROBLEM 2.62

For $W=800 \mathrm{~N}, P=200 \mathrm{~N}$, and $d=600 \mathrm{~mm}$, determine the value of $h$ consistent with equilibrium.

## SOLUTION

## Free-Body Diagram


$T_{A C}=T_{B C}=800 \mathrm{~N}$

$$
A C=B C=\sqrt{\left(h^{2}+d^{2}\right)}
$$

$$
\Sigma F_{y}=0: \quad 2(800 \mathrm{~N}) \frac{h}{\sqrt{h^{2}+d^{2}}}-P=0
$$

$$
800=\frac{P}{2} \sqrt{1+\left(\frac{d}{h}\right)^{2}}
$$

Data: $\quad P=200 \mathrm{~N}, d=600 \mathrm{~mm}$ and solving for $h$

$$
800 \mathrm{~N}=\frac{200 \mathrm{~N}}{2} \sqrt{1+\left(\frac{600 \mathrm{~mm}}{h}\right)^{2}}
$$

$$
h=75.6 \mathrm{~mm}
$$



## SOLUTION

(a) Free Body: Collar $\boldsymbol{A}$


Force Triangle

$$
\frac{P}{4.5}=\frac{50 \mathrm{lb}}{20.5}
$$

$$
P=10.98 \mathrm{lb}
$$

## Force Triangle

$\frac{P}{15}=\frac{50 \mathrm{lb}}{25} \quad P=30.0 \mathrm{lb}$


## SOLUTION

Free Body: Collar $A$


## Force Triangle



$$
\begin{aligned}
N^{2} & =(50)^{2}-(48)^{2}=196 \\
N & =14.00 \mathrm{lb}
\end{aligned}
$$

## Similar Triangles

$$
\frac{x}{20 \mathrm{in} .}=\frac{48 \mathrm{lb}}{14 \mathrm{lb}}
$$



$$
x=68.6 \mathrm{in} .
$$

## PROBLEM 2.65

Three forces are applied to a bracket as shown. The directions of the two $150-\mathrm{N}$ forces may vary, but the angle between these forces is always $50^{\circ}$. Determine the range of values of $\alpha$ for which the magnitude of the resultant of the forces acting at $A$ is less than 600 N .

## SOLUTION

Combine the two $150-\mathrm{N}$ forces into a resultant force $Q$ :


Equivalent loading at $A$ :


Using the law of cosines:

$$
\begin{aligned}
(600 \mathrm{~N})^{2} & =(500 \mathrm{~N})^{2}+(271.89 \mathrm{~N})^{2}+2(500 \mathrm{~N})(271.89 \mathrm{~N}) \cos \left(55^{\circ}+\alpha\right) \\
\cos \left(55^{\circ}+\alpha\right) & =0.132685
\end{aligned}
$$

Two values for $\alpha$ : $\quad 55^{\circ}+\alpha=82.375$
$\alpha=27.4^{\circ}$

$$
\begin{aligned}
& 55^{\circ}+\alpha=-82.375^{\circ} \\
& 55^{\circ}+\alpha=360^{\circ}-82.375^{\circ}
\end{aligned}
$$

or

$$
\alpha=222.6^{\circ}
$$

For $R<600 \mathrm{lb}$ : $27.4^{\circ}<\alpha<222.6^{\circ}$


## SOLUTION

## Free-Body Diagram: Pulley $\boldsymbol{A}$



$$
\begin{aligned}
\xrightarrow{+} \Sigma F_{x} & =0:-2 P\left(\frac{5}{\sqrt{281}}\right)+P \cos \alpha=0 \\
\cos \alpha & =0.59655 \\
\alpha & = \pm 53.377^{\circ}
\end{aligned}
$$

For $\alpha=+53.377^{\circ}$ :

$$
+\uparrow \Sigma F_{y}=0: \quad 2 P\left(\frac{16}{\sqrt{281}}\right)+P \sin 53.377^{\circ}-1962 \mathrm{~N}=0
$$

$$
\mathbf{P}=724 \mathrm{~N}<53.4^{\circ}
$$

For $\alpha=-53.377^{\circ}$ :

$$
+\dagger \Sigma F_{y}=0: \quad 2 P\left(\frac{16}{\sqrt{281}}\right)+P \sin \left(-53.377^{\circ}\right)-1962 \mathrm{~N}=0
$$

$$
\mathbf{P}=1773\left\ulcorner 53.4^{\circ}\right.
$$



## SOLUTION

## Free-Body Diagram of Pulley

(a)

$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 2 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{2}(600 \mathrm{lb})
\end{aligned}
$$

160016

$$
T=300 \mathrm{lb}
$$

(b)


$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 2 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{2}(600 \mathrm{lb})
\end{aligned}
$$

$$
T=300 \mathrm{lb}
$$

(c)

$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 3 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{3}(600 \mathrm{lb})
\end{aligned}
$$

(d)

$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 3 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{3}(600 \mathrm{lb})
\end{aligned}
$$

$T=200 \mathrm{lb}$


$$
T=200 \mathrm{lb}
$$

(e)

$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 4 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{4}(600 \mathrm{lb})
\end{aligned}
$$

$$
T=150.0 \mathrm{lb}
$$



## PROBLEM 2.68

Solve Parts $b$ and $d$ of Problem 2.67, assuming that the free end of the rope is attached to the crate.

PROBLEM 2.67 A 600-lb crate is supported by several rope-and-pulley arrangements as shown. Determine for each arrangement the tension in the rope. (See the hint for Problem 2.66.)

## SOLUTION

## Free-Body Diagram of Pulley and Crate

(b)


$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 3 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{3}(600 \mathrm{lb})
\end{aligned}
$$

$$
T=200 \mathrm{lb}
$$

(d)


$$
\begin{aligned}
+\uparrow \Sigma F_{y}=0: \quad 4 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{4}(600 \mathrm{lb})
\end{aligned}
$$

$$
T=150.0 \mathrm{lb}
$$



## SOLUTION

## Free-Body Diagram: Pulley C


(a) $\xrightarrow{+} \Sigma F_{x}=0: \quad T_{A C B}\left(\cos 25^{\circ}-\cos 55^{\circ}\right)-(750 \mathrm{~N}) \cos 55^{\circ}=0$

Hence: $\quad T_{A C B}=1292.88 \mathrm{~N}$

$$
T_{A C B}=1293 \mathrm{~N}
$$

(b) $\quad+\dagger \Sigma F_{y}=0: \quad T_{A C B}\left(\sin 25^{\circ}+\sin 55^{\circ}\right)+(750 \mathrm{~N}) \sin 55^{\circ}-Q=0$

$$
(1292.88 \mathrm{~N})\left(\sin 25^{\circ}+\sin 55^{\circ}\right)+(750 \mathrm{~N}) \sin 55^{\circ}-Q=0
$$

or

$$
Q=2219.8 \mathrm{~N} \quad Q=2220 \mathrm{~N}
$$

## PROBLEM 2.70

An $1800-\mathrm{N}$ load $\mathbf{Q}$ is applied to the pulley $C$, which can roll on the cable $A C B$. The pulley is held in the position shown by a second cable $C A D$, which passes over the pulley $A$ and supports a load $\mathbf{P}$. Determine $(a)$ the tension in cable $A C B$, (b) the magnitude of load $\mathbf{P}$.

## SOLUTION

## Free-Body Diagram: Pulley C

$$
\xrightarrow{+} \Sigma F_{x}=0: \quad T_{A C B}\left(\cos 25^{\circ}-\cos 55^{\circ}\right)-P \cos 55^{\circ}=0
$$


or

$$
\begin{equation*}
P=0.58010 T_{A C B} \tag{1}
\end{equation*}
$$

$+\dagger \Sigma F_{y}=0: \quad T_{A C B}\left(\sin 25^{\circ}+\sin 55^{\circ}\right)+P \sin 55^{\circ}-1800 \mathrm{~N}=0$
or

$$
\begin{equation*}
1.24177 T_{A C B}+0.81915 P=1800 \mathrm{~N} \tag{2}
\end{equation*}
$$

(a) Substitute Equation (1) into Equation (2):

$$
1.24177 T_{A C B}+0.81915\left(0.58010 T_{A C B}\right)=1800 \mathrm{~N}
$$

Hence:

$$
T_{A C B}=1048.37 \mathrm{~N}
$$

$$
T_{A C B}=1048 \mathrm{~N}
$$

(b) Using (1), $\quad P=0.58010(1048.37 \mathrm{~N})=608.16 \mathrm{~N}$

$$
P=608 \mathrm{~N}
$$



## SOLUTION

(a)
$F_{x}=(600 \mathrm{~N}) \sin 25^{\circ} \cos 30^{\circ}$
$F_{x}=219.60 \mathrm{~N} \quad F_{x}=220 \mathrm{~N}$
$F_{y}=(600 \mathrm{~N}) \cos 25^{\circ}$
$F_{y}=543.78 \mathrm{~N} \quad F_{y}=544 \mathrm{~N}$
$F_{z}=(380.36 \mathrm{~N}) \sin 25^{\circ} \sin 30^{\circ}$
$F_{z}=126.785 \mathrm{~N}$
$F_{z}=126.8 \mathrm{~N}$
(b)
$\cos \theta_{x}=\frac{F_{x}}{F}=\frac{219.60 \mathrm{~N}}{600 \mathrm{~N}}$
$\theta_{x}=68.5^{\circ}$
$\cos \theta_{y}=\frac{F_{y}}{F}=\frac{543.78 \mathrm{~N}}{600 \mathrm{~N}}$
$\theta_{y}=25.0^{\circ}$
$\cos \theta_{z}=\frac{F_{z}}{F}=\frac{126.785 \mathrm{~N}}{600 \mathrm{~N}}$

$$
\theta_{z}=77.8^{\circ}
$$



## SOLUTION

(a)
$F_{x}=-(450 \mathrm{~N}) \cos 35^{\circ} \sin 40^{\circ}$
$F_{x}=-236.94 \mathrm{~N} \quad F_{x}=-237 \mathrm{~N}$
$F_{y}=(450 \mathrm{~N}) \sin 35^{\circ}$
$F_{y}=258.11 \mathrm{~N} \quad F_{y}=258 \mathrm{~N}$
$F_{z}=(450 \mathrm{~N}) \cos 35^{\circ} \cos 40^{\circ}$
$F_{z}=282.38 \mathrm{~N}$
$F_{z}=282 \mathrm{~N}$
$\cos \theta_{x}=\frac{F_{x}}{F}=\frac{-236.94 \mathrm{~N}}{450 \mathrm{~N}}$
$\theta_{x}=121.8^{\circ}$
$\cos \theta_{y}=\frac{F_{y}}{F}=\frac{258.11 \mathrm{~N}}{450 \mathrm{~N}} \quad \theta_{y}=55.0^{\circ}$
$\cos \theta_{z}=\frac{F_{z}}{F}=\frac{282.38 \mathrm{~N}}{450 \mathrm{~N}} \quad \theta_{z}=51.1^{\circ}$
Note: From the given data, we could have computed directly $\theta_{y}=90^{\circ}-35^{\circ}=55^{\circ}$, which checks with the answer obtained.

## PROBLEM 2.73

A gun is aimed at a point $A$ located $35^{\circ}$ east of north. Knowing that the barrel of the gun forms an angle of $40^{\circ}$ with the horizontal and that the maximum recoil force is 400 N , determine (a) the $x, y$, and $z$ components of that force, $(b)$ the values of the angles $\theta_{x}, \theta_{y}$, and $\theta_{z}$ defining the direction of the recoil force. (Assume that the $x, y$, and $z$ axes are directed, respectively, east, up, and south.)

## SOLUTION

Recoil force

$$
\begin{aligned}
F & =400 \mathrm{~N} \\
\therefore \quad F_{H} & =(400 \mathrm{~N}) \cos 40^{\circ} \\
& =306.42 \mathrm{~N}
\end{aligned}
$$


(a)

$$
\begin{aligned}
F_{x} & =-F_{H} \sin 35^{\circ} & \\
& =-(306.42 \mathrm{~N}) \sin 35^{\circ} & F_{x}=-175.8 \mathrm{~N} \\
& =-175.755 \mathrm{~N} & \\
F_{y} & =-F \sin 40^{\circ} & \\
& =-(400 \mathrm{~N}) \sin 40^{\circ} & F_{y}=-257 \mathrm{~N}
\end{aligned}
$$

$$
F_{z}=+F_{H} \cos 35^{\circ}
$$

$$
=+(306.42 \mathrm{~N}) \cos 35^{\circ}
$$

$$
=+251.00 \mathrm{~N}
$$

$$
F_{z}=+251 \mathrm{~N}
$$

(b)

$$
\begin{array}{rlr}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{-175.755 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{x}=116.1^{\circ} \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-257.12 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{y}=130.0^{\circ} \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{251.00 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{z}=51.1^{\circ}
\end{array}
$$

## PROBLEM 2.74

Solve Problem 2.73, assuming that point $A$ is located $15^{\circ}$ north of west and that the barrel of the gun forms an angle of $25^{\circ}$ with the horizontal.

PROBLEM 2.73 A gun is aimed at a point $A$ located $35^{\circ}$ east of north. Knowing that the barrel of the gun forms an angle of $40^{\circ}$ with the horizontal and that the maximum recoil force is 400 N , determine (a) the $x, y$, and $z$ components of that force, (b) the values of the angles $\theta_{x}, \theta_{y}$, and $\theta_{z}$ defining the direction of the recoil force. (Assume that the $x, y$, and $z$ axes are directed, respectively, east, up, and south.)

## SOLUTION

Recoil force

$$
\begin{aligned}
F & =400 \mathrm{~N} \\
\therefore \quad F_{H} & =(400 \mathrm{~N}) \cos 25^{\circ} \\
& =362.52 \mathrm{~N}
\end{aligned}
$$


(a)

$$
\begin{aligned}
F_{x} & =+F_{H} \cos 15^{\circ} \\
& =+(362.52 \mathrm{~N}) \cos 15^{\circ}
\end{aligned}
$$

$$
=+350.17 \mathrm{~N} \quad F_{x}=+350 \mathrm{~N}
$$

$$
F_{y}=-F \sin 25^{\circ}
$$

$$
=-(400 \mathrm{~N}) \sin 25^{\circ}
$$

$$
=-169.047 \mathrm{~N}
$$

$$
F_{y}=-169.0 \mathrm{~N}
$$

$$
F_{z}=+F_{H} \sin 15^{\circ}
$$

$$
=+(362.52 \mathrm{~N}) \sin 15^{\circ}
$$

$$
=+93.827 \mathrm{~N} \quad F_{z}=+93.8 \mathrm{~N}
$$

(b)

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{+350.17 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{x}=28.9^{\circ} \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-169.047 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{y}=115.0^{\circ} \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{+93.827 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{z}=76.4^{\circ}
\end{array}
$$



## SOLUTION

$$
\begin{aligned}
F_{h} & =F \cos 60^{\circ} \\
& =(50 \mathrm{lb}) \cos 60^{\circ} \\
F_{h} & =25.0 \mathrm{lb}
\end{aligned}
$$

| $F_{x}=-F_{h} \cos 35^{\circ}$ | $F_{y}=F \sin 60^{\circ}$ | $F_{z}=-F_{h} \sin 35^{\circ}$ |
| :--- | :--- | :--- |
| $F_{x}=(-25.0 \mathrm{lb}) \cos 35^{\circ}$ | $F_{y}=(50.0 \mathrm{lb}) \sin 60^{\circ}$ | $F_{z}=(-25.0 \mathrm{lb}) \sin 35^{\circ}$ |
| $F_{x}=-20.479 \mathrm{lb}$ | $F_{y}=43.301 \mathrm{lb}$ | $F_{z}=-14.3394 \mathrm{lb}$ |

(a)

$$
\begin{gathered}
F_{x}=-20.5 \mathrm{lb} \\
F_{y}=43.3 \mathrm{lb}
\end{gathered}
$$

$$
F_{z}=-14.33 \mathrm{lb}
$$

(b)

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{-20.479 \mathrm{lb}}{50 \mathrm{lb}} & \theta_{x}=114.2^{\circ} \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{43.301 \mathrm{lb}}{50 \mathrm{lb}} & \theta_{y}=30.0^{\circ} \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{-14.3394 \mathrm{lb}}{50 \mathrm{lb}} & \theta_{z}=106.7^{\circ}
\end{array}
$$



## SOLUTION

$$
\begin{aligned}
F_{h} & =F \cos 60^{\circ} \\
& =(40 \mathrm{lb}) \cos 60^{\circ} \\
F_{h} & =20.0 \mathrm{lb}
\end{aligned}
$$


(a)

$$
\begin{aligned}
F_{x} & =F_{h} \cos 35^{\circ} & F_{y} & =F \sin 60^{\circ}
\end{aligned} \begin{array}{rlrl} 
& =-F_{h} \sin 35^{\circ} \\
& =(20.0 \mathrm{lb}) \cos 35^{\circ} & & =(40 \mathrm{lb}) \sin 60^{\circ}
\end{array} \begin{array}{ll} 
& =-(20.0 \mathrm{lb}) \sin 35^{\circ} \\
F_{x} & =16.3830 \mathrm{lb}
\end{array} \quad F_{y}=34.641 \mathrm{lb} \quad F_{z}=-11.4715 \mathrm{lb}
$$

$$
\begin{gathered}
F_{x}=16.38 \mathrm{lb} \\
F_{y}=34.6 \mathrm{lb}
\end{gathered}
$$

$$
F_{z}=-11.47 \mathrm{lb}
$$

(b)

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{16.3830 \mathrm{lb}}{40 \mathrm{lb}} & \theta_{x}=65.8^{\circ} \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{34.641 \mathrm{lb}}{40 \mathrm{lb}} & \theta_{y}=30.0^{\circ} \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{-11.4715 \mathrm{lb}}{40 \mathrm{lb}} & \theta_{z}=106.7^{\circ}
\end{array}
$$





| SOLUTION |  |
| ---: | :--- |
| $A C$ | $=70 \mathrm{ft}$ |
| $O A$ | $=56 \mathrm{ft}$ |
| $F$ | $=5250 \mathrm{lb}$ |
| $\cos \theta_{y}$ | $=\frac{56 \mathrm{ft}}{70 \mathrm{ft}}$ |
| $\theta_{y}$ | $=36.870^{\circ}$ |
| $F_{H}$ | $=F \sin \theta_{y}$ |
|  | $=(5250 \mathrm{lb}) \sin 36.870^{\circ}$ |
|  | $=3150.0 \mathrm{lb}$ |

(a) $F_{x}=-F_{H} \sin 50^{\circ}=-(3150.0 \mathrm{lb}) \sin 50^{\circ}=-2413.0 \mathrm{lb}$

$$
F_{x}=-2410 \mathrm{lb}
$$

$$
F_{y}=+F \cos \theta_{y}=+(5250 \mathrm{lb}) \cos 36.870^{\circ}=+4200.0 \mathrm{lb}
$$

$$
\begin{array}{ll}
F_{z}=-F_{H} \cos 50^{\circ}=-3150 \cos 50^{\circ}=-2024.8 \mathrm{lb} & F_{z}=-2025 \mathrm{lb}
\end{array}
$$

(b) $\cos \theta_{x}=\frac{F_{x}}{F}=\frac{-2413.0 \mathrm{lb}}{5250 \mathrm{lb}} \quad \theta_{x}=117.4^{\circ}$

From above: $\quad \theta_{y}=36.870^{\circ} \quad \theta_{y}=36.9^{\circ}$

$$
\theta_{z}=\frac{F_{z}}{F}=\frac{-2024.8 \mathrm{lb}}{5250 \mathrm{lb}} \quad \theta_{z}=112.7^{\circ}
$$

## PROBLEM 2.79

Determine the magnitude and direction of the force $\mathbf{F}=(240 \mathrm{~N}) \mathbf{i}-(270 \mathrm{~N}) \mathbf{j}+(680 \mathrm{~N}) \mathbf{k}$.

## SOLUTION

$$
\begin{array}{rlrl}
F & =\sqrt{F_{x}^{2}+F_{y}^{2}+F_{z}^{2}} & & \\
F & =\sqrt{(240 \mathrm{~N})^{2}+(-270 \mathrm{~N})^{2}+(-680 \mathrm{~N})^{2}} & F & =770 \mathrm{~N} . \\
\cos \theta_{x} & =\frac{F_{x}}{F}=\frac{240 \mathrm{~N}}{770 \mathrm{~N}} & \theta_{x}=71.8^{\circ} . \\
\cos \theta_{y} & =\frac{F_{y}}{F}=\frac{-270 \mathrm{~N}}{770 \mathrm{~N}} & \theta_{y}=110.5^{\circ} . \\
\cos \theta_{y} & =\frac{F_{z}}{F}=\frac{680 \mathrm{~N}}{770 \mathrm{~N}} & \theta_{z}=28.0^{\circ} .
\end{array}
$$

## PROBLEM 2.80

Determine the magnitude and direction of the force $\mathbf{F}=(320 \mathrm{~N}) \mathbf{i}+(400 \mathrm{~N}) \mathbf{j}-(250 \mathrm{~N}) \mathbf{k}$.

## SOLUTION

$$
\begin{array}{rlrl}
F & =\sqrt{F_{x}^{2}+F_{y}^{2}+F_{z}^{2}} & & \\
F & =\sqrt{(320 \mathrm{~N})^{2}+(400 \mathrm{~N})^{2}+(-250 \mathrm{~N})^{2}} & F & =570 \mathrm{~N} \\
\cos \theta_{x} & =\frac{F_{x}}{F}=\frac{320 \mathrm{~N}}{570 \mathrm{~N}} & \theta_{x}=55.8^{\circ} \\
\cos \theta_{y} & =\frac{F_{y}}{F}=\frac{400 \mathrm{~N}}{570 \mathrm{~N}} & \theta_{y}=45.4^{\circ} \\
\cos \theta_{y} & =\frac{F_{z}}{F}=\frac{-250 \mathrm{~N}}{570 \mathrm{~N}} & \theta_{z} & =116.0^{\circ}
\end{array}
$$

## PROBLEM 2.81

A force acts at the origin of a coordinate system in a direction defined by the angles $\theta_{x}=69.3^{\circ}$ and $\theta_{z}$ $=57.9^{\circ}$. Knowing that the $y$ component of the force is -174.0 lb , determine $(a)$ the angle $\theta_{y},(b)$ the other components and the magnitude of the force.

## SOLUTION

$$
\begin{aligned}
\cos ^{2} \theta_{x}+\cos ^{2} \theta_{y}+\cos ^{2} \theta_{z} & =1 \\
\cos ^{2}\left(69.3^{\circ}\right)+\cos ^{2} \theta_{y}+\cos ^{2}\left(57.9^{\circ}\right) & =1 \\
\cos \theta_{y} & = \pm 0.7699
\end{aligned}
$$

(a) Since $F_{y}<0$, we choose $\cos \theta_{y}=-0.7699$ $\therefore \quad \theta_{y}=140.3^{\circ}$
(b)

$$
\begin{array}{cc}
F_{y}=F \cos \theta_{y} & \\
-174.0 \mathrm{lb}=F(-0.7699) & \\
F=226.0 \mathrm{lb} & F=226 \mathrm{lb} \\
F_{x}=F \cos \theta_{x}=(226.0 \mathrm{lb}) \cos 69.3^{\circ} & F_{x}=79.9 \mathrm{lb} \\
F_{z}=F \cos \theta_{z}=(226.0 \mathrm{lb}) \cos 57.9^{\circ} & F_{z}=120.1 \mathrm{lb}
\end{array}
$$

## PROBLEM 2.82

A force acts at the origin of a coordinate system in a direction defined by the angles $\theta_{x}=70.9^{\circ}$ and $\theta_{y}=144.9^{\circ}$. Knowing that the $z$ component of the force is -52.0 lb , determine (a) the angle $\theta_{z}$, (b) the other components and the magnitude of the force.

## SOLUTION

$$
\begin{aligned}
\cos ^{2} \theta_{x}+\cos ^{2} \theta_{y}+\cos ^{2} \theta_{z} & =1 \\
\cos ^{2} 70.9^{\circ}+\cos ^{2} 144.9^{\circ}+\cos ^{2} \theta_{z}^{\circ} & =1 \\
\cos \theta_{z} & = \pm 0.47282
\end{aligned}
$$

(a) Since $F_{z}<0$, we choose $\cos \theta_{z}=-0.47282$ $\therefore \quad \theta_{z}=118.2^{\circ}$
(b)

$$
\begin{array}{rlrl}
F_{z} & =F \cos \theta_{z} & \\
-52.0 l b & =F(-0.47282) & \\
F & =110.0 \mathrm{lb} & F=110.0 \mathrm{lb} \\
F_{x} & =F \cos \theta_{x}=(110.0 \mathrm{lb}) \cos 70.9^{\circ} & F_{x}=36.0 \mathrm{lb} \\
F_{y} & =F \cos \theta_{y}=(110.0 \mathrm{lb}) \cos 144.9^{\circ} & F_{y}=-90.0 \mathrm{lb}
\end{array}
$$

## PROBLEM 2.83

A force $\mathbf{F}$ of magnitude 210 N acts at the origin of a coordinate system. Knowing that $F_{x}=80 \mathrm{~N}$, $\theta_{z}=151.2^{\circ}$, and $F_{y}<0$, determine $(a)$ the components $F_{y}$ and $F_{z},(b)$ the angles $\theta_{x}$ and $\theta_{y}$.

## SOLUTION

(a)

$$
\begin{array}{rlr}
F_{z}=F \cos \theta_{z} & =(210 \mathrm{~N}) \cos 151.2^{\circ} \\
& =-184.024 \mathrm{~N} & F_{z}=-184.0 \mathrm{~N}
\end{array}
$$

Then:

$$
F^{2}=F_{x}^{2}+F_{y}^{2}+F_{z}^{2}
$$

So:

$$
(210 \mathrm{~N})^{2}=(80 \mathrm{~N})^{2}+\left(F_{y}\right)^{2}+(184.024 \mathrm{~N})^{2}
$$

Hence:

$$
\begin{aligned}
F_{y} & =-\sqrt{(210 \mathrm{~N})^{2}-(80 \mathrm{~N})^{2}-(184.024 \mathrm{~N})^{2}} & \\
& =-61.929 \mathrm{~N} & F_{y}=-62.0 \mathrm{lb}
\end{aligned}
$$

$$
\begin{aligned}
& \cos \theta_{x}=\frac{F_{x}}{F}=\frac{80 \mathrm{~N}}{210 \mathrm{~N}}=0.38095 \\
& \cos \theta_{y}=\frac{F_{y}}{F}=\frac{61.929 \mathrm{~N}}{210 \mathrm{~N}}=-0.29490
\end{aligned}
$$

$$
\theta_{x}=67.6^{\circ}
$$

$$
\theta_{y}=107.2^{\circ}
$$

## PROBLEM 2.84

A force $\mathbf{F}$ of magnitude 1200 N acts at the origin of a coordinate system. Knowing that $\theta_{x}=65^{\circ}, \theta_{y}=40^{\circ}$, and $F_{z}>0$, determine (a) the components of the force, (b) the angle $\theta_{z}$.

## SOLUTION

$$
\begin{aligned}
\cos ^{2} \theta_{x}+\cos ^{2} \theta_{y}+\cos ^{2} \theta_{z} & =1 \\
\cos ^{2} 65^{\circ}+\cos ^{2} 40^{\circ}+\cos ^{2} \theta_{z}^{\circ} & =1 \\
\cos \theta_{z} & = \pm 0.48432
\end{aligned}
$$

(b) Since $F_{z}>0$, we choose $\cos \theta_{z}=0.48432$, or $\theta_{z}=61.032^{\circ}$ $\therefore \quad \theta_{z}=61.0^{\circ}$
(a)

$$
F=1200 \mathrm{~N}
$$

$$
\begin{array}{ll}
F_{x}=F \cos \theta_{x}=(1200 \mathrm{~N}) \cos 65^{\circ} & F_{x}=507 \mathrm{~N} \\
F_{y}=F \cos \theta_{y}=(1200 \mathrm{~N}) \cos 40^{\circ} & F_{y}=919 \mathrm{~N} \\
F_{z}=F \cos \theta_{z}=(1200 \mathrm{~N}) \cos 61.032^{\circ} & F_{z}=582 \mathrm{~N}
\end{array}
$$



## SOLUTION

$$
\begin{aligned}
\overrightarrow{D B} & =(480 \mathrm{~mm}) \mathbf{i}-(510 \mathrm{~mm}) \mathbf{j}+(320 \mathrm{~mm}) \mathbf{k} \\
D B & =\sqrt{(480 \mathrm{~mm})^{2}+\left(510 \mathrm{~mm}^{2}\right)+(320 \mathrm{~mm})^{2}} \\
& =770 \mathrm{~mm} \\
\mathbf{F} & =F \boldsymbol{\lambda}_{D B} \\
& =F \frac{\overrightarrow{D B}}{D B} \\
& =\frac{385 \mathrm{~N}}{770 \mathrm{~mm}}[(480 \mathrm{~mm}) \mathbf{i}-(510 \mathrm{~mm}) \mathbf{j}+(320 \mathrm{~mm}) \mathbf{k}] \\
& =(240 \mathrm{~N}) \mathbf{i}-(255 \mathrm{~N}) \mathbf{j}+(160 \mathrm{~N}) \mathbf{k} \\
& F_{x}=+240 \mathrm{~N}, \quad F_{y}=-255 \mathrm{~N}, \quad F_{z}=+160.0 \mathrm{~N}
\end{aligned}
$$



## SOLUTION

$$
\begin{aligned}
\overrightarrow{E B} & =(270 \mathrm{~mm}) \mathbf{i}-(400 \mathrm{~mm}) \mathbf{j}+(600 \mathrm{~mm}) \mathbf{k} \\
E B & =\sqrt{(270 \mathrm{~mm})^{2}+(400 \mathrm{~mm})^{2}+(600 \mathrm{~mm})^{2}} \\
& =770 \mathrm{~mm} \\
\mathbf{F} & =F \lambda_{E B} \\
& =F \frac{\overrightarrow{E B}}{E B} \\
& =\frac{385 \mathrm{~N}}{770 \mathrm{~mm}}[(270 \mathrm{~mm}) \mathbf{i}-(400 \mathrm{~mm}) \mathbf{j}+(600 \mathrm{~mm}) \mathbf{k}] \\
\mathbf{F} & =(135 \mathrm{~N}) \mathbf{i}-(200 \mathrm{~N}) \mathbf{j}+(300 \mathrm{~N}) \mathbf{k} \\
& F_{x}=+135.0 \mathrm{~N}, \quad F_{y}=-200 \mathrm{~N}, \quad F_{z}=+300 \mathrm{~N}
\end{aligned}
$$



## SOLUTION



$$
\Delta B=74.216 \mathrm{ft} \quad A C=85.590 \mathrm{ft}
$$

Cable $A B$ :

$$
\begin{aligned}
& \lambda_{A B}=\frac{\overrightarrow{A B}}{A B}=\frac{(-46.765 \mathrm{ft}) \mathbf{i}+(45 \mathrm{ft}) \mathbf{j}+(36 \mathrm{ft}) \mathbf{k}}{74.216 \mathrm{ft}} \\
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=\frac{-46.765 \mathbf{i}+45 \mathbf{j}+36 \mathbf{k}}{74.216}
\end{aligned}
$$

$$
\left(T_{A B}\right)_{x}=-1.260 \mathrm{kips}
$$

$$
\left(T_{A B}\right)_{y}=+1.213 \mathrm{kips}
$$

$$
\left(T_{A B}\right)_{z}=+0.970 \mathrm{kips}
$$



## SOLUTION



Cable $A B$ :

$$
\begin{aligned}
& \lambda_{A C}=\frac{\overrightarrow{A C}}{A C}=\frac{(-46.765 \mathrm{ft}) \mathbf{i}+(55.8 \mathrm{ft}) \mathbf{j}+(-45 \mathrm{ft}) \mathbf{k}}{85.590 \mathrm{ft}} \\
& \mathbf{T}_{A C}=T_{A C} \boldsymbol{\lambda}_{A C}=(1.5 \mathrm{kips}) \frac{-46.765 \mathbf{i}+55.8 \mathbf{j}-45 \mathbf{k}}{85.590}
\end{aligned}
$$

$$
\left(T_{A C}\right)_{x}=-0.820 \mathrm{kips}
$$

$$
\left(T_{A C}\right)_{y}=+0.978 \mathrm{kips}
$$

$$
\left(T_{A C}\right)_{z}=-0.789 \mathrm{kips}
$$



## PROBLEM 2.89

A rectangular plate is supported by three cables as shown. Knowing that the tension in cable $A B$ is 408 N , determine the components of the force exerted on the plate at $B$.

## SOLUTION

We have:

$$
\overrightarrow{B A}=+(320 \mathrm{~mm}) \mathbf{i}+(480 \mathrm{~mm}) \mathbf{j}-(360 \mathrm{~mm}) \mathbf{k} \quad B A=680 \mathrm{~mm}
$$

Thus:

$$
\begin{gathered}
\mathrm{F}_{B}=T_{B A} \lambda_{B A}=T_{B A} \frac{\overrightarrow{B A}}{B A}=T_{B A}\left(\frac{8}{17} \mathbf{i}+\frac{12}{17} \mathbf{j}-\frac{9}{17} \mathbf{k}\right) \\
\left(\frac{8}{17} T_{B A}\right) \mathbf{i}+\left(\frac{12}{17} T_{B A}\right) \mathbf{j}-\left(\frac{9}{17} T_{B A}\right) \mathbf{k}=0
\end{gathered}
$$

Setting $T_{B A}=408 \mathrm{~N}$ yields,

$$
F_{x}=+192.0 \mathrm{~N}, \quad F_{y}=+288 \mathrm{~N}, \quad F_{z}=-216 \mathrm{~N}
$$



## SOLUTION

We have:

$$
\overrightarrow{D A}=-(250 \mathrm{~mm}) \mathbf{i}+(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} \quad D A=650 \mathrm{~mm}
$$

Thus:

$$
\begin{aligned}
\mathrm{F}_{D}= & T_{D A} \lambda_{D A}=T_{D A} \frac{\overrightarrow{D A}}{D A}=T_{D A}\left(-\frac{5}{13} \mathbf{i}+\frac{48}{65} \mathbf{j}+\frac{36}{65} \mathbf{k}\right) \\
& -\left(\frac{5}{13} T_{D A}\right) \mathbf{i}+\left(\frac{48}{65} T_{D A}\right) \mathbf{j}+\left(\frac{36}{65} T_{D A}\right) \mathbf{k}=0
\end{aligned}
$$

Setting $T_{D A}=429 \mathrm{~N}$ yields,

$$
F_{x}=-165.0 \mathrm{~N}, \quad F_{y}=+317 \mathrm{~N}, \quad F_{z}=+238 \mathrm{~N}
$$



## SOLUTION

$$
\begin{array}{rlrl}
\mathbf{P} & =(300 \mathrm{~N})\left[-\cos 30^{\circ} \sin 15^{\circ} \mathbf{i}+\sin 30^{\circ} \mathbf{j}+\cos 30^{\circ} \cos 15^{\circ} \mathbf{k}\right] & \\
& =-(67.243 \mathrm{~N}) \mathbf{i}+(150 \mathrm{~N}) \mathbf{j}+(250.95 \mathrm{~N}) \mathbf{k} & \\
\mathbf{Q} & =(400 \mathrm{~N})\left[\cos 50^{\circ} \cos 20^{\circ} \mathbf{i}+\sin 50^{\circ} \mathbf{j}-\cos 50^{\circ} \sin 20^{\circ} \mathbf{k}\right] & & \\
& =(400 \mathrm{~N})[0.60402 \mathbf{i}+0.76604 \mathbf{j}-0.21985] & \\
& =(241.61 \mathrm{~N}) \mathbf{i}+(306.42 \mathrm{~N}) \mathbf{j}-(87.939 \mathrm{~N}) \mathbf{k} & & \\
\mathbf{R} & =\mathbf{P}+\mathbf{Q} & & R=515 \mathrm{~N} \\
& =(174.367 \mathrm{~N}) \mathbf{i}+(456.42 \mathrm{~N}) \mathbf{j}+(163.011 \mathrm{~N}) \mathbf{k} & \\
R & =\sqrt{(174.367 \mathrm{~N})^{2}+(456.42 \mathrm{~N})^{2}+(163.011 \mathrm{~N})^{2}} & \theta_{x}=70.2^{\circ} \\
& =515.07 \mathrm{~N} & & \theta_{y}=27.6^{\circ} \\
\cos \theta_{x} & =\frac{R_{x}}{R}=\frac{174.367 \mathrm{~N}}{515.07 \mathrm{~N}}=0.33853 & & \theta_{z}=71.5^{\circ} \\
\cos \theta_{y} & =\frac{R_{y}}{R}=\frac{456.42 \mathrm{~N}}{515.07 \mathrm{~N}}=0.88613 & & \\
\cos \theta_{z} & =\frac{R_{z}}{R}=\frac{163.011 \mathrm{~N}}{515.07 \mathrm{~N}}=0.31648 &
\end{array}
$$



## SOLUTION

$$
\begin{array}{rlrl}
\mathbf{P} & =(400 \mathrm{~N})\left[-\cos 30^{\circ} \sin 15^{\circ} \mathbf{i}+\sin 30^{\circ} \mathbf{j}+\cos 30^{\circ} \cos 15^{\circ} \mathbf{k}\right] & & \\
& =-(89.678 \mathrm{~N}) \mathbf{i}+(200 \mathrm{~N}) \mathbf{j}+(334.61 \mathrm{~N}) \mathbf{k} & \\
\mathbf{Q} & =(300 \mathrm{~N})\left[\cos 50^{\circ} \cos 20^{\circ} \mathbf{i}+\sin 50^{\circ} \mathbf{j}-\cos 50^{\circ} \sin 20^{\circ} \mathbf{k}\right] & & \\
& =(181.21 \mathrm{~N}) \mathbf{i}+(229.81 \mathrm{~N}) \mathbf{j}-(65.954 \mathrm{~N}) \mathbf{k} & R & \mathbf{R} \\
& =\mathbf{P}+\mathbf{Q} & & \theta_{x}=79.8^{\circ} \\
& =(91.532 \mathrm{~N}) \mathbf{i}+(429.81 \mathrm{~N}) \mathbf{j}+(268.66 \mathrm{~N}) \mathbf{k} & \\
R & =\sqrt{(91.532 \mathrm{~N})^{2}+(429.81 \mathrm{~N})^{2}+(268.66 \mathrm{~N})^{2}} & \theta_{y}=33.4^{\circ} \\
& =515.07 \mathrm{~N} & \theta_{z}=58.6^{\circ} \\
\cos \theta_{x} & =\frac{R_{x}}{R}=\frac{91.532 \mathrm{~N}}{515.07 \mathrm{~N}}=0.177708 & & \\
\cos \theta_{y} & =\frac{R_{y}}{R}=\frac{429.81 \mathrm{~N}}{515.07 \mathrm{~N}}=0.83447 & & R_{z} \\
\cos \theta_{z} & =\frac{268.66 \mathrm{~N}}{R}=0.52160 & 515.07 \mathrm{~N} &
\end{array}
$$



## PROBLEM 2.93

Knowing that the tension is 425 lb in cable $A B$ and 510 lb in cable $A C$, determine the magnitude and direction of the resultant of the forces exerted at $A$ by the two cables.

## SOLUTION

$$
\begin{aligned}
\overrightarrow{A B} & =(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
A B & =\sqrt{(40 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}}=85 \mathrm{in} . \\
\overrightarrow{A C} & =(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
A C & =\sqrt{(100 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}}=125 \mathrm{in} . \\
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=(425 \mathrm{lb})\left[\frac{(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{85 \mathrm{in} .}\right] \\
\mathbf{T}_{A B} & =(200 \mathrm{lb}) \mathbf{i}-(225 \mathrm{lb}) \mathbf{j}+(300 \mathrm{lb}) \mathbf{k} \\
\mathbf{T}_{A C} & =T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=(510 \mathrm{lb})\left[\frac{(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in.}) \mathbf{k}}{125 \mathrm{in} .}\right] \\
\mathbf{T}_{A C} & =(408 \mathrm{lb}) \mathbf{i}-(183.6 \mathrm{lb}) \mathbf{j}+(244.8 \mathrm{lb}) \mathbf{k} \\
\mathbf{R} & =\mathbf{T}_{A B}+\mathbf{T}_{A C}=(608) \mathbf{i}-(408.6 \mathrm{lb}) \mathbf{j}+(544.8 \mathrm{lb}) \mathbf{k}
\end{aligned}
$$

Then:

$$
R=912.92 \mathrm{lb}
$$

$$
R=913 \mathrm{lb}
$$

and

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{608 \mathrm{lb}}{912.92 \mathrm{lb}}=0.66599 & \theta_{x}=48.2^{\circ} \\
\cos \theta_{y}=\frac{408.6 \mathrm{lb}}{912.92 \mathrm{lb}}=-0.44757 & \theta_{y}=116.6^{\circ} \\
\cos \theta_{z}=\frac{544.8 \mathrm{lb}}{912.92 \mathrm{lb}}=0.59677 & \theta_{z}=53.4^{\circ}
\end{array}
$$



## PROBLEM 2.94

Knowing that the tension is 510 lb in cable $A B$ and 425 lb in cable $A C$, determine the magnitude and direction of the resultant of the forces exerted at $A$ by the two cables.

## SOLUTION

$$
\begin{aligned}
\overrightarrow{A B} & =(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
A B & =\sqrt{(40 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}}=85 \mathrm{in} . \\
\overrightarrow{A C} & =(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
A C & =\sqrt{(100 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}}=125 \mathrm{in} . \\
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=(510 \mathrm{lb})\left[\frac{(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{85 \mathrm{in} .}\right] \\
\mathbf{T}_{A B} & =(240 \mathrm{lb}) \mathbf{i}-(270 \mathrm{lb}) \mathbf{j}+(360 \mathrm{lb}) \mathbf{k} \\
\mathbf{T}_{A C} & =T_{A C} \boldsymbol{\lambda}_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=(425 \mathrm{lb})\left[\frac{(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{125 \mathrm{in} .}\right] \\
\mathbf{T}_{A C} & =(340 \mathrm{lb}) \mathbf{i}-(153 \mathrm{lb}) \mathbf{j}+(204 \mathrm{lb}) \mathbf{k} \\
\mathbf{R} & =\mathbf{T}_{A B}+\mathbf{T}_{A C}=(580 \mathrm{lb}) \mathbf{i}-(423 \mathrm{lb}) \mathbf{j}+(564 \mathrm{lb}) \mathbf{k}
\end{aligned}
$$

Then:

$$
R=912.92 \mathrm{lb}
$$

$$
R=913 \mathrm{lb}
$$

and

$$
\cos \theta_{x}=\frac{580 \mathrm{lb}}{912.92 \mathrm{lb}}=0.63532
$$

$$
\theta_{x}=50.6^{\circ}
$$

$$
\cos \theta_{y}=\frac{-423 \mathrm{lb}}{912.92 \mathrm{lb}}=-0.46335
$$

$$
\theta_{y}=117.6^{\circ}
$$

$$
\cos \theta_{z}=\frac{564 \mathrm{lb}}{912.92 \mathrm{lb}}=0.61780
$$

$$
\theta_{z}=51.8^{\circ}
$$



## PROBLEM 2.95

For the frame of Problem 2.85, determine the magnitude and direction of the resultant of the forces exerted by the cable at $B$ knowing that the tension in the cable is 385 N .

PROBLEM 2.85 A frame $A B C$ is supported in part by cable $D B E$ that passes through a frictionless ring at $B$. Knowing that the tension in the cable is 385 N , determine the components of the force exerted by the cable on the support at $D$.

## SOLUTION

$$
\begin{array}{rlrl}
\overrightarrow{B D} & =-(480 \mathrm{~mm}) \mathbf{i}+(510 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \\
B D & =\sqrt{(480 \mathrm{~mm})^{2}+(510 \mathrm{~mm})^{2}+(320 \mathrm{~mm})^{2}}=770 \mathrm{~mm} \\
\mathbf{F}_{B D} & =T_{B D} \lambda_{B D}=T_{B D} \frac{\overrightarrow{B D}}{B D} \\
& =\frac{(385 \mathrm{~N})}{(770 \mathrm{~mm})}[-(480 \mathrm{~mm}) \mathbf{i}+(510 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k}] & \\
& =-(240 \mathrm{~N}) \mathbf{i}+(255 \mathrm{~N}) \mathbf{j}-(160 \mathrm{~N}) \mathbf{k} \\
\overrightarrow{B E} & =-(270 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}-(600 \mathrm{~mm}) \mathbf{k} \\
B E & =\sqrt{(270 \mathrm{~mm})^{2}+(400 \mathrm{~mm})^{2}+(600 \mathrm{~mm})^{2}}=770 \mathrm{~mm} & \\
\mathbf{F}_{B E} & =T_{B E} \lambda_{B E}=T_{B E} \frac{\overrightarrow{B E}}{B E} & \\
& =\frac{(385 \mathrm{~N})}{(770 \mathrm{~mm})}[-(270 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}-(600 \mathrm{~mm}) \mathbf{k}] & \\
& =-(135 \mathrm{~N}) \mathbf{i}+(200 \mathrm{~N}) \mathbf{j}-(300 \mathrm{~N}) \mathbf{k} & \\
\mathbf{R} & =\mathbf{F}_{B D}+\mathbf{F}_{B E}=-(375 \mathrm{~N}) \mathbf{i}+(455 \mathrm{~N}) \mathbf{j}-(460 \mathrm{~N}) \mathbf{k} & \\
R & =\sqrt{(375 \mathrm{~N})^{2}+(455 \mathrm{~N})^{2}+(460 \mathrm{~N})^{2}}=747.83 \mathrm{~N} & R=748 \mathrm{~N} \\
\cos \theta_{x} & =\frac{-375 \mathrm{~N}}{747.83 \mathrm{~N}} & \theta_{x}=120.1^{\circ} \\
\cos \theta_{y} & =\frac{455 \mathrm{~N}}{747.83 \mathrm{~N}} & \theta_{y}=52.5^{\circ} \\
\cos \theta_{z} & =\frac{-460 \mathrm{~N}}{747.83 \mathrm{~N}} & \theta_{z}=128.0^{\circ}
\end{array}
$$



Dimensions in mm

## PROBLEM 2.96

For the plate of Prob. 2.89, determine the tensions in cables $A B$ and $A D$ knowing that the tension in cable $A C$ is 54 N and that the resultant of the forces exerted by the three cables at $A$ must be vertical.

## SOLUTION

We have:

$$
\begin{array}{ll}
\overrightarrow{A B}=-(320 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A B=680 \mathrm{~mm} \\
\overrightarrow{A C}=(450 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A C=750 \mathrm{~mm} \\
\overrightarrow{A D}=(250 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}-(360 \mathrm{~mm}) \mathbf{k} & A D=650 \mathrm{~mm}
\end{array}
$$

Thus:

$$
\begin{aligned}
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=\frac{T_{A B}}{680}(-320 \mathbf{i}-480 \mathbf{j}+360 \mathbf{k}) \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=\frac{54}{750}(450 \mathbf{i}-480 \mathbf{j}+360 \mathbf{k}) \\
& \mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D}=\frac{T_{A D}}{650}(250 \mathbf{i}-480 \mathbf{j}-360 \mathbf{k})
\end{aligned}
$$

Substituting into the Eq. $\mathbf{R}=\Sigma \mathbf{F}$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{aligned}
& \mathbf{R}=\left(-\frac{320}{680} T_{A B}+32.40+\frac{250}{650} T_{A D}\right) \mathbf{i} \\
& +\left(-\frac{480}{680} T_{A B}-34.560-\frac{480}{650} T_{A D}\right) \mathbf{j} \\
& +\left(\frac{360}{680} T_{A B}+25.920-\frac{360}{650} T_{A D}\right) \mathbf{k}
\end{aligned}
$$

## PROBLEM 2.96 (Continued)

Since $\mathbf{R}$ is vertical, the coefficients of $\mathbf{i}$ and $\mathbf{k}$ are zero:

$$
\begin{align*}
& \text { i: } \quad-\frac{320}{680} T_{A B}+32.40+\frac{250}{650} T_{A D}=0  \tag{1}\\
& \text { k: } \quad  \tag{2}\\
& \quad \frac{360}{680} T_{A B}+25.920-\frac{360}{650} T_{A D}=0
\end{align*}
$$

Multiply (1) by 3.6 and (2) by 2.5 then add:

$$
\begin{aligned}
& -\frac{252}{680} T_{A B}+181.440=0 \\
& T_{A B}=489.60 \mathrm{~N}
\end{aligned}
$$

$$
T_{A B}=490 \mathrm{~N}
$$

Substitute into (2) and solve for $T_{A D}$ :

$$
\begin{aligned}
\frac{360}{680}(489.60 \mathrm{~N})+25.920-\frac{360}{650} T_{A D} & =0 \\
T_{A D} & =514.80 \mathrm{~N}
\end{aligned}
$$

$$
T_{A D}=515 \mathrm{~N}
$$



## PROBLEM 2.97

The boom $O A$ carries a load $\mathbf{P}$ and is supported by two cables as shown. Knowing that the tension in cable $A B$ is 183 lb and that the resultant of the load $\mathbf{P}$ and of the forces exerted at $A$ by the two cables must be directed along $O A$, determine the tension in cable $A C$.

## SOLUTION



Cable $A B: \quad T_{A B}=183 \mathrm{lb}$

$$
\begin{aligned}
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=(183 \mathrm{lb}) \frac{(-48 \mathrm{in} .) \mathbf{i}+(29 \mathrm{in} .) \mathbf{j}+(24 \mathrm{in} .) \mathbf{k}}{61 \mathrm{in} .} \\
& \mathbf{T}_{A B}=-(144 \mathrm{lb}) \mathbf{i}+(87 \mathrm{lb}) \mathbf{j}+(72 \mathrm{lb}) \mathbf{k}
\end{aligned}
$$

Cable $A C$ :

$$
\begin{aligned}
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=T_{A C} \frac{(-48 \mathrm{in} .) \mathbf{i}+(25 \mathrm{in} .) \mathbf{j}+(-36 \mathrm{in} .) \mathbf{k}}{65 \mathrm{in} .} \\
& \mathbf{T}_{A C}=-\frac{48}{65} T_{A C} \mathbf{i}+\frac{25}{65} T_{A C} \mathbf{j}-\frac{36}{65} T_{A C} \mathbf{k}
\end{aligned}
$$

Load $P$ :

$$
\mathbf{P}=P \mathbf{j}
$$

For resultant to be directed along $O A$, i.e., $x$-axis

$$
R_{z}=0: \quad \Sigma F_{z}=(72 \mathrm{lb})-\frac{36}{65} T_{A C}^{\prime}=0 \quad T_{A C}=130.0 \mathrm{lb}
$$



## SOLUTION

See Problem 2.97. Since resultant must be directed along $O A$, i.e., the $x$-axis, we write

$$
R_{y}=0: \quad \Sigma F_{y}=(87 \mathrm{lb})+\frac{25}{65} T_{A C}-P=0
$$

$T_{A C}=130.0 \mathrm{lb}$ from Problem 2.97.

Then

$$
(87 \mathrm{lb})+\frac{25}{65}(130.0 \mathrm{lb})-P=0
$$

$$
P=137.0 \mathrm{lb}
$$



## PROBLEM 2.99

A container is supported by three cables that are attached to a ceiling as shown. Determine the weight $W$ of the container, knowing that the tension in cable $A B$ is 6 kN .

## SOLUTION

## Free-Body Diagram at A:



The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D}, \text { and } \mathbf{W}
$$

where $\mathbf{W}=W \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write

$$
\begin{array}{ll}
\overrightarrow{A B}=-(450 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j} & A B=750 \mathrm{~mm} \\
\overrightarrow{A C}=+(600 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} & A C=680 \mathrm{~mm} \\
\overrightarrow{A D}=+(500 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A D=860 \mathrm{~mm}
\end{array}
$$

$$
\mathbf{T}_{A B}=\lambda_{A B} T_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=T_{A B} \frac{(-450 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}}{750 \mathrm{~mm}}
$$

$$
=\left(-\frac{45}{75} \mathbf{i}+\frac{60}{75} \mathbf{j}\right) T_{A B}
$$

$$
\mathbf{T}_{A C}=\lambda_{A C} T_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=T_{A C} \frac{(600 \mathrm{~mm}) \mathbf{i}-(320 \mathrm{~mm}) \mathbf{j}}{680 \mathrm{~mm}}
$$

$$
=\left(\frac{60}{68} \mathbf{j}-\frac{32}{68} \mathbf{k}\right) T_{A C}
$$

$$
\mathbf{T}_{A D}=\lambda_{A D} T_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D}=T_{A D} \frac{(500 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k}}{860 \mathrm{~mm}}
$$

$$
=\left(\frac{50}{86} \mathbf{i}+\frac{60}{86} \mathbf{j}+\frac{36}{86} \mathbf{k}\right) T_{A D}
$$

## PROBLEM 2.99 (Continued)

Equilibrium condition: $\quad \Sigma F=0: \therefore \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+\mathbf{W}=0$
Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$; factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$; and equating each of the coefficients to zero gives the following equations:

From i:

$$
\begin{equation*}
-\frac{45}{75} T_{A B}+\frac{50}{86} T_{A D}=0 \tag{1}
\end{equation*}
$$

From $\mathbf{j}: \quad \frac{60}{75} T_{A B}+\frac{60}{68} T_{A C}+\frac{60}{86} T_{A D}-W=0$

From $\mathbf{k}$ :

$$
-\frac{32}{68} T_{A C}+\frac{36}{86} T_{A D}=0
$$

Setting $T_{A B}=6 \mathrm{kN}$ in (1) and (2), and solving the resulting set of equations gives

$$
\begin{aligned}
& T_{A C}=6.1920 \mathrm{kN} \\
& T_{A C}=5.5080 \mathrm{kN} \quad W=13.98 \mathrm{kN}
\end{aligned}
$$



## PROBLEM 2.100

A container is supported by three cables that are attached to a ceiling as shown. Determine the weight $W$ of the container, knowing that the tension in cable $A D$ is 4.3 kN .

## SOLUTION

See Problem 2.99 for the figure and analysis leading to the following set of linear algebraic equations:

$$
\begin{align*}
-\frac{45}{75} T_{A B}+\frac{50}{86} T_{A D} & =0  \tag{1}\\
\frac{60}{75} T_{A B}+\frac{60}{68} T_{A C}+\frac{60}{86} T_{A D}-W & =0  \tag{2}\\
-\frac{32}{68} T_{A C}+\frac{36}{86} T_{A D} & =0 \tag{3}
\end{align*}
$$

Setting $T_{A D}=4.3 \mathrm{kN}$ into the above equations gives

$$
\begin{aligned}
T_{A B} & =4.1667 \mathrm{kN} \\
T_{A C} & =3.8250 \mathrm{kN}
\end{aligned}
$$

$$
W=9.71 \mathrm{kN}
$$



## SOLUTION

## FREE-BODY DIAGRAM AT $\boldsymbol{A}$



The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D}, \text { and } \mathbf{P}
$$

where $\mathbf{P}=P \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write

$$
\begin{array}{lc}
\overrightarrow{A B}=-(4.20 \mathrm{~m}) \mathbf{i}-(5.60 \mathrm{~m}) \mathbf{j} & A B=7.00 \mathrm{~m} \\
\overrightarrow{A C}=(2.40 \mathrm{~m}) \mathbf{i}-(5.60 \mathrm{~m}) \mathbf{j}+(4.20 \mathrm{~m}) \mathbf{k} & A C=7.40 \mathrm{~m} \\
\overrightarrow{A D}=-(5.60 \mathrm{~m}) \mathbf{j}-(3.30 \mathrm{~m}) \mathbf{k} & A D=6.50 \mathrm{~m}
\end{array}
$$

and

$$
\begin{aligned}
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{\frac{A B}{A}}=(-0.6 \mathbf{i}-0.8 \mathbf{j}) T_{A B} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=(0.32432 \mathbf{i}-0.75676 \mathbf{j}+0.56757 \mathbf{k}) T_{A C} \\
& \mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D}=(-0.86154 \mathbf{j}-0.50769 \mathbf{k}) T_{A D}
\end{aligned}
$$

## PROBLEM 2.101 (Continued)

Equilibrium condition:

$$
\Sigma F=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{gathered}
\left(-0.6 T_{A B}+0.32432 T_{A C}\right) \mathbf{i}+\left(-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P\right) \mathbf{j} \\
+\left(0.56757 T_{A C}-0.50769 T_{A D}\right) \mathbf{k}=0
\end{gathered}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{align*}
-0.6 T_{A B}+0.32432 T_{A C} & =0  \tag{1}\\
-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P & =0  \tag{2}\\
0.56757 T_{A C}-0.50769 T_{A D} & =0 \tag{3}
\end{align*}
$$

Setting $T_{A D}=481 \mathrm{~N}$ in (2) and (3), and solving the resulting set of equations gives

$$
\begin{array}{ll}
T_{A C}=430.26 \mathrm{~N} & \\
T_{A D}=232.57 \mathrm{~N} & \mathbf{P}=926 \mathrm{~N} \uparrow
\end{array}
$$



## PROBLEM 2.102

Three cables are used to tether a balloon as shown. Knowing that the balloon exerts an $800-\mathrm{N}$ vertical force at $A$, determine the tension in each cable.

## SOLUTION

See Problem 2.101 for the figure and analysis leading to the linear algebraic Equations (1), (2), and (3).

$$
\begin{array}{r}
-0.6 T_{A B}+0.32432 T_{A C}=0 \\
-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P=0 \\
0.56757 T_{A C}-0.50769 T_{A D}=0 \tag{3}
\end{array}
$$

From Eq. (1):

$$
T_{A B}=0.54053 T_{A C}
$$

From Eq. (3):

$$
T_{A D}=1.11795 T_{A C}
$$

Substituting for $T_{A B}$ and $T_{A D}$ in terms of $T_{A C}$ into Eq. (2) gives

$$
\begin{aligned}
&-0.8\left(0.54053 T_{A C}\right)-0.75676 T_{A C}-0.86154\left(1.11795 T_{A C}\right)+P=0 \\
& 2.1523 T_{A C}=P ; \quad P=800 \mathrm{~N} \\
& T_{A C}=\frac{800 \mathrm{~N}}{2.1523} \\
&=371.69 \mathrm{~N}
\end{aligned}
$$

Substituting into expressions for $T_{A B}$ and $T_{A D}$ gives

$$
\begin{aligned}
& T_{A B}=0.54053(371.69 \mathrm{~N}) \\
& T_{A D}=1.11795(371.69 \mathrm{~N})
\end{aligned}
$$

$$
T_{A B}=201 \mathrm{~N}, \quad T_{A C}=372 \mathrm{~N}, \quad T_{A D}=416 \mathrm{~N}
$$



## SOLUTION

By Symmetry $T_{D B}=T_{D C}$

## Free-Body Diagram of Point $D$ :



The forces applied at $D$ are:

$$
\mathbf{T}_{D B}, \mathbf{T}_{D C}, \mathbf{T}_{D A}, \text { and } \mathbf{P}
$$

where $\mathbf{P}=P \mathbf{j}=(36 \mathrm{lb}) \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{array}{ll}
\overrightarrow{D A}=(16 \mathrm{in}) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j} & D A=28.844 \mathrm{in} . \\
\overrightarrow{D B}=-(8 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j}+(6 \mathrm{in} .) \mathbf{k} & D B=26.0 \mathrm{in} . \\
\overrightarrow{D C}=-(8 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j}-(6 \mathrm{in} .) \mathbf{k} & D C=26.0 \mathrm{in} . \\
\mathbf{T}_{D A}=T_{D A} \lambda_{D A}=T_{D A} \frac{\overrightarrow{D A}}{D A}=(0.55471 \mathbf{i}-0.83206 \mathbf{j}) T_{D A} \\
\mathbf{T}_{D B}=T_{D B} \lambda_{D B}=T_{D B} \frac{\overrightarrow{D B}}{D B}=(-0.30769 \mathbf{i}-0.92308 \mathbf{j}+0.23077 \mathbf{k}) T_{D B} \\
\mathbf{T}_{D C}=T_{D C} \lambda_{D C}=T_{D C} \frac{\overrightarrow{D C}}{D C}=(-0.30769 \mathbf{i}-0.92308 \mathbf{j}-0.23077 \mathbf{k}) T_{D C}
\end{array}
$$

## PROBLEM 2.103 (Continued)

Equilibrium condition: $\quad \Sigma F=0: \quad \mathbf{T}_{D A}+\mathbf{T}_{D B}+\mathbf{T}_{D C}+(36 \mathrm{lb}) \mathbf{j}=0$
Substituting the expressions obtained for $\mathbf{T}_{D A}, \mathbf{T}_{D B}$, and $\mathbf{T}_{D C}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{gathered}
\left(0.55471 T_{D A}-0.30769 T_{D B}-0.30769 T_{D C}\right) \mathbf{i}+\left(-0.83206 T_{D A}-0.92308 T_{D B}-0.92308 T_{D C}+36 \mathrm{lb}\right) \mathbf{j} \\
+ \\
+\left(0.23077 T_{D B}-0.23077 T_{D C}\right) \mathbf{k}=0
\end{gathered}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{array}{r}
0.55471 T_{D A}-0.30769 T_{D B}-0.30769 T_{D C}=0 \\
-0.83206 T_{D A}-0.92308 T_{D B}-0.92308 T_{D C}+36 \mathrm{lb}=0 \\
0.23077 T_{D B}-0.23077 T_{D C}=0 \tag{3}
\end{array}
$$

Equation (3) confirms that $T_{D B}=T_{D C}$. Solving simultaneously gives,

$$
T_{D A}=14.42 \mathrm{lb} ; \quad T_{D B}=T_{D C}=13.00 \mathrm{lb}
$$



## PROBLEM 2.104

Solve Prob. 2.103, assuming that $a=8$ in.
PROBLEM 2.103 A 36-lb triangular plate is supported by three wires as shown. Determine the tension in each wire, knowing that $a=6 \mathrm{in}$.

## SOLUTION

By Symmetry $T_{D B}=T_{D C}$

## Free-Body Diagram of Point $D$ :



The forces applied at $D$ are:

$$
\mathbf{T}_{D B}, \mathbf{T}_{D C}, \mathbf{T}_{D A} \text {, and } \mathbf{P}
$$

where $\mathbf{P}=P \mathbf{j}=(36 \mathrm{lb}) \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write

$$
\begin{array}{ll}
\overrightarrow{D A}=(16 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j} & D A=28.844 \mathrm{in} . \\
\overrightarrow{D B}=-(8 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j}+(8 \mathrm{in}) \mathbf{k} & D B=26.533 \mathrm{in} . \\
\overrightarrow{D C}=-(8 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j}-(8 \mathrm{in} .) \mathbf{k} & D C=26.533 \mathrm{in} .
\end{array}
$$

and

$$
\begin{aligned}
& \mathbf{T}_{D A}=T_{D A} \boldsymbol{\lambda}_{D A}=T_{D A} \frac{\overrightarrow{D A}}{D A}=(0.55471 \mathbf{i}-0.83206 \mathbf{j}) T_{D A} \\
& \mathbf{T}_{D B}=T_{D B} \lambda_{D B}=T_{D B} \frac{\overrightarrow{D B}}{\frac{D B}{D}}=(-0.30151 \mathbf{i}-0.90453 \mathbf{j}+0.30151 \mathbf{k}) T_{D B} \\
& \mathbf{T}_{D C}=T_{D C} \boldsymbol{\lambda}_{D C}=T_{D C} \frac{\overrightarrow{D C}}{D C}=(-0.30151 \mathbf{i}-0.90453 \mathbf{j}-0.30151 \mathbf{k}) T_{D C}
\end{aligned}
$$

## PROBLEM 2.104 (Continued)

Equilibrium condition: $\quad \quad \Sigma F=0: \quad \mathbf{T}_{D A}+\mathbf{T}_{D B}+\mathbf{T}_{D C}+(36 \mathrm{lb}) \mathbf{j}=0$
Substituting the expressions obtained for $\mathbf{T}_{D A}, \mathbf{T}_{D B}$, and $\mathbf{T}_{D C}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{gathered}
\left(0.55471 T_{D A}-0.30151 T_{D B}-0.30151 T_{D C}\right) \mathbf{i}+\left(-0.83206 T_{D A}-0.90453 T_{D B}-0.90453 T_{D C}+36 \mathrm{lb}\right) \mathbf{j} \\
+\left(0.30151 T_{D B}-0.30151 T_{D C}\right) \mathbf{k}=0
\end{gathered}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{array}{r}
0.55471 T_{D A}-0.30151 T_{D B}-0.30151 T_{D C}=0 \\
-0.83206 T_{D A}-0.90453 T_{D B}-0.90453 T_{D C}+36 \mathrm{lb}=0 \\
0.30151 T_{D B}-0.30151 T_{D C}=0 \tag{3}
\end{array}
$$

Equation (3) confirms that $T_{D B}=T_{D C}$. Solving simultaneously gives,

$$
T_{D A}=14.42 \mathrm{lb} ; \quad T_{D B}=T_{D C}=13.27 \mathrm{lb}
$$



## PROBLEM 2.105

A crate is supported by three cables as shown. Determine the weight of the crate knowing that the tension in cable $A C$ is 544 lb .

Solution The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D} \text { and } \mathbf{W}
$$

where $\mathbf{P}=P \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write

$$
\begin{aligned}
& \overrightarrow{A B}=-(36 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(27 \mathrm{in} .) \mathbf{k} \\
& A B=75 \mathrm{in} . \\
& \overrightarrow{A C}=(60 \mathrm{in} .) \mathbf{j}+(32 \mathrm{in} .) \mathbf{k} \\
& A C=68 \mathrm{in} . \\
& \overrightarrow{A D}=(40 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(27 \mathrm{in} .) \mathbf{k} \\
& A D=77 \mathrm{in} .
\end{aligned}
$$

and

$$
\begin{aligned}
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B} \\
& =(-0.48 \mathbf{i}+0.8 \mathbf{j}-0.36 \mathbf{k}) T_{A B} \\
\mathbf{T}_{A C} & =T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C} \\
& =(0.88235 \mathbf{j}+0.47059 \mathbf{k}) T_{A C} \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D} \\
& =(0.51948 \mathbf{i}+0.77922 \mathbf{j}-0.35065 \mathbf{k}) T_{A D}
\end{aligned}
$$



$$
\mathbf{W}=-W \mathbf{j}
$$

$$
\Sigma F=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}-W \mathbf{j}=0
$$

## PROBLEM 2.105 (Continued)

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{aligned}
\left(-0.48 T_{A B}\right. & \left.+0.51948 T_{A D}\right) \mathbf{i}+\left(0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W\right) \mathbf{j} \\
& +\left(-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D}\right) \mathbf{k}=0
\end{aligned}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{align*}
-0.48 T_{A B}+0.51948 T_{A D} & =0  \tag{1}\\
0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W & =0  \tag{2}\\
-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D} & =0 \tag{3}
\end{align*}
$$

Substituting $T_{A C}=544 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms, gives:

$$
\begin{aligned}
& T_{A B}=374.27 \mathrm{lb} \\
& T_{A D}=345.82 \mathrm{lb}
\end{aligned} \quad W=1049 \mathrm{lb}
$$



## PROBLEM 2.106

A $1600-1 \mathrm{lb}$ crate is supported by three cables as shown. Determine the tension in each cable.

## SOLUTION

The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D} \text { and } \mathbf{W}
$$

where $\mathbf{P}=P \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}$, $\mathbf{k}$, we write

$$
\begin{aligned}
& \overrightarrow{A B}=-(36 \mathrm{in}) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(27 \mathrm{in} .) \mathbf{k} \\
& A B=75 \mathrm{in} . \\
& \overrightarrow{A C}=(60 \mathrm{in} .) \mathbf{j}+(32 \mathrm{in} .) \mathbf{k} \\
& A C=68 \mathrm{in} . \\
& \overrightarrow{A D}=(40 \mathrm{in}) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(27 \mathrm{in} .) \mathbf{k} \\
& A D=77 \mathrm{in} .
\end{aligned}
$$

and

$$
\begin{aligned}
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B} \\
& =(-0.48 \mathbf{i}+0.8 \mathbf{j}-0.36 \mathbf{k}) T_{A B} \\
\mathbf{T}_{A C} & =T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C} \\
& =(0.88235 \mathbf{j}+0.47059 \mathbf{k}) T_{A C} \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D} \\
& =(0.51948 \mathbf{i}+0.77922 \mathbf{j}-0.35065 \mathbf{k}) T_{A D}
\end{aligned}
$$



$$
\mathbf{V}=-W \mathbf{j}
$$

$$
\Sigma F=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}-W \mathbf{j}=0
$$

## PROBLEM 2.106 (Continued)

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{aligned}
\left(-0.48 T_{A B}+\right. & \left.0.51948 T_{A D}\right) \mathbf{i}+\left(0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W\right) \mathbf{j} \\
& +\left(-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D}\right) \mathbf{k}=0
\end{aligned}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{align*}
-0.48 T_{A B}+0.51948 T_{A D} & =0  \tag{1}\\
0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W & =0  \tag{2}\\
-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D} & =0 \tag{3}
\end{align*}
$$

Substituting $W=1600 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms gives,

$$
\begin{aligned}
T_{A B} & =571 \mathrm{lb} \\
T_{A C} & =830 \mathrm{lb} \\
T_{A D} & =528 \mathrm{lb}
\end{aligned}
$$



## SOLUTION

$$
\begin{aligned}
& \Sigma \mathbf{F}_{A}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+\mathbf{P}=0 \quad \text { where } \quad \mathbf{P}=P \mathbf{i} \\
& \overrightarrow{A B}=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}+(380 \mathrm{~mm}) \mathbf{k} \quad A B=1060 \mathrm{~mm} \\
& \overrightarrow{A C}
\end{aligned}=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \quad A C=1040 \mathrm{~mm}, ~ \begin{aligned}
& \overrightarrow{A D}=-(960 \mathrm{~mm}) \mathbf{i}+(720 \mathrm{~mm}) \mathbf{j}-(220 \mathrm{~mm}) \mathbf{k} \quad A D=1220 \mathrm{~mm} \\
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=T_{A B}\left(-\frac{48}{53} \mathbf{i}-\frac{12}{53} \mathbf{j}+\frac{19}{53} \mathbf{k}\right) \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=T_{A C}\left(-\frac{12}{13} \mathbf{i}-\frac{3}{13} \mathbf{j}-\frac{4}{13} \mathbf{k}\right) \\
& \mathbf{T}_{A D}=T_{A D} \lambda_{A D}=\frac{305 \mathrm{~N}}{1220 \mathrm{~mm}}[(-960 \mathrm{~mm}) \mathbf{i}+(720 \mathrm{~mm}) \mathbf{j}-(220 \mathrm{~mm}) \mathbf{k}] \\
&=-(240 \mathrm{~N}) \mathbf{i}+(180 \mathrm{~N}) \mathbf{j}-(55 \mathrm{~N}) \mathbf{k}
\end{aligned}
$$

Substituting into $\Sigma \mathbf{F}_{A}=0$, factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$, and setting each coefficient equal to $\phi$ gives:

$$
\begin{align*}
& \mathbf{i}: \quad P=\frac{48}{53} T_{A B}+\frac{12}{13} T_{A C}+240 \mathrm{~N}  \tag{1}\\
& \mathbf{j}: \quad \frac{12}{53} T_{A B}+\frac{3}{13} T_{A C}=180 \mathrm{~N}  \tag{2}\\
& \text { k: } \quad \frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}=55 \mathrm{~N} \tag{3}
\end{align*}
$$

Solving the system of linear equations using conventional algorithms gives:

$$
\begin{aligned}
& T_{A B}=446.71 \mathrm{~N} \\
& T_{A C}=341.71 \mathrm{~N}
\end{aligned} P=960 \mathrm{~N}
$$



## PROBLEM 2.108

Three cables are connected at $A$, where the forces $\mathbf{P}$ and $\mathbf{Q}$ are applied as shown. Knowing that $P=1200 \mathrm{~N}$, determine the values of $Q$ for which cable $A D$ is taut.

## SOLUTION

We assume that $T_{A D}=0$ and write $\quad \Sigma \mathbf{F}_{A}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+Q \mathbf{j}+(1200 \mathrm{~N}) \mathbf{i}=0$

$$
\begin{aligned}
& \overrightarrow{A B}=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}+(380 \mathrm{~mm}) \mathbf{k} \quad A B=1060 \mathrm{~mm} \\
& \overrightarrow{A C}=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \quad A C=1040 \mathrm{~mm} \\
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=\left(-\frac{48}{53} \mathbf{i}-\frac{12}{53} \mathbf{j}+\frac{19}{53} \mathbf{k}\right) T_{A B} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=\left(-\frac{12}{13} \mathbf{i}-\frac{3}{13} \mathbf{j}-\frac{4}{13} \mathbf{k}\right) T_{A C}
\end{aligned}
$$

Substituting into $\Sigma \mathbf{F}_{A}=0$, factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$, and setting each coefficient equal to $\phi$ gives:

$$
\begin{align*}
& \mathbf{i}: \quad-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}+1200 \mathrm{~N}=0  \tag{1}\\
& \mathbf{j}: \quad-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+Q=0  \tag{2}\\
& \mathbf{k}: \quad \frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}=0 \tag{3}
\end{align*}
$$

Solving the resulting system of linear equations using conventional algorithms gives:

$$
\begin{aligned}
T_{A B} & =605.71 \mathrm{~N} \\
T_{A C} & =705.71 \mathrm{~N} \\
Q & =300.00 \mathrm{~N}
\end{aligned}
$$

$$
0 \leq Q<300 \mathrm{~N}
$$

Note: This solution assumes that $Q$ is directed upward as shown $(Q \geq 0)$, if negative values of $Q$ are considered, cable $A D$ remains taut, but $A C$ becomes slack for $Q=-460 \mathrm{~N}$.


## SOLUTION

We note that the weight of the plate is equal in magnitude to the force $\mathbf{P}$ exerted by the support on Point $A$.

Free Body $\boldsymbol{A}$ :

$$
\Sigma F=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0
$$

We have:

$$
\begin{array}{ll}
\overrightarrow{A B}=-(320 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A B=680 \mathrm{~mm} \\
\overrightarrow{A C}=(450 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A C=750 \mathrm{~mm} \\
\overrightarrow{A D}=(250 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}-(360 \mathrm{~mm}) \mathbf{k} & A D=650 \mathrm{~mm}
\end{array}
$$

Thus:


$$
\begin{aligned}
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=\left(-\frac{8}{17} \mathbf{i}-\frac{12}{17} \mathbf{j}+\frac{9}{17} \mathbf{k}\right) T_{A B} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=(0.6 \mathbf{i}-0.64 \mathbf{j}+0.48 \mathbf{k}) T_{A C} \\
& \mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D}=\left(\frac{5}{13} \mathbf{i}-\frac{9.6}{13} \mathbf{j}-\frac{7.2}{13} \mathbf{k}\right) T_{A D}
\end{aligned}
$$

Substituting into the Eq. $\Sigma F=0$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{aligned}
& \left(-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D}\right) \mathbf{i} \\
& +\left(-\frac{12}{17} T_{A B}-0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P\right) \mathbf{j} \\
& +\left(\frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D}\right) \mathbf{k}=0
\end{aligned}
$$

## PROBLEM 2.109 (Continued)

Setting the coefficient of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to zero:

$$
\begin{array}{ll}
\text { i: } & -\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D}=0 \\
\text { j: } & -\frac{12}{7} T_{A B}-0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P=0 \\
\text { k: } & \frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D}=0 \tag{3}
\end{array}
$$

Making $T_{A C}=60 \mathrm{~N}$ in (1) and (3):

$$
\begin{align*}
-\frac{8}{17} T_{A B}+36 \mathrm{~N}+\frac{5}{13} T_{A D} & =0 \\
\frac{9}{17} T_{A B}+28.8 \mathrm{~N}-\frac{7.2}{13} T_{A D} & =0 \tag{3'}
\end{align*}
$$

Multiply ( $1^{\prime}$ ) by $9,\left(3^{\prime}\right)$ by 8 , and add:

$$
554.4 \mathrm{~N}-\frac{12.6}{13} T_{A D}=0 \quad T_{A D}=572.0 \mathrm{~N}
$$

Substitute into ( $1^{\prime}$ ) and solve for $T_{A B}$ :

$$
T_{A B}=\frac{17}{8}\left(36+\frac{5}{13} \times 572\right) \quad T_{A B}=544.0 \mathrm{~N}
$$

Substitute for the tensions in Eq. (2) and solve for $P$ :

$$
P=\frac{12}{17}(544 \mathrm{~N})+0.64(60 \mathrm{~N})+\frac{9.6}{13}(572 \mathrm{~N})
$$

$$
=844.8 \mathrm{~N} \quad \text { Weight of plate }=P=845 \mathrm{~N}
$$



## SOLUTION

See Problem 2.109 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{align*}
-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D} & =0  \tag{1}\\
-\frac{12}{17} T_{A B}+0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P & =0  \tag{2}\\
\frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D} & =0 \tag{3}
\end{align*}
$$

Making $T_{A D}=520 \mathrm{~N}$ in Eqs. (1) and (3):

$$
\begin{align*}
& -\frac{8}{17} T_{A B}+0.6 T_{A C}+200 \mathrm{~N}=0 \\
& \frac{9}{17} T_{A B}+0.48 T_{A C}-288 \mathrm{~N}=0 \tag{3'}
\end{align*}
$$

Multiply ( $1^{\prime}$ ) by $9,\left(3^{\prime}\right)$ by 8 , and add:

$$
9.24 T_{A C}-504 \mathrm{~N}=0 \quad T_{A C}=54.5455 \mathrm{~N}
$$

Substitute into (1') and solve for $T_{A B}$ :

$$
T_{A B}=\frac{17}{8}(0.6 \times 54.5455+200) \quad T_{A B}=494.545 \mathrm{~N}
$$

Substitute for the tensions in Eq. (2) and solve for $P$ :

$$
P=\frac{12}{17}(494.545 \mathrm{~N})+0.64(54.5455 \mathrm{~N})+\frac{9.6}{13}(520 \mathrm{~N})
$$

$$
=768.00 \mathrm{~N} \quad \text { Weight of plate }=P=768 \mathrm{~N}
$$



## SOLUTION

$$
\Sigma \mathbf{F}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0 \quad \text { Free-Body Diagram at } A:
$$

$$
\begin{array}{lc}
\overrightarrow{A B}=-20 \mathbf{i}-100 \mathbf{j}+25 \mathbf{k} & A B=105 \mathrm{ft} \\
\overrightarrow{A C}=60 \mathbf{i}-100 \mathbf{j}+18 \mathbf{k} & A C=118 \mathrm{ft} \\
\overrightarrow{A D}=-20 \mathbf{i}-100 \mathbf{j}-74 \mathbf{k} & A D=126 \mathrm{ft}
\end{array}
$$



We write

$$
\begin{aligned}
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B} \\
& =\left(-\frac{4}{21} \mathbf{i}-\frac{20}{21} \mathbf{j}+\frac{5}{21} \mathbf{k}\right) T_{A B} \\
\mathbf{T}_{A C} & =T_{A C} \boldsymbol{\lambda}_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C} \\
& =\left(\frac{30}{59} \mathbf{i}-\frac{50}{59} \mathbf{j}+\frac{9}{59} \mathbf{k}\right) T_{A C} \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D} \\
& =\left(-\frac{10}{63} \mathbf{i}-\frac{50}{63} \mathbf{j}-\frac{37}{63} \mathbf{k}\right) T_{A D}
\end{aligned}
$$

Substituting into the Eq. $\Sigma \mathbf{F}=0$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

## PROBLEM 2.111 (Continued)

$$
\begin{aligned}
& \left(-\frac{4}{21} T_{A B}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D}\right) \mathbf{i} \\
& +\left(-\frac{20}{21} T_{A B}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+P\right) \mathbf{j} \\
& +\left(\frac{5}{21} T_{A B}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D}\right) \mathbf{k}=0
\end{aligned}
$$

Setting the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$, equal to zero:

$$
\begin{align*}
& \text { i: } \quad-\frac{4}{21} T_{A B}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D}=0  \tag{1}\\
& \mathbf{j}: \quad-\frac{20}{21} T_{A B}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+P=0  \tag{2}\\
& \mathbf{k}: \quad \frac{5}{21} T_{A B}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D}=0 \tag{3}
\end{align*}
$$

Set $T_{A B}=840 \mathrm{lb}$ in Eqs. (1) - (3):

$$
\begin{align*}
-160 \mathrm{lb}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D} & =0 \\
-800 \mathrm{lb}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+P & =0 \\
200 \mathrm{lb}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D} & =0 \tag{3'}
\end{align*}
$$

Solving, $\quad T_{A C}=458.12 \mathrm{lb} \quad T_{A D}=459.53 \mathrm{lb} \quad P=1552.94 \mathrm{lb}$ $P=1553 \mathrm{lb}$


## SOLUTION

$$
\Sigma \mathbf{F}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0 \quad \text { Free-Body Diagram at } A:
$$

$$
\begin{array}{ll}
\overrightarrow{A B}=-20 \mathbf{i}-100 \mathbf{j}+25 \mathbf{k} & A B=105 \mathrm{ft} \\
\overrightarrow{A C}=60 \mathbf{i}-100 \mathbf{j}+18 \mathbf{k} & A C=118 \mathrm{ft} \\
\overrightarrow{A D}=-20 \mathbf{i}-100 \mathbf{j}-74 \mathbf{k} & A D=126 \mathrm{ft}
\end{array}
$$



We write

$$
\begin{aligned}
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B} \\
& =\left(-\frac{4}{21} \mathbf{i}-\frac{20}{21} \mathbf{j}+\frac{5}{21} \mathbf{k}\right) T_{A B} \\
\mathbf{T}_{A C} & =T_{A C} \boldsymbol{\lambda}_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C} \\
& =\left(\frac{30}{59} \mathbf{i}-\frac{50}{59} \mathbf{j}+\frac{9}{59} \mathbf{k}\right) T_{A C} \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D} \\
& =\left(-\frac{10}{63} \mathbf{i}-\frac{50}{63} \mathbf{j}-\frac{37}{63} \mathbf{k}\right) T_{A D}
\end{aligned}
$$

Substituting into the Eq. $\Sigma \mathbf{F}=0$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

## PROBLEM 2.112 (Continued)

$$
\begin{aligned}
& \left(-\frac{4}{21} T_{A B}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D}\right) \mathbf{i} \\
& +\left(-\frac{20}{21} T_{A B}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+P\right) \mathbf{j} \\
& +\left(\frac{5}{21} T_{A B}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D}\right) \mathbf{k}=0
\end{aligned}
$$

Setting the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$, equal to zero:

$$
\begin{array}{ll}
\text { i: } & \quad-\frac{4}{21} T_{A B}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D}=0 \\
\text { j: } & -\frac{20}{21} T_{A B}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+P=0 \\
\text { k: } & \quad \frac{5}{21} T_{A B}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D}=0 \tag{3}
\end{array}
$$

Set $T_{A C}=590 \mathrm{lb}$ in Eqs. (1) - (3):

$$
\begin{align*}
-\frac{4}{21} T_{A B}+300 \mathrm{lb}-\frac{10}{63} T_{A D} & =0  \tag{1'}\\
-\frac{20}{21} T_{A B}-500 \mathrm{lb}-\frac{50}{63} T_{A D}+P & =0  \tag{2'}\\
\frac{5}{21} T_{A B}+90 \mathrm{lb}-\frac{37}{63} T_{A D} & =0 \tag{3'}
\end{align*}
$$

Solving,

$$
T_{A B}=1081.82 \mathrm{lb} \quad T_{A D}=591.82 \mathrm{lb}
$$

$$
P=2000 \mathrm{lb}
$$



## PROBLEM 2.113

In trying to move across a slippery icy surface, a $175-\mathrm{lb}$ man uses two ropes $A B$ and $A C$. Knowing that the force exerted on the man by the icy surface is perpendicular to that surface, determine the tension in each rope.

## SOLUTION

## Free-Body Diagram at $A$


$\mathbf{N}=N\left(\frac{16}{34} \mathbf{i}+\frac{30}{34} \mathbf{j}\right)$
and $\mathbf{W}=W \mathbf{j}=-(175 \mathrm{lb}) \mathbf{j}$

$$
\begin{aligned}
\mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C} & =T_{A C} \frac{(-30 \mathrm{ft}) \mathbf{i}+(20 \mathrm{ft}) \mathbf{j}-(12 \mathrm{ft}) \mathbf{k}}{38 \mathrm{ft}} \\
& =T_{A C}\left(-\frac{15}{19} \mathbf{i}+\frac{10}{19} \mathbf{j}-\frac{6}{19} \mathbf{k}\right) \\
\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B} & =T_{A B} \frac{(-30 \mathrm{ft}) \mathbf{i}+(24 \mathrm{ft}) \mathbf{j}+(32 \mathrm{ft}) \mathbf{k}}{50 \mathrm{ft}} \\
& =T_{A B}\left(-\frac{15}{25} \mathbf{i}+\frac{12}{25} \mathbf{j}+\frac{16}{25} \mathbf{k}\right)
\end{aligned}
$$

Equilibrium condition: $\Sigma \mathbf{F}=0$

$$
\mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{N}+\mathbf{W}=0
$$

## PROBLEM 2.113 (Continued)

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{N}$, and $\mathbf{W}$; factoring $\mathbf{i}$, $\mathbf{j}$, and $\mathbf{k}$; and equating each of the coefficients to zero gives the following equations:

From i: $\quad-\frac{15}{25} T_{A B}-\frac{15}{19} T_{A C}+\frac{16}{34} N=0$
From j: $\quad \frac{12}{25} T_{A B}+\frac{10}{19} T_{A C}+\frac{30}{34} N-(175 \mathrm{lb})=0$
From $\mathbf{k}$ :

$$
\begin{equation*}
\frac{16}{25} T_{A B}-\frac{6}{19} T_{A C}=0 \tag{2}
\end{equation*}
$$

Solving the resulting set of equations gives:

$$
T_{A B}=30.8 \mathrm{lb} ; T_{A C}=62.5 \mathrm{lb}
$$



## PROBLEM 2.114

Solve Problem 2.113, assuming that a friend is helping the man at $A$ by pulling on him with a force $\mathbf{P}=-(45 \mathrm{lb}) \mathbf{k}$.

PROBLEM 2.113 In trying to move across a slippery icy surface, a $175-\mathrm{lb}$ man uses two ropes $A B$ and $A C$. Knowing that the force exerted on the man by the icy surface is perpendicular to that surface, determine the tension in each rope.

## SOLUTION

Refer to Problem 2.113 for the figure and analysis leading to the following set of equations, Equation (3) being modified to include the additional force $\mathbf{P}=(-45 \mathrm{lb}) \mathbf{k}$.

$$
\begin{align*}
-\frac{15}{25} T_{A B}-\frac{15}{19} T_{A C}+\frac{16}{34} N & =0  \tag{1}\\
\frac{12}{25} T_{A B}+\frac{10}{19} T_{A C}+\frac{30}{34} N-(175 \mathrm{lb}) & =0  \tag{2}\\
\frac{16}{25} T_{A B}-\frac{6}{19} T_{A C}-(45 \mathrm{lb}) & =0 \tag{3}
\end{align*}
$$

Solving the resulting set of equations simultaneously gives:

$$
\begin{aligned}
T_{A B} & =81.3 \mathrm{lb} \\
T_{A C} & =22.2 \mathrm{lb}
\end{aligned}
$$



## SOLUTION

See Problem 2.109 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below. Setting $P=792$ N gives:

$$
\begin{align*}
-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D} & =0  \tag{1}\\
-\frac{12}{17} T_{A B}-0.64 T_{A C}-\frac{9.6}{13} T_{A D}+792 \mathrm{~N} & =0  \tag{2}\\
\frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D} & =0 \tag{3}
\end{align*}
$$

Solving Equations (1), (2), and (3) by conventional algorithms gives

$$
\begin{array}{lc}
T_{A B}=510.00 \mathrm{~N} & T_{A B}=510 \mathrm{~N} \\
T_{A C}=56.250 \mathrm{~N} & T_{A C}=56.2 \mathrm{~N} \\
T_{A D}=536.25 \mathrm{~N} & T_{A D}=536 \mathrm{~N}
\end{array}
$$



## SOLUTION

$$
\Sigma \mathbf{F}_{A}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+\mathbf{P}+\mathbf{Q}=0
$$

Where

$$
\begin{aligned}
\mathbf{P} & =P \mathbf{i} \text { and } \mathbf{Q}=Q \mathbf{j} \\
\overrightarrow{A B} & =-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}+(380 \mathrm{~mm}) \mathbf{k} \quad A B=1060 \mathrm{~mm} \\
\overrightarrow{A C} & =-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \quad A C=1040 \mathrm{~mm} \\
\overrightarrow{A D} & =-(960 \mathrm{~mm}) \mathbf{i}+(720 \mathrm{~mm}) \mathbf{j}-(220 \mathrm{~mm}) \mathbf{k} \quad A D=1220 \mathrm{~mm} \\
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=T_{A B}\left(-\frac{48}{53} \mathbf{i}-\frac{12}{53} \mathbf{j}+\frac{19}{53} \mathbf{k}\right) \\
\mathbf{T}_{A C} & =T_{A C} \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=T_{A C}\left(-\frac{12}{13} \mathbf{i}-\frac{3}{13} \mathbf{j}-\frac{4}{13} \mathbf{k}\right) \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D}=T_{A D}\left(-\frac{48}{61} \mathbf{i}+\frac{36}{61} \mathbf{j}-\frac{11}{61} \mathbf{k}\right)
\end{aligned}
$$

Substituting into $\Sigma \mathbf{F}_{A}=0$, setting $P=(2880 \mathrm{~N}) \mathbf{i}$ and $Q=0$, and setting the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to 0 , we obtain the following three equilibrium equations:

$$
\begin{align*}
& \mathbf{i}: \quad-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}-\frac{48}{61} T_{A D}+2880 \mathrm{~N}=0  \tag{1}\\
& \mathbf{j}: \quad-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+\frac{36}{61} T_{A D}=0  \tag{2}\\
& \mathbf{k}: \quad \frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}-\frac{11}{61} T_{A D}=0 \tag{3}
\end{align*}
$$

## PROBLEM 2．116（Continued）

Solving the system of linear equations using conventional algorithms gives：

$$
\begin{aligned}
& T_{A B}=1340.14 \mathrm{~N} \\
& T_{A C}=1025.12 \mathrm{~N} \\
& T_{A D}=915.03 \mathrm{~N}
\end{aligned}
$$

$$
\begin{gathered}
T_{A B}=1340 \mathrm{~N} \text { 《 } \\
T_{A C}=1025 \mathrm{~N} \text { 《 } \\
T_{A D}=915 \mathrm{~N} \text { 乙 }
\end{gathered}
$$



## SOLUTION

See Problem 2.116 for the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{align*}
-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}-\frac{48}{61} T_{A D}+P & =0  \tag{1}\\
-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+\frac{36}{61} T_{A D}+Q & =0  \tag{2}\\
\frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}-\frac{11}{61} T_{A D} & =0 \tag{3}
\end{align*}
$$

Setting $P=2880 \mathrm{~N}$ and $Q=576 \mathrm{~N}$ gives:

$$
\begin{align*}
-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}-\frac{48}{61} T_{A D}+2880 \mathrm{~N} & =0  \tag{1'}\\
-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+\frac{36}{61} T_{A D}+576 \mathrm{~N} & =0  \tag{2'}\\
\frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}-\frac{11}{61} T_{A D} & =0 \tag{3'}
\end{align*}
$$

Solving the resulting set of equations using conventional algorithms gives:

$$
\begin{aligned}
& T_{A B}=1431.00 \mathrm{~N} \\
& T_{A C}=1560.00 \mathrm{~N} \\
& T_{A D}=183.010 \mathrm{~N}
\end{aligned}
$$

$$
T_{A B}=1431 \mathrm{~N}
$$

$$
T_{A C}=1560 \mathrm{~N}
$$

$$
T_{A D}=183.0 \mathrm{~N}
$$



## PROBLEM 2.118

For the cable system of Problems 2.107 and 2.108, determine the tension in each cable knowing that $P=2880 \mathrm{~N}$ and $Q=-576 \mathrm{~N}$. ( $\mathbf{Q}$ is directed downward).

## SOLUTION

See Problem 2.116 for the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{align*}
-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}-\frac{48}{61} T_{A D}+P & =0  \tag{1}\\
-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+\frac{36}{61} T_{A D}+Q & =0  \tag{2}\\
\frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}-\frac{11}{61} T_{A D} & =0 \tag{3}
\end{align*}
$$

Setting $P=2880 \mathrm{~N}$ and $Q=-576 \mathrm{~N}$ gives:

$$
\begin{align*}
-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}-\frac{48}{61} T_{A D}+2880 \mathrm{~N} & =0 \\
-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+ & \frac{36}{61} T_{A D}-576 \mathrm{~N}
\end{align*}=0
$$

Solving the resulting set of equations using conventional algorithms gives:

$$
\begin{aligned}
& T_{A B}=1249.29 \mathrm{~N} \\
& T_{A C}=490.31 \mathrm{~N} \\
& T_{A D}=1646.97 \mathrm{~N}
\end{aligned}
$$

$$
\begin{gathered}
T_{A B}=1249 \mathrm{~N} \\
T_{A C}=490 \mathrm{~N} \\
T_{A D}=1647 \mathrm{~N}
\end{gathered}
$$



## PROBLEM 2.119

For the transmission tower of Probs. 2.111 and 2.112, determine the tension in each guy wire knowing that the tower exerts on the pin at $A$ an upward vertical force of 1800 lb .

PROBLEM 2.111 A transmission tower is held by three guy wires attached to a pin at $A$ and anchored by bolts at $B, C$, and $D$. If the tension in wire $A B$ is 840 lb , determine the vertical force $\mathbf{P}$ exerted by the tower on the pin at $A$.

## SOLUTION

See Problem 2.111 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{align*}
& \mathbf{i}: \quad-\frac{4}{21} T_{A B}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D}=0  \tag{1}\\
& \mathbf{j}: \quad-\frac{20}{21} T_{A B}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+P=0  \tag{2}\\
& \mathbf{k}: \quad \frac{5}{21} T_{A B}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D}=0 \tag{3}
\end{align*}
$$

Substituting for $P=1800 \mathrm{lb}$ in Equations (1), (2), and (3) above and solving the resulting set of equations using conventional algorithms gives:

$$
\begin{align*}
-\frac{4}{21} T_{A B}+\frac{30}{59} T_{A C}-\frac{10}{63} T_{A D} & =0 \\
-\frac{20}{21} T_{A B}-\frac{50}{59} T_{A C}-\frac{50}{63} T_{A D}+1800 \mathrm{lb} & =0  \tag{2'}\\
\frac{5}{21} T_{A B}+\frac{9}{59} T_{A C}-\frac{37}{63} T_{A D} & =0  \tag{3'}\\
T_{A B} & =973.64 \mathrm{lb} \\
T_{A C} & =531.00 \mathrm{lb} \\
T_{A D} & =532.64 \mathrm{lb}
\end{align*}
$$

$T_{A B}=974 \mathrm{lb}$

$$
T_{A C}=531 \mathrm{lb}
$$

$$
T_{A D}=533 \mathrm{lb}
$$



## PROBLEM 2.120

Three wires are connected at point $D$, which is located 18 in. below the T-shaped pipe support $A B C$. Determine the tension in each wire when a $180-\mathrm{lb}$ cylinder is suspended from point $D$ as shown.

## SOLUTION

## Free-Body Diagram of Point $D$ :



The forces applied at $D$ are:

$$
\mathbf{T}_{D A}, \mathbf{T}_{D B}, \mathbf{T}_{D C} \text { and } \mathbf{W}
$$

where $\mathbf{W}=-180.0 \mathrm{lbj}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write

$$
\begin{aligned}
& \overrightarrow{D A}=(18 \mathrm{in} .) \mathbf{j}+(22 \mathrm{in} .) \mathbf{k} \\
& D A=28.425 \mathrm{in} . \\
& \overrightarrow{D B}=-(24 \mathrm{in} .) \mathbf{i}+(18 \mathrm{in} .) \mathbf{j}-(16 \mathrm{in} .) \mathbf{k} \\
& D B=34.0 \mathrm{in} . \\
& \overrightarrow{D C}=(24 \mathrm{in} .) \mathbf{i}+(18 \mathrm{in} .) \mathbf{j}-(16 \mathrm{in} .) \mathbf{k} \\
& D C=34.0 \text { in. }
\end{aligned}
$$

## PROBLEM 2.120 (Continued)

and

$$
\begin{aligned}
\mathbf{T}_{D A} & =T_{D a} \lambda_{D A}=T_{D a} \frac{\overrightarrow{D A}}{D A} \\
& =(0.63324 \mathbf{j}+0.77397 \mathbf{k}) T_{D A} \\
\mathbf{T}_{D B} & =T_{D B} \lambda_{D B}=T_{D B} \frac{\overrightarrow{D B}}{D B} \\
& =(-0.70588 \mathbf{i}+0.52941 \mathbf{j}-0.47059 \mathbf{k}) T_{D B} \\
\mathbf{T}_{D C} & =T_{D C} \lambda_{D C}=T_{D C} \frac{\overrightarrow{D C}}{D C} \\
& =(0.70588 \mathbf{i}+0.52941 \mathbf{j}-0.47059 \mathbf{k}) T_{D C}
\end{aligned}
$$

Equilibrium Condition with $\quad \mathbf{W}=-W \mathbf{j}$

$$
\Sigma F=0: \quad \mathbf{T}_{D A}+\mathbf{T}_{D B}+\mathbf{T}_{D C}-W \mathbf{j}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{D A}, \mathbf{T}_{D B}$, and $\mathbf{T}_{D C}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{array}{r}
\left(-0.70588 T_{D B}+0.70588 T_{D C}\right) \mathbf{i} \\
\left(0.63324 T_{D A}+0.52941 T_{D B}+0.52941 T_{D C}-W\right) \mathbf{j} \\
\left(0.77397 T_{D A}-0.47059 T_{D B}-0.47059 T_{D C}\right) \mathbf{k}
\end{array}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{array}{r}
-0.70588 T_{D B}+0.70588 T_{D C}=0 \\
0.63324 T_{D A}+0.52941 T_{D B}+0.52941 T_{D C}-W=0 \\
0.77397 T_{D A}-0.47059 T_{D B}-0.47059 T_{D C}=0 \tag{3}
\end{array}
$$

Substituting $W=180 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms gives,

$$
\begin{gathered}
T_{D A}=119.7 \mathrm{lb} \\
T_{D B}=98.4 \mathrm{lb} \\
T_{D C}=98.4 \mathrm{lb}
\end{gathered}
$$



## SOLUTION

The (vector) force in each cable can be written as the product of the (scalar) force and the unit vector along the cable. That is, with

$$
\begin{aligned}
\overrightarrow{A B} & =-(0.78 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0 \mathrm{~m}) \mathbf{k} \\
A B & =\sqrt{(-0.78 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(0)^{2}} \\
& =1.78 \mathrm{~m} \\
\mathbf{T}_{A B} & =T \lambda_{A B}=T_{A B} \frac{\overrightarrow{A B}}{A B} \\
& =\frac{T_{A B}}{1.78 \mathrm{~m}}[-(0.78 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A B} & =T_{A B}(-0.4382 \mathbf{i}+0.8989 \mathbf{j}+0 \mathbf{k})
\end{aligned}
$$

and
and

$$
\begin{aligned}
\overrightarrow{A C} & =(0) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(1.2 \mathrm{~m}) \mathbf{k} \\
A C & =\sqrt{(0 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(1.2 \mathrm{~m})^{2}}=2 \mathrm{~m} \\
\mathbf{T}_{A C} & =T \boldsymbol{\lambda}_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}=\frac{T_{A C}}{2 \mathrm{~m}}[(0) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(1.2 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A C} & =T_{A C}(0.8 \mathbf{j}+0.6 \mathbf{k})
\end{aligned}
$$

$$
\begin{aligned}
& \overrightarrow{A D}=(1.3 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0.4 \mathrm{~m}) \mathbf{k} \\
& A D=\sqrt{(1.3 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(0.4 \mathrm{~m})^{2}}=2.1 \mathrm{~m} \\
& \mathbf{T}_{A D}=T \lambda_{A D}=T_{A D} \frac{\overrightarrow{A D}}{A D}=\frac{T_{A D}}{2.1 \mathrm{~m}}[(1.3 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0.4 \mathrm{~m}) \mathbf{k}] \\
& \mathbf{T}_{A D}=T_{A D}(0.6190 \mathbf{i}+0.7619 \mathbf{j}+0.1905 \mathbf{k})
\end{aligned}
$$

## PROBLEM 2.121 (Continued)

Finally,

$$
\begin{aligned}
\overrightarrow{A E} & =-(0.4 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}-(0.86 \mathrm{~m}) \mathbf{k} \\
A E & =\sqrt{(-0.4 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(-0.86 \mathrm{~m})^{2}}=1.86 \mathrm{~m} \\
\mathbf{T}_{A E} & =T \lambda_{A E}=T_{A E} \frac{\overrightarrow{A E}}{A E} \\
& =\frac{T_{A E}}{1.86 \mathrm{~m}}[-(0.4 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}-(0.86 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A E} & =T_{A E}(-0.2151 \mathbf{i}+0.8602 \mathbf{j}-0.4624 \mathbf{k})
\end{aligned}
$$

With the weight of the container

$$
\mathbf{W}=-W \mathbf{j}, \text { at } A \text { we have: }
$$

$$
\Sigma \mathbf{F}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}-W \mathbf{j}=0
$$

Equating the factors of $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ to zero, we obtain the following linear algebraic equations:

$$
\begin{align*}
-0.4382 T_{A B}+0.6190 T_{A D}-0.2151 T_{A E} & =0  \tag{1}\\
0.8989 T_{A B}+0.8 T_{A C}+0.7619 T_{A D}+0.8602 T_{A E}-W & =0  \tag{2}\\
0.6 T_{A C}+0.1905 T_{A D}-0.4624 T_{A E} & =0 \tag{3}
\end{align*}
$$

Knowing that $W=1000 \mathrm{~N}$ and that because of the pulley system at $B T_{A B}=T_{A D}=P$, where $P$ is the externally applied (unknown) force, we can solve the system of linear Equations (1), (2) and (3) uniquely for $P$.

$$
P=378 \mathrm{~N}
$$



## SOLUTION

Here, as in Problem 2.121, the support of the container consists of the four cables $A E, A C, A D$, and $A B$, with the condition that the force in cables $A B$ and $A D$ is equal to the externally applied force $P$. Thus, with the condition

$$
T_{A B}=T_{A D}=P
$$

and using the linear algebraic equations of Problem 2.131 with $T_{A C}=150 \mathrm{~N}$, we obtain
(a) $\quad P=454 \mathrm{~N}$
(b) $\quad W=1202 \mathrm{~N}$


## PROBLEM 2.123

Cable $B A C$ passes through a frictionless ring $A$ and is attached to fixed supports at $B$ and $C$, while cables $A D$ and $A E$ are both tied to the ring and are attached, respectively, to supports at $D$ and $E$. Knowing that a $200-\mathrm{lb}$ vertical load $\mathbf{P}$ is applied to ring $A$, determine the tension in each of the three cables.

## SOLUTION

## Free Body Diagram at $A$ :

Since $T_{B A C}=$ tension in cable $B A C$, it follows that

$$
I_{A C}
$$

$$
\begin{gathered}
T_{A B}=T_{A C}=T_{B A C} \\
\mathbf{T}_{A B}=T_{B A C} \boldsymbol{\lambda}_{A B}=T_{B A C} \frac{(-17.5 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}}{62.5 \mathrm{in} .}=T_{B A C}\left(\frac{-17.5}{62.5} \mathbf{i}+\frac{60}{62.5} \mathbf{j}\right) \\
\mathbf{T}_{A C}=T_{B A C} \boldsymbol{\lambda}_{A C}=T_{B A C} \frac{(60 \mathrm{in} .) \mathbf{i}+(25 \mathrm{in} .) \mathbf{k}}{65 \mathrm{in} .}=T_{B A C}\left(\frac{60}{65} \mathbf{j}+\frac{25}{65} \mathbf{k}\right) \\
\mathbf{T}_{A D}=T_{A D} \boldsymbol{\lambda}_{A D}=T_{A D} \frac{(80 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}}{100 \mathrm{in} .}=T_{A D}\left(\frac{4}{5} \mathbf{i}+\frac{3}{5} \mathbf{j}\right) \\
\mathbf{T}_{A E}=T_{A E} \boldsymbol{\lambda}_{A E}=T_{A E} \frac{(60 \mathrm{in} .) \mathbf{j}-(45 \mathrm{in} .) \mathbf{k}}{75 \mathrm{in} .}=T_{A E}\left(\frac{4}{5} \mathbf{j}-\frac{3}{5} \mathbf{k}\right)
\end{gathered}
$$

## PROBLEM 2.123 (Continued)

Substituting into $\Sigma \mathbf{F}_{A}=0$, setting $\mathbf{P}=(-200 \mathrm{lb}) \mathbf{j}$, and setting the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to $\phi$, we obtain the following three equilibrium equations:

From i: $-\frac{17.5}{62.5} T_{B A C}+\frac{4}{5} T_{A D}=0$
From
j: $\left(\frac{60}{62.5}+\frac{60}{65}\right) T_{B A C}+\frac{3}{5} T_{A D}+\frac{4}{5} T_{A E}-200 \mathrm{lb}=0$
From
$\mathbf{k}: \frac{25}{65} T_{B A C}-\frac{3}{5} T_{A E}=0$
Solving the system of linear equations using conventional algorithms gives:

$$
T_{B A C}=76.7 \mathrm{lb} ; T_{A D}=26.9 \mathrm{lb} ; T_{A E}=49.2 \mathrm{lb}
$$



## PROBLEM 2.124

Knowing that the tension in cable $A E$ of Prob. 2.123 is 75 lb , determine $(a)$ the magnitude of the load $\mathbf{P},(b)$ the tension in cables $B A C$ and $A D$.

PROBLEM 2.123 Cable $B A C$ passes through a frictionless ring $A$ and is attached to fixed supports at $B$ and $C$, while cables $A D$ and $A E$ are both tied to the ring and are attached, respectively, to supports at $D$ and $E$. Knowing that a $200-\mathrm{lb}$ vertical load $\mathbf{P}$ is applied to ring $A$, determine the tension in each of the three cables.

## SOLUTION

Refer to the solution to Problem 2.123 for the figure and analysis leading to the following set of equilibrium equations, Equation (2) being modified to include $P \mathbf{j}$ as an unknown quantity:

$$
\begin{align*}
& -\frac{17.5}{62.5} T_{B A C}+\frac{4}{5} T_{A D}=0 \\
& \left(\frac{60}{62.5}+\frac{60}{65}\right) T_{B A C}+\frac{3}{5} T_{A D}+\frac{4}{5} T_{A E}-P=0 \\
& \frac{25}{65} T_{B A C}-\frac{3}{5} T_{A E}=0 \tag{3}
\end{align*}
$$

Substituting for $T_{A E}=75 \mathrm{lb}$ and solving simultaneously gives:
(a) $P=305 \mathrm{lb}$
(b) $\quad T_{B A C}=117.0 \mathrm{lb} ; T_{A D}=40.9 \mathrm{lb}$


## PROBLEM 2.125

Collars $A$ and $B$ are connected by a $525-\mathrm{mm}-l o n g$ wire and can slide freely on frictionless rods. If a force $\mathbf{P}=(341 \mathrm{~N}) \mathbf{j}$ is applied to collar $A$, determine $(a)$ the tension in the wire when $y=155 \mathrm{~mm}$, $(b)$ the magnitude of the force $\mathbf{Q}$ required to maintain the equilibrium of the system.

## SOLUTION

For both Problems 2.125 and 2.126:

$$
(A B)^{2}=x^{2}+y^{2}+z^{2}
$$

Here

$$
(0.525 \mathrm{~m})^{2}=(0.20 \mathrm{~m})^{2}+y^{2}+z^{2}
$$

or

$$
y^{2}+z^{2}=0.23563 \mathrm{~m}^{2}
$$

Thus, when $y$ given, $z$ is determined,
Free-Body Diagrams of Collars:

Now

$$
\begin{aligned}
\lambda_{A B} & =\frac{\overrightarrow{A B}}{A B} \\
& =\frac{1}{0.525 \mathrm{~m}}(0.20 \mathbf{i}-y \mathbf{j}+z \mathbf{k}) \mathrm{m} \\
& =0.38095 \mathbf{i}-1.90476 y \mathbf{j}+1.90476 z \mathbf{k}
\end{aligned}
$$



Where $y$ and $z$ are in units of meters, m .
From the F.B. Diagram of collar $A: \quad \Sigma \mathbf{F}=0: \quad N_{x} \mathbf{i}+N_{z} \mathbf{k}+P \mathbf{j}+T_{A B} \lambda_{A B}=0$
Setting the $\mathbf{j}$ coefficient to zero gives

$$
P-(1.90476 y) T_{A B}=0
$$

With

$$
P=341 \mathrm{~N}
$$

$$
T_{A B}=\frac{341 \mathrm{~N}}{1.90476 y}
$$

Now, from the free body diagram of collar $B$ :

$$
\Sigma \mathbf{F}=0: \quad N_{x} \mathbf{i}+N_{y} \mathbf{j}+Q \mathbf{k}-T_{A B} \lambda_{A B}=0
$$

Setting the $\mathbf{k}$ coefficient to zero gives

$$
Q-T_{A B}(1.90476 z)=0
$$

And using the above result for $T_{A B}$, we have

$$
Q=T_{A B} z=\frac{341 \mathrm{~N}}{(1.90476) y}(1.90476 z)=\frac{(341 \mathrm{~N})(z)}{y}
$$

## PROBLEM 2.125 (Continued)

Then from the specifications of the problem, $y=155 \mathrm{~mm}=0.155 \mathrm{~m}$

$$
\begin{aligned}
z^{2} & =0.23563 \mathrm{~m}^{2}-(0.155 \mathrm{~m})^{2} \\
z & =0.46 \mathrm{~m}
\end{aligned}
$$

and
(a)

$$
\begin{aligned}
T_{A B} & =\frac{341 \mathrm{~N}}{0.155(1.90476)} \\
& =1155.00 \mathrm{~N}
\end{aligned}
$$

or
and
(b)

$$
\begin{aligned}
Q & =\frac{341 \mathrm{~N}(0.46 \mathrm{~m})(0.866)}{(0.155 \mathrm{~m})} \\
& =(1012.00 \mathrm{~N})
\end{aligned}
$$

or

$$
T_{A B}=1155 \mathrm{~N}
$$



## PROBLEM 2.126

Solve Problem 2.125 assuming that $y=275 \mathrm{~mm}$.
PROBLEM 2.125 Collars $A$ and $B$ are connected by a $525-\mathrm{mm}$-long wire and can slide freely on frictionless rods. If a force $\mathbf{P}=(341 \mathrm{~N}) \mathbf{j}$ is applied to collar $A$, determine (a) the tension in the wire when $y=155 \mathrm{~mm}$, (b) the magnitude of the force $\mathbf{Q}$ required to maintain the equilibrium of the system.

## SOLUTION

From the analysis of Problem 2.125, particularly the results:

$$
\begin{aligned}
y^{2}+z^{2} & =0.23563 \mathrm{~m}^{2} \\
T_{A B} & =\frac{341 \mathrm{~N}}{1.90476 y} \\
Q & =\frac{341 \mathrm{~N}}{y} z
\end{aligned}
$$

With $y=275 \mathrm{~mm}=0.275 \mathrm{~m}$, we obtain:

$$
\begin{aligned}
z^{2} & =0.23563 \mathrm{~m}^{2}-(0.275 \mathrm{~m})^{2} \\
z & =0.40 \mathrm{~m}
\end{aligned}
$$

and
(a)

$$
T_{A B}=\frac{341 \mathrm{~N}}{(1.90476)(0.275 \mathrm{~m})}=651.00
$$

or

$$
T_{A B}=651 \mathrm{~N}
$$

and
(b)

$$
Q=\frac{341 \mathrm{~N}(0.40 \mathrm{~m})}{(0.275 \mathrm{~m})}
$$

or $Q=496 \mathrm{~N}$


## SOLUTION

Using the force triangle and the laws of cosines and sines, we have

$$
\begin{aligned}
\gamma & =180^{\circ}-\left(40^{\circ}+20^{\circ}\right) \\
& =120^{\circ}
\end{aligned}
$$

Then

$$
\begin{aligned}
R^{2}= & (15 \mathrm{kN})^{2}+(10 \mathrm{kN})^{2} \\
& -2(15 \mathrm{kN})(10 \mathrm{kN}) \cos 120^{\circ} \\
= & 475 \mathrm{kN}^{2} \\
R= & 21.794 \mathrm{kN}
\end{aligned}
$$


and

$$
\begin{aligned}
\frac{10 \mathrm{kN}}{\sin \alpha} & =\frac{21.794 \mathrm{kN}}{\sin 120^{\circ}} \\
\sin \alpha & =\left(\frac{10 \mathrm{kN}}{21.794 \mathrm{kN}}\right) \sin 120^{\circ} \\
& =0.39737 \\
\alpha & =23.414
\end{aligned}
$$

Hence:

$$
\phi=\alpha+50^{\circ}=73.414
$$

$$
\mathbf{R}=21.8 \mathrm{kN} \nabla .73 .4^{\circ}
$$



## PROBLEM 2.128

Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

Compute the following distances:

$$
\begin{aligned}
O A & =\sqrt{(24 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}} \\
& =51.0 \mathrm{in} . \\
O B & =\sqrt{(28 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}} \\
& =53.0 \mathrm{in} . \\
O C & =\sqrt{(40 \mathrm{in} .)^{2}+(30 \mathrm{in} .)^{2}} \\
& =50.0 \mathrm{in} .
\end{aligned}
$$

102-lb Force:
$F_{x}=-102 \mathrm{lb} \frac{24 \mathrm{in} .}{51.0 \mathrm{in} .}$

$F_{y}=+102 \mathrm{lb} \frac{45 \mathrm{in} .}{51.0 \mathrm{in} .}$

$$
F_{x}=-48.0 \mathrm{lb}
$$

$$
F_{y}=+90.0 \mathrm{lb}
$$

106-lb Force:
$F_{x}=+106 \mathrm{lb} \frac{28 \mathrm{in} .}{53.0 \mathrm{in} .}$
$F_{x}=+56.0 \mathrm{lb}$
$F_{y}=+106 \mathrm{lb} \frac{45 \mathrm{in} .}{53.0 \mathrm{in} .}$ $F_{y}=+90.0 \mathrm{lb}$

200-lb Force:
$F_{x}=-200 \mathrm{lb} \frac{40 \mathrm{in} .}{50.0 \mathrm{in} .}$
$F_{x}=-160.0 \mathrm{lb}$
$F_{y}=-200 \mathrm{lb} \frac{30 \mathrm{in} .}{50.0 \mathrm{in} .}$
$F_{y}=-120.0 \mathrm{lb}$


## PROBLEM 2.129

A hoist trolley is subjected to the three forces shown. Knowing that $\alpha=40^{\circ}$, determine $(a)$ the required magnitude of the force $\mathbf{P}$ if the resultant of the three forces is to be vertical, (b) the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
R_{x} & =\xrightarrow{+} \Sigma F_{x}=P+(200 \mathrm{lb}) \sin 40^{\circ}-(400 \mathrm{lb}) \cos 40^{\circ} \\
R_{x} & =P-177.860 \mathrm{lb}  \tag{1}\\
R_{y} & =+\searrow \Sigma F_{y}=(200 \mathrm{lb}) \cos 40^{\circ}+(400 \mathrm{lb}) \sin 40^{\circ} \\
R_{y} & =410.32 \mathrm{lb} \tag{2}
\end{align*}
$$

(a) For $\mathbf{R}$ to be vertical, we must have $R_{x}=0$.

Set
$R_{x}=0$ in Eq. (1)

$$
\begin{aligned}
0 & =P-177.860 \mathrm{lb} \\
P & =177.860 \mathrm{lb}
\end{aligned}
$$

$$
P=177.9 \mathrm{lb}
$$

(b) Since $\mathbf{R}$ is to be vertical:

$$
R=R_{y}=410 \mathrm{lb} \quad R=410 \mathrm{lb}
$$



## SOLUTION

## Free-Body Diagram



## Force Triangle



Law of sines:

$$
\frac{F_{A C}}{\sin 35^{\circ}}=\frac{T_{B C}}{\sin 50^{\circ}}=\frac{300 \mathrm{lb}}{\sin 95^{\circ}}
$$

(a)

$$
F_{A C}=\frac{300 \mathrm{lb}}{\sin 95^{\circ}} \sin 35^{\circ}
$$

$$
F_{A C}=172.7 \mathrm{lb}
$$

(b)

$$
T_{B C}=\frac{300 \mathrm{lb}}{\sin 95^{\circ}} \sin 50^{\circ}
$$

$$
T_{B C}=231 \mathrm{lb}
$$



## SOLUTION


(a)

$$
\Sigma \mathbf{F}_{x}=0: \quad-\frac{12}{13} T_{A C}+\frac{4}{5}(360 \mathrm{~N})=0
$$

$$
T_{A C}=312 \mathrm{~N}
$$

(b)

$$
\begin{array}{rlr}
\Sigma \mathbf{F}_{y}=0: & \frac{5}{13}(312 \mathrm{~N})+T_{B C}+\frac{3}{5}(360 \mathrm{~N})-480 \mathrm{~N}=0 & \\
T_{B C}=480 \mathrm{~N}-120 \mathrm{~N}-216 \mathrm{~N} & T_{B C}=144.0 \mathrm{~N}
\end{array}
$$



## SOLUTION

Free-Body Diagram: $C$


Force Triangle


Force triangle is isosceles with

$$
\begin{aligned}
2 \beta & =180^{\circ}-85^{\circ} \\
\beta & =47.5^{\circ}
\end{aligned}
$$

(a)
$P=2(800 \mathrm{~N}) \cos 47.5^{\circ}=1081 \mathrm{~N}$
Since $P>0$, the solution is correct.
(b)
$\alpha=180^{\circ}-50^{\circ}-47.5^{\circ}=82.5^{\circ}$

$$
P=1081 \mathrm{~N}
$$

$$
\alpha=82.5^{\circ}
$$



## PROBLEM 2.133

The end of the coaxial cable $A E$ is attached to the pole $A B$, which is strengthened by the guy wires $A C$ and $A D$. Knowing that the tension in wire $A C$ is 120 lb , determine $(a)$ the components of the force exerted by this wire on the pole, $(b)$ the angles $\theta_{x}, \theta_{y}$, and $\theta_{z}$ that the force forms with the coordinate axes.

## SOLUTION

(a)

$$
\begin{array}{ll}
F_{x}=(120 \mathrm{lb}) \cos 60^{\circ} \cos 20^{\circ} & \\
F_{x}=56.382 \mathrm{lb} & F_{x}=+56.4 \mathrm{lb} \\
F_{y}=-(120 \mathrm{lb}) \sin 60^{\circ} & \\
F_{y}=-103.923 \mathrm{lb} & F_{y}=-103.9 \mathrm{lb} \\
F_{z}=-(120 \mathrm{lb}) \cos 60^{\circ} \sin 20^{\circ} & \\
F_{z}=-20.521 \mathrm{lb} & F_{z}=-20.5 \mathrm{lb}
\end{array}
$$

$\cos \theta_{x}=\frac{F_{x}}{F}=\frac{56.382 \mathrm{lb}}{120 \mathrm{lb}}$

$$
\theta_{x}=62.0^{\circ}
$$

$\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-103.923 \mathrm{lb}}{120 \mathrm{lb}}$

$$
\theta_{y}=150.0^{\circ}
$$

$\cos \theta_{z}=\frac{F_{z}}{F}=\frac{-20.52 \mathrm{lb}}{120 \mathrm{lb}}$

$$
\theta_{z}=99.8^{\circ}
$$



## SOLUTION

$$
\begin{aligned}
\overrightarrow{C A} & =-(900 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}-(920 \mathrm{~mm}) \mathbf{k} \\
C A & =\sqrt{(900 \mathrm{~mm})^{2}+(600 \mathrm{~mm})^{2}+(920 \mathrm{~mm})^{2}} \\
& =1420 \mathrm{~mm} \\
\mathbf{T}_{C A} & =T_{C A} \lambda_{C A} \\
& =T_{C A} \frac{\overrightarrow{C A}}{C A} \\
\mathbf{T}_{C A} & =\frac{2130 \mathrm{~N}}{1420 \mathrm{~mm}}[-(900 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}-(920 \mathrm{~mm}) \mathbf{k}] \\
& =-(1350 \mathrm{~N}) \mathbf{i}+(900 \mathrm{~N}) \mathbf{j}-(1380 \mathrm{~N}) \mathbf{k} \\
& \left(T_{C A}\right)_{x}=-1350 \mathrm{~N}, \quad\left(T_{C A}\right)_{y}=900 \mathrm{~N}, \quad\left(T_{C A}\right)_{z}=-1380 \mathrm{~N}
\end{aligned}
$$



## PROBLEM 2.135

Find the magnitude and direction of the resultant of the two forces shown knowing that $P=600 \mathrm{~N}$ and $Q=450 \mathrm{~N}$.

## SOLUTION

$$
\begin{array}{rlrl}
\mathbf{P} & =(600 \mathrm{~N})\left[\sin 40^{\circ} \sin 25^{\circ} \mathbf{i}+\cos 40^{\circ} \mathbf{j}+\sin 40^{\circ} \cos 25^{\circ} \mathbf{k}\right] & & \\
& =(162.992 \mathrm{~N}) \mathbf{i}+(459.63 \mathrm{~N}) \mathbf{j}+(349.54 \mathrm{~N}) \mathbf{k} & \\
\mathbf{Q} & =(450 \mathrm{~N})\left[\cos 55^{\circ} \cos 30^{\circ} \mathbf{i}+\sin 55^{\circ} \mathbf{j}-\cos 55^{\circ} \sin 30^{\circ} \mathbf{k}\right] & & \\
& =(223.53 \mathrm{~N}) \mathbf{i}+(368.62 \mathrm{~N}) \mathbf{j}-(129.055 \mathrm{~N}) \mathbf{k} & & \\
\mathbf{R} & =\mathbf{P}+\mathbf{Q} & & R=940 \mathrm{~N} \\
& =(386.52 \mathrm{~N}) \mathbf{i}+(828.25 \mathrm{~N}) \mathbf{j}+(220.49 \mathrm{~N}) \mathbf{k} & \theta_{x}=65.7^{\circ} \\
R & =\sqrt{(386.52 \mathrm{~N})^{2}+(828.25 \mathrm{~N})^{2}+(220.49 \mathrm{~N})^{2}} & \\
& =940.22 \mathrm{~N} & & \theta_{y}=28.2^{\circ} \\
\cos \theta_{x} & =\frac{R_{x}}{R}=\frac{386.52 \mathrm{~N}}{940.22 \mathrm{~N}} & \theta_{z}=76.4^{\circ} \\
\cos \theta_{y} & =\frac{R_{y}}{R}=\frac{828.25 \mathrm{~N}}{940.22 \mathrm{~N}} & & \\
\cos \theta_{z} & =\frac{R_{z}}{R}=\frac{220.49 \mathrm{~N}}{940.22 \mathrm{~N}} &
\end{array}
$$



## PROBLEM 2.136

A container of weight $W$ is suspended from ring $A$. Cable $B A C$ passes through the ring and is attached to fixed supports at $B$ and $C$. Two forces $\mathbf{P}=P \mathbf{i}$ and $\mathbf{Q}=Q \mathbf{k}$ are applied to the ring to maintain the container in the position shown. Knowing that $W$ $=376 \mathrm{~N}$, determine $P$ and $Q$. (Hint: The tension is the same in both portions of cable $B A C$.)

## SOLUTION

$$
\begin{aligned}
\mathbf{T}_{A B} & =T \boldsymbol{\lambda}_{A B} \\
& =T \frac{\overline{A B}}{A B} \\
& =T \frac{(-130 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}+(160 \mathrm{~mm}) \mathbf{k}}{450 \mathrm{~mm}} \\
& =T\left(-\frac{13}{45} \mathbf{i}+\frac{40}{45} \mathbf{j}+\frac{16}{45} \mathbf{k}\right) \\
\mathbf{T}_{A C} & =T \lambda_{A C} \\
& =T \frac{\overline{A C}}{A C} \\
& =T \frac{(-150 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}+(-240 \mathrm{~mm}) \mathbf{k}}{490 \mathrm{~mm}} \\
& =T\left(-\frac{15}{49} \mathbf{i}+\frac{40}{49} \mathbf{j}-\frac{24}{49} \mathbf{k}\right) \\
\Sigma F & =0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{Q}+\mathbf{P}+\mathbf{W}=0
\end{aligned}
$$

## Free-Body $\boldsymbol{A}$ :



Setting coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to zero:
i: $-\frac{13}{45} T-\frac{15}{49} T+P=0 \quad 0.59501 T=P$
$\mathbf{j}: \quad+\frac{40}{45} T+\frac{40}{49} T-W=0 \quad 1.70521 T=W$
$\mathbf{k}: \quad+\frac{16}{45} T-\frac{24}{49} T+Q=0 \quad 0.134240 T=Q$

PROBLEM 2.136 (Continued)

Data:

$$
\begin{array}{rlrl}
W & =376 \mathrm{~N} & 1.70521 T=376 \mathrm{~N} & T=220.50 \mathrm{~N} \\
0.59501(220.50 \mathrm{~N}) & =P & & P=131.2 \mathrm{~N} \\
0.134240(220.50 \mathrm{~N}) & =Q & & Q=29.6 \mathrm{~N}
\end{array}
$$



## SOLUTION

## Free-Body Diagrams of Collars:

A:

$B$ :


$$
\lambda_{A B}=\frac{\overrightarrow{A B}}{A B}=\frac{-x \mathbf{i}-(20 \mathrm{in} .) \mathbf{j}+z \mathbf{k}}{25 \mathrm{in} .}
$$

Collar $A$ :

$$
\Sigma \mathbf{F}=0: \quad P \mathbf{i}+N_{y} \mathbf{j}+N_{z} \mathbf{k}+T_{A B} \lambda_{A B}=0
$$

Substitute for $\lambda_{A B}$ and set coefficient of $\mathbf{i}$ equal to zero:

$$
\begin{equation*}
P-\frac{T_{A B} x}{25 \text { in. }}=0 \tag{1}
\end{equation*}
$$

Collar B:

$$
\Sigma \mathbf{F}=0: \quad(60 \mathrm{lb}) \mathbf{k}+N_{x}^{\prime} \mathbf{i}+N_{y}^{\prime} \mathbf{j}-T_{A B} \boldsymbol{\lambda}_{A B}=0
$$

Substitute for $\lambda_{A B}$ and set coefficient of $\mathbf{k}$ equal to zero:

$$
\begin{equation*}
60 \mathrm{lb}-\frac{T_{A B} z}{25 \mathrm{in} .}=0 \tag{2}
\end{equation*}
$$

(a)

$$
\begin{aligned}
x=9 \text { in. } \quad(9 \mathrm{in} .)^{2}+(20 \mathrm{in} .)^{2}+z^{2} & =(25 \mathrm{in} .)^{2} \\
z & =12 \mathrm{in} .
\end{aligned}
$$

From Eq. (2):
$\frac{60 \mathrm{lb}-T_{A B}(12 \mathrm{in} .)}{25 \mathrm{in} .}$

$$
T_{A B}=125.0 \mathrm{lb}
$$

(b) From Eq. (1):

$$
P=\frac{(125.0 \mathrm{lb})(9 \mathrm{in} .)}{25 \mathrm{in} .}
$$

$$
P=45.0 \mathrm{lb}
$$



## SOLUTION

See Problem 2.137 for the diagrams and analysis leading to Equations (1) and (2) below:

$$
\begin{align*}
P=\frac{T_{A B} x}{25 \mathrm{in} .}=0  \tag{1}\\
60 \mathrm{lb}-\frac{T_{A B} z}{25 \mathrm{in.}}=0 \tag{2}
\end{align*}
$$

For $P=120 \mathrm{lb}$, Eq. (1) yields

$$
\begin{align*}
T_{A B} x & =(25 \mathrm{in} .)(20 \mathrm{lb})  \tag{1'}\\
T_{A B} z & =(25 \mathrm{in} .)(60 \mathrm{lb}) \\
\frac{x}{z} & =2 \tag{3}
\end{align*}
$$

From Eq. (2):

Now write

$$
\begin{equation*}
x^{2}+z^{2}+(20 \mathrm{in} .)^{2}=(25 \mathrm{in} .)^{2} \tag{4}
\end{equation*}
$$

Solving (3) and (4) simultaneously,

$$
\begin{aligned}
4 z^{2}+z^{2}+400 & =625 \\
z^{2} & =45 \\
z & =6.7082 \mathrm{in.} \\
x & =2 z=2(6.70 \\
& =13.4164 \mathrm{in} .
\end{aligned}
$$

$$
\text { From Eq. (3): } \quad x=2 z=2(6.7082 \text { in.) }
$$

$$
x=13.42 \mathrm{in} ., \quad z=6.71 \mathrm{in} .
$$



## PROBLEM 2F1

Two cables are tied together at $C$ and loaded as shown. Draw the free-body diagram needed to determine the tension in $A C$ and $B C$.

## SOLUTION

Free-Body Diagram of Point $C$ :

$W=(1600 \mathrm{~kg})\left(9.81 \mathrm{~m} / \mathrm{s}^{2}\right)$
$W=15.6960\left(10^{3}\right) \mathrm{N}$
$W=15.696 \mathrm{kN}$


## SOLUTION

## Free-Body Diagram of Point E:




## SOLUTION

## Free-Body Diagram of Point $A$ :




## SOLUTION

## Free-Body Diagram of Point $B$ :



$$
\begin{aligned}
& W_{E}=250 \mathrm{~N}+765 \mathrm{~N}=1015 \mathrm{~N} \\
& \theta_{A B}=\tan ^{-1} \frac{8.25}{14}=30.510^{\circ} \\
& \theta_{B C}=\tan ^{-1} \frac{10}{24}=22.620^{\circ}
\end{aligned}
$$

Use this free body to determine $T_{A B}$ and $T_{B C}$.

## Free-Body Diagram of Point $C$ :


$\theta_{C D}=\tan ^{-1} \frac{1.1}{6}=10.3889^{\circ}$
Use this free body to determine $T_{C D}$ and $W_{F}$.
Then weight of skier $W_{S}$ is found by

$$
W_{S}=W_{F}-250 \mathrm{~N}
$$



## SOLUTION

## Free-Body Diagram of Point $\boldsymbol{A}$ :




## SOLUTION

Free-Body Diagram of Point $A$ :


$$
\begin{aligned}
W & =(120 \mathrm{~kg})\left(9.81 \mathrm{~m} / \mathrm{s}^{2}\right) \\
& =1177.2 \mathrm{~N}
\end{aligned}
$$



## SOLUTION

Free-Body Diagram of Point $C$ :



## PROBLEM 2.F8

A transmission tower is held by three guy wires attached to a pin at $A$ and anchored by bolts at $B, C$, and $D$. Knowing that the tension in wire $A B$ is 630 lb , draw the free-body diagram needed to determine the vertical force $\mathbf{P}$ exerted by the tower on the pin at $A$.

## SOLUTION

Free-Body Diagram of point $\boldsymbol{A}$ :


